

PERFORMANCE OPTIMIZATION OF AN OXYGEN PLANT USING A COMPUTER SIMULATOR

٢٠١٧

BY
REYAD KHALID YASEEN AWWAD

SUPERVISOR
Dr. NASRI J. RABADI

*Submitted in partial fulfillment of the requirements for the degree of master of science
in mechanical engineering, Faculty of Graduate Studies, University of Jordan.*

March, 1991

نوقشت هذه الرسالة بتاريخ ٢/٣/١٩٩١م واجبزت .

لجنة المناقشة

- ١) د. نصري جبرائيل الربضي / مقرر ا-
- د. كمال خوده
- ٢) د. محمود سليم عوده / عضوا-
- ٣) د. نعيم محمد الفقير / عضوا-
- Chairman of the Committee

الخلاصة

هذه الرسالة تبحث الدارة التشغيلية لمصنع الأكسجين سوفييتي المصنع وهو من النوع K-0.15 الذي يستخدم دورة تبريد ذات ضغط متوسط. إن هذا البحث يوضح أساساً إلى تفاصي إمكانية تحسين القدرة الإنتاجية للمصنع لمواجهة الطلب المتزايد على الأوكسجين.

وللإنجاز حسابات الإلتزامن الحراري والكتلي للعمليات التي تؤديها معدات المصنع فقد تم استخدام برنامج حاسوب للمحاكاة يعرف باسم CODE حيث تم بمساعدته إنشاء مخطط عمليات يحاكي دارة التشغيل للمصنع المذكور.

إن متطلبات الطاقة تعتبر العامل الأساسي في حساب كلفة الإنتاج ضمن مصنع الأوكسجين. ومثل هذه المتطلبات يمكن التحكم بها من قبل شبكة المبادلات الحرارية الموظفة في المصنع.

وللتتبُّوء بشبكة مثلثي للمبادلات الحرارية تم تبني تحليل تكاملي للطاقة باستخدام أسلوب يعرف بجدول المسألة (المشكلة). وهو أسلوب غايته تحديد المتطلبات الدنيا من الطاقة للدارة التشغيلية.

ولما كانت دارة المصنع تتتوفر على نوع واحد من هذه المتطلبات (متطلبات تبريد) أصبح ممكناً استخدام ما يعرف بالتصميم ذي الإسترداد الدقيق للطاقة ويرمز له بر MER Design . وعن طريق التأثير يمكن توقع ومن ثم الإختبار بالحاسوب للوضع الدائم الذي ينبغي أن تكون عليه مسارات العمليات. إن ما تم الحصول عليه من نتائج يبين أن الوضع المشار إليه يتافق مع الظروف التشغيلية لدارة المصنع . إضافة إلى أنه تم التثبت من أن الوضع الجديد لمسارات العمليات يحقق خفضاً مقداره (٤٢%) من المساحة اللازمة للانتقال الحرارة وذلك بثبوت القدرة الإنتاجية.

وعلاوة على ما تقدم ، فقد ظهر أن عمودي (برجي) التكثير بإمكانهما استيعاب معدل إضافي من المائع تيد التكثير يصل إلى (٦٠٪) من الحال القيمة الاسمية لمعدل التدفق . مع ذلك فإن معدات رئيسية في المصنع (الضاغطه ، مضخة الأوكسجين السائل ، منقى الهواء ، المدد التوربيني ، وبعض أجزاء من شبكة الأنابيب) ينبغي إعادة تصمييمها اذا تقرر التوسع في القدرة الإنتاجية .

وبدراسة اقتصادية مقارنة في آخر البحث تبين أن علامة الوحدة المنتجة (متر مكعب واحد من غاز التكسجين) في المصنع بعد رفع قدرته الإنتاجية بمقدار (٤٦٠) تعادل ما نسبته (٤٩٠) من الكلفة ذاتها إذا ما تم استخدام مصنع آخر مماثل للموجود حالياً.

إن النتائج التي ينتهي إليها هذا البحث لا تشجع على اللجوء بقرار توسيع المصنع الحالي .

CONTENTS

	<u>page</u>
AKNOWLEDGEMENT	v
ABSTRACT	vi
NOMENCLATURE	viii
List of Tables	xi
List of Figures	xii
Chapter One	
INTRODUCTION	1
1.1 Problem Statement	1
1.2 Oxygen Plant	2
1.2.1 Application	2
1.2.2 Circuit arrangement and operating principles	3
a. General information	3
b. Modes of operation	3
Mode i	3
Mode ii	7
Mode iii	7
Mode iv	7
1.2.3 Technical data	8
1.2.4 Equipment list	9
1.2.5 Individual description of equipment	9

* Separation unit	9
* Heat exchanger, A4	10
* Preliminary heat exchanger, A3 ..	12
* Sub-cooler, A21	13
* Sub-cooler, A6	14
* Lower column, A7	15
* Upper column, A9	15
* Packet (plate-fin condenser)	16
* Moisture separator	17
* Liquid-oxygen pump	18
* Turbo-expander	18
* Air purifier	19
* Air compressor	21

Chapter Two

LITERATURE SURVEY	24
2.1 Oxygen Discovery	24
2.2 Old Production Techniques	26
2.3 Gas Liquefaction	29
2.4 Heat Exchanger Network Optimization	32

Chapter Three

METHOD OF ANALYSIS	36
3.1 CODE Program	37
3.1.1 CODE skills	39
3.1.2 Run procedure	42
3.1.3 Equipment modules	43

3.2 Plant Simulation	45
- Air compressor	47
- Preliminary heat exchanger	47
- Air purifier	48
- Heat exchanger/ upper half	48
- Heat exchanger/ lower half	49
- Lower column	50
- Sub-cooler, A21	50
- Sub-cooler, A6	50
- Upper Column	50

Chapter Four

INVESTIGATION OF THE EXISTING PLANT	54
4.1 Introduction	54
4.1.1 Optimum process design	54
4.2 Energy Integration Analysis	56
4.2.1 Method of approach	56
4.2.2 Stream definition	58
4.2.3 Calculation of minimum energy requirements	62
4.3 Maximum Energy Recovery Design	68
4.3.1 Stream matching/ exchanger network	68
4.4 Column-Tray Sizing	78

Chapter Five

DISCUSSION OF RESULTS	82
5.1 Base Design	82

5.1.1 Plant simulation	82
5.1.2 Evaluation of the heat exchanger network	83
5.1.3 Inspection of the air-separation unit/ rectifying columns	96
5.2 Economical Comparison of the Existing and the Proposed Designs	99
 Chapter Six	
CONCLUSION	106
REFERENCES	108
 APPENDICES	
Appendix A : Temperature interval analysis	113
Appendix B : Computer results of the simula- ting flowsheet of the oxygen plant of type K-0.15	136
Appendix C : Computer results of the optimi- zed heat exchanger network of the oxygen plant	171
Appendix D : Computer results of the column- tray sizing under the nominal flow rate of air	183
Appendix E : Computer results of the column- tray sizing under 1.7 times the nominal flow rate of air	202

AKNOWLEDGEMENT

I acknowledge with gratitude the careful supervision and the continuous encouragement of my supervisor, Dr. Nasri J. Rabadi. His valuable pointers and friendship were very helpful.

My best regards to Prof. Siham A. Temtami for her assistance in the early stages of the research and for her valuable cooperation during my training on the computer simulator.

I also extend my thanks and appreciation to the Jordan Iron and Steel Industries Company for their financial support and technical assistance by the Vice Chairman of the Board, Mr. Mohammed Y. Taher, the factories manager, Mr. Khalid Najdawi and the Soviet expert, Mr. Vitaly G. Kosichianko.

Finally, I deeply appreciate the sincere help, encouragement and cooperation of my family during the difficult times of my work on this thesis.

ABSTRACT

This thesis investigates the operating circuit of a Soviet made oxygen plant of type K-0.15 which employs a medium-pressure refrigeration cycle. The purpose of this investigation is to explore the possibility of improving the production capacity of the plant to meet the increasing market demand on oxygen.

A computer program, CODE, which is a steady state process simulator has been used to calculate heat and material balances of process equipment. A flowsheet is generated to simulate the operating circuit of the plant under consideration.

Energy requirements are considered to be the primary factor determining the production cost of the oxygen plant. Such requirements are controlled by the employed network of heat exchangers of the plant.

To predict the optimum exchanger network, the Energy Integration analysis is adopted using the Problem Table method. This method determines the minimum energy requirements of the operating circuit.

Since the plant circuit includes only one type of energy

requirements (that is cooling requirement), it has become possible to use the Maximum Energy Recovery (MER) design. Such type of a problem is sometimes referred to as threshold problem.

Using MER, an optimum configuration of process streams are predicted and checked by the computer simulator. The results obtained show that the predicted configuration complies with the operating conditions of the plant circuit. A reduction in heat transfer area of about 21% is found possible while keeping the production capacity invariant.

Furthermore, it is determined that the rectifying columns are capable of handling a flow rate 60% higher than the nominal flow rate value. However, major components of the plant (compressor, pump, purifier, turbo-expander and some piping) should be redesigned if expansion is implemented. A comparative economical study shows that the cost of unit production after expanding the existing plant by 60% points to 90% of the unit production cost if a new similar plant is erected. This result indicates that a decision to expand the plant may not be favourable.

NOMENCLATURE

A : Surface area of heat transfer, m^2

A1 & A2 : Moisture separators

A3 : Preliminary heat exchanger

A4 : Heat exchanger

A6 : Three-way subcooler

A7 : Lower column

A8 : Plate-fin condenser

A9 : Upper column

A15 & A16 : Cylinders of the air purifier

BP1, BP2 & BP3 : Throttle valves

CP : Heat-capacity flow rate, $\text{kW}/^\circ\text{C}$

CP_i : Heat-capacity flow rate of stream i, $\text{kW}/^\circ\text{C}$

Cp : Specific heat, $\text{kW}/\text{kg} \cdot {}^\circ\text{C}$

d : Tube diameter, m

D_e : Hydraulic diameter, m

D_i : Inside diameter of the outer tube for
tube-in-tube section, m

d_i : Inside diameter of the inner tube for
tube-in-tube section, m

D_o : Outside diameter of the outer tube for
tube-in-tube section, m

d_o : Outside diameter of the inner tube for

- tube-in-tube section, m
 $d\dot{Q}$: differential heat flow-rate, kW
 \dot{H} : Enthalpy flow rate, kW
 H1 : Liquid-oxygen pump
 HAF : Heat availability function
 h : Film coefficient of heat transfer, $W/m^2.^oC$
 h_i : Film coefficient of heat transfer based on
 the inside diameter of the tube, $W/m^2.^oC$
 h_{io} : Modified h_i , $W/m^2.^oC$
 h_o : Film coefficient of heat transfer based on
 the outside diameter of the tube, $W/m^2.^oC$
 JISICO : Jordan Iron and Steel Industries Company
 K1 : Air compressor
 L : Tube length, m
 LMTD : logarithmic mean temperature difference, oC
 \dot{m} : Mass flow rate, kg/s
 MER : Maximum Energy Recovery (design)
 O_2 : Oxygen
 Pr : Prandtl number
 Q : Heat flow rate, kW
 Re : Reynolds number
 T : temperature, oC
 T1 : Air temperature before preliminary heat
 exchanger, oC
 T2 : Air temperature after preliminary heat
 exchanger, oC
 T3 : Air temperature after air purifier, oC
 T4 : Nitrogen temperature after heat exchanger

ΔT_1 , °C

T₅ : Nitrogen temperature after preliminary heat exchanger A3, °C

T₉ : Air temperature before turbo-expander, °C

T₁₀ : Air temperature after turbo-expander, °C

TA1 : Turbo-expander

T_s : Supply temperature of a stream, °C

T_t : Target temperature of a stream, °C

U : Overall coefficient of heat transfer, W/m². °C

UA : The "UA" product of U times A, W/ °C

v : Flow velocity, m/s

ΔT : Temperature difference, °C

ΔT_i : Temperature difference for stream i, °C

ΔT_{min} : Minimum temperature difference, °C

ΔH : Difference of enthalpy flow rate, kW

μ : Kinetic viscosity, Pa.s

ρ : Density, kg/m³

LIST OF TABLES

Table		Page
4.1 : Stream data		62
4.2 : Temperature interval analysis		65
5.1 : "UA" for actual design		86
5.2 : "UA" for predicted flowsheet		87
5.3 : "U" for actual design		88
5.4 : Heat transfer area for the predicted design		89
5.5 : Pr & Re numbers for air and oxygen		93
5.6 : Cost elements for a new oxygen plant of type K-0.15 and the extra capacity with respect to 60% expansion		100
5.7 : Cost of expanding the production capacity of the existing plant by 60%		101
5.8 : Annual costs for the existing and expanded plants		102
5.9 : Alternative - decision analysis		104

LIST OF FIGURES

<u>Figure</u>	<u>Page</u>
1.1 : Schematic flow diagram of the oxygen plant of type K-0.15	5
3.1 : Simulating flowsheet of the oxygen plant	52
3.2 : Scheme of equipment connection	53
4.1 : Stream - equipment layout	59
4.2 : Stream formation	59
4.3 : Temperature - enthalpy diagram	61
4.4 : Temperature interval analysis	64
4.5 : Heat cascading	67
4.6 : Stream enthalpy	69
4.7 : Stream matches	71
4.8 : Final configuration	72
4.9 : Matches 1 and 2/first stage	74
4.10 : Matches 1 and 2/second stage	75
4.11 : Match 4	77
4.12 : Optimized exchanger - network of the oxygen plant	80
..	
4.13 : Tray - sizing of the air rectification columns	81
5.1 : Tube-in-tube section	91

INTRODUCTION

1.1 Problem Statement

In 1988-89, Jordan Iron and Steel Industries Company (JISICO) has installed and commissioned a new imported oxygen plant at its factories in Zerka. The installation and commissioning were done under the supervision of the manufacturer.

During his work as plant engineer at JISICO, the author tried to investigate the plant circuit and its operating conditions. During this investigation, the following questions were raised

- Do the operating conditions set by the manufacturer lead to the optimum performance of the plant ?
- Is it possible to increase the production capacity of the plant without replacing any parts of the plant? If so, what will be the individual contribution of each component to this increase?
- Is there a bottleneck in implementing such an

increase ? If yes, how is it possible to get through this bottleneck ?

- Is it possible to increase the plant capacity by changing the present configuration of the process streams and/or the operating conditions of the plant only ?

These questions are the basis of the present research. In this thesis, answers to the questions listed above are presented.

The following section is a brief description of both the circuit and components of the oxygen plant under consideration.

1.2 Oxygen Plant

1.2.1 Application

The plant under consideration is of type K-0.15⁽¹⁾ and is designed to produce gaseous oxygen of grade 1 according to the USSR standard ГОСТ 5583-78. The plant is manufactured in versions intended for operation in temperate and tropical climatic regions; It displays stable operation in the ambient temperature range of -40 to 40°C.

The oxygen produced by this plant is intended for use mainly in machine building and metal working - welding and

flame cutting of metals -as well as in chemical industry and other fields of technology.

The plant can also be used to produce technical gaseous oxygen of higher purity (99.9%) according to the standard ГОСТ 5583-78 or technical liquid oxygen of grade 1 according to the standard ГОСТ 6631-78 or liquid nitrogen of grade 2 according to the standard ГОСТ 9293-74.

1.2.2 Circuit arrangement and operating principle

a. General information

Air is precooled in the preliminary heat exchanger. Water vapor, hydrocarbon products and carbon dioxide are removed from air by the zeolite air purifier. The refrigeration losses are compensated by the medium pressure refrigeration process employing the turboexpander. Refrigeration is recuperated in a tubular heat exchanger. The air is separated into oxygen and nitrogen by a two- column rectification system.

b. Modes of operation

The plant has the following four modes of operation :

Mode i : Making grade 1 gaseous oxygen

After passing through a dust filter, air is compressed by

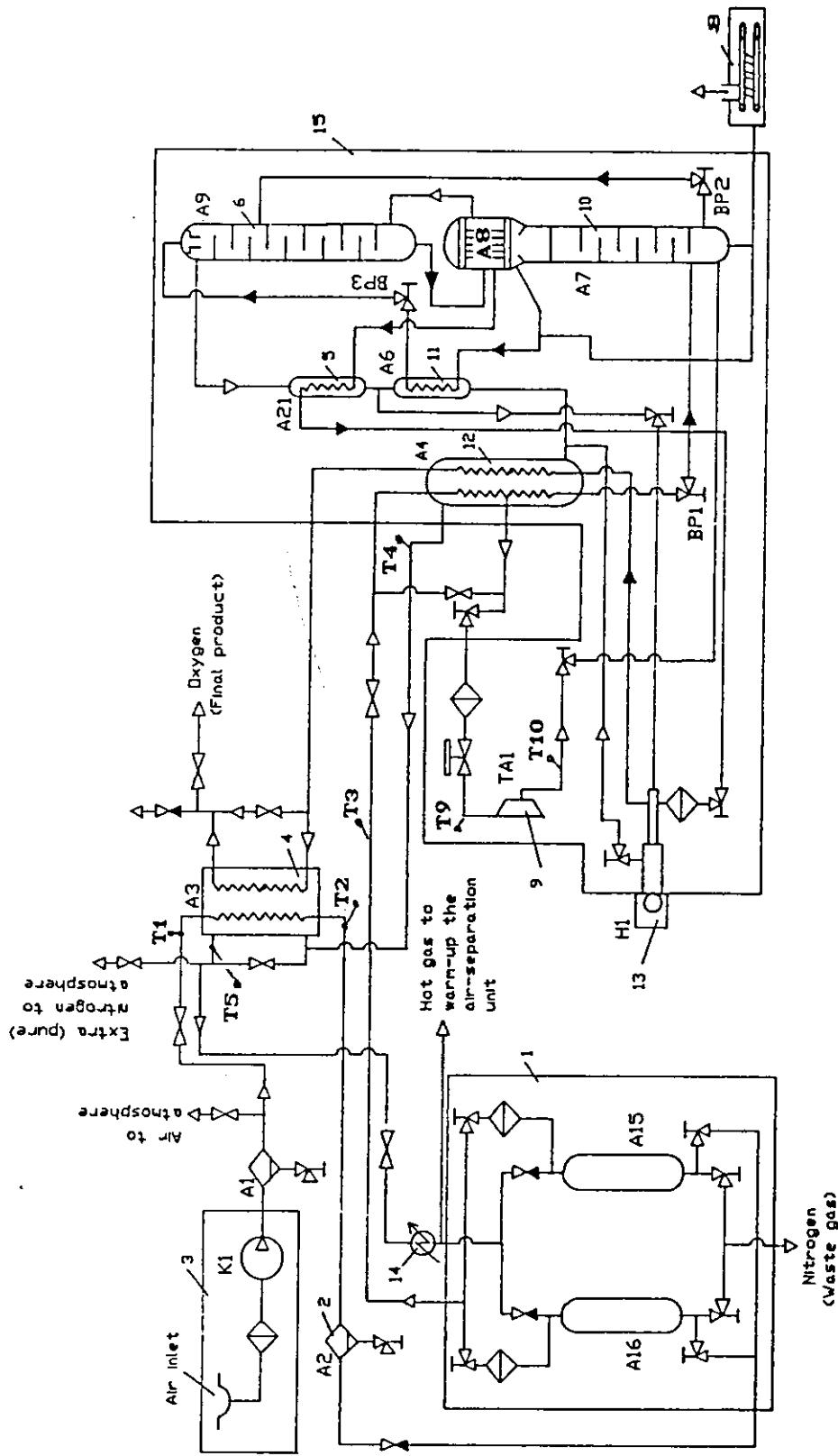


Figure 1.1 : Schematic flow diagram of the oxygen plant type K-0.15

1-Air purifier, 2-Moisture separator, 3-Air compressor, 4-Preliminary heat-exchanger, 5-Two-way sub-cooler, 6-Upper column, 7-Plate-fin condenser, 8-Emergency oxygen-evaporator, 9-Turbo-expander, 10-Lower column, 11-Three-way sub-cooler, 12-Heat exchanger, 13-Heat exchanger, 14-Liquid-oxygen pump, 15-Electric heater, 16-Casing of the air-separation unit

In the lower column, air is separated into oxygen-rich (33.5%) liquid and nitrogen reflux containing 2.0-2.5% oxygen.

The still liquid in the amount of 60% of the processed air is throttled to a pressure of 0.04 MPa by valve BP2 and passed to the 40th plate of the upper column A9 (if counting from bottom to top).

The nitrogen reflux in the amount of 40% of the processed air is passed to subcooler A6 in which it is cooled by the gaseous nitrogen leaving the upper column. Then the nitrogen reflux is throttled by valve BP3 and passed to the top plate of the upper column.

In the upper column, air is finally separated into gaseous nitrogen and liquid oxygen.

Liquid oxygen is withdrawn from the still of the upper column to condenser-evaporator A8. Liquid oxygen in the amount of 19% of the processed air is passed from the condenser-evaporator to subcooler A21. On passing through the subcooler, liquid oxygen is used for cooling the jacket of the liquid oxygen pump H1 whereupon it is passed to the suction side of this pump. Liquid oxygen is pumped at any required pressure within the range of 0 to 20 MPa and directed to the heat exchanger A4 in which it is evaporated and passed to the preliminary heat exchanger. Gaseous oxygen

from the preliminary heat exchanger is passed to a filling manifold.

Gaseous nitrogen in the amount of 80% of the processed air is passed from the upper column to subcooler A6 and then to heat exchanger A4 in which it is heated to temperature T4. Thereafter it is directed to preliminary heat exchanger A3 and heated to temperature T5. Part of the gaseous nitrogen is passed to the air purifier for regeneration of the zeolite and cooling the absorbers. The remaining gas is released to the atmosphere.

Mode ii : Making high purity gaseous oxygen

When the plant is operated in mode ii, higher purity gaseous oxygen is produced by reducing the rate of product withdrawl (decreasing the pump delivery.)

Mode iii : Making grade 1 liquid oxygen

When the plant is operated in this mode, liquid oxygen is withdrawn and transferred to a storage tank. In this case the liquid oxygen pump is kept inoperative. The air pressure at the compressor exit is increased to 6.4 MPa.

Mode iv : Making liquid nitrogen

When the plant is operated in this mode, a part of the sub-cooled nitrogen reflux is withdrawn as product. In this

case the pressure after the compressor is raised up to 6.4 MPa. The liquid oxygen pump is kept inoperative.

1.2.3 Technical data

The nominal production capacity of the plant is 170 m³/hr of 99.7% pure gaseous oxygen at 10°C and 15 MPa. This capacity is restricted to the following conditions

- Delivery of the air compressor employed in the plant complies with the technical specifications in its certificate and its negative tolerance does not exceed 5%.
- Suction conditions conform with FOCT 2939-63, whereby the pressure equals 101,325 Pa (760 mm Hg), and the temperature equals 293.15 K (20°C).
- Temperature of the cooling water delivered to the compressor is 283.15 K (10°C).
- Purging losses of air do not exceed 3% .

The plant will normally operate with the volumetric flow rate of the air supplied for separation ranging from 750 m³/hr to 1056 m³/hr .

1.2.4 Equipment list

- (a) Air separation unit KK 0216.00.000 (15)
- (b) Turboexpander AT-0.6/4 (9)
- (c) Air purifier KK 0926.000 (1)
- (d) Set of testing instruments and devices KK 0036.10.000
- (e) Preliminary heat exchanger KK 3283.000 (A3)
- (f) Air compressor 305 ВП 16/70 (K1)
- (g) Two moisture separators KK 5013.000 (A15,A16)

The complete list also incorporates valves which are installed on the pipelines interconnecting units of the plant, as well as instruments for measuring purity of the product and a set of spare parts, service tools and accessories.

1.2.5 Individual description of equipment

* Separation unit

The separation unit is designed for cooling, liquefying and separating compressed atmospheric air into oxygen and nitrogen. A medium pressure cycle employing a turboexpander is used for cooling the air. Oxygen and nitrogen are produced from air by double-column rectification.

The separation unit consists of a housing, apparatus for

392091

separating air into oxygen and nitrogen (high- and low-pressure rectifying columns, condenser-evaporator), heat exchanger, liquid oxygen subcooler and subcooler of still liquid and nitrogen reflux, turboexpander, liquid-oxygen pump, valve equipment and instruments.

All the apparatus of the separation unit are accommodated in the housing. This housing is filled with a heat insulating material (expanded perlite powder), which reduces refrigeration losses to environment.

* Heat exchanger, A4

The heat exchanger is designed for cooling the air coming into the separation unit by the outgoing nitrogen and by compressed oxygen.

It is a three-stream wound tube-in-tube type apparatus. Air is the hot stream while oxygen and nitrogen are the cold streams. The number of wound tube layers is 8, and the number of tubes is 19. The outer tubes are of 10×1 mm diameter while the inner tubes are of 5×1 mm diameter.

* The outside diameter of the tube equals 10 mm while its thickness equals 1 mm. This convention of tube diameter is adopted all through this thesis.

The apparatus weighs 274 kg. The working pressure in 5x1 mm diameter tubes is 20 MPa. These tubes are subjected to a test pressure of 25 MPa.

The working pressure in 10x1 mm diameter tubes is 7.0 MPa. These tubes are subjected to a test pressure of 9.0 MPa. The working pressure in the space between tubes is 0.07 MPa.

The operating temperature is from +10°C to -176°C. The external heat exchange surface of 10x1 mm diameter air tubes is 9.16 m². The external exchange heat surface of 5x1 mm diameter oxygen tubes is 4.58 m².

The operating temperature is from +10°C to -176°C. The external heat exchange surface of 10x1 mm diameter air tubes is 9.16 m². The external exchange heat surface of 5x1 mm diameter oxygen tubes is 4.58 m².

When the exchanger is in operation, the high pressure oxygen is passed through the 5x1 mm diameter tubes; the air to be cooled is passed through the annular space inside the 10x1 mm diameter tubes while the cooling nitrogen flows through the space between the tubes.

The hydraulic resistance in the inter-tubular space as determined by testing at 20°C is as follows :

- in the upper zone it is of 330 80 mm H₂O,
- in the lower zone it is of 30 10 mm H₂O,

The air flow rate during testing is 400 m³/hr.

* Preliminary heat-exchanger, A3

The preliminary heat-exchanger is intended for cooling the air before it enters the purifier. It is a coiled-tube three -stream apparatus with parallel wound coils of air and oxygen tubes. The coils are wound in eight layers. The air tubes are 8x1 mm in diameter and 16 in number. The oxygen tubes are 8x1.5 mm in diameter and 3 in number.

The working pressure in the air tubes amounts to 7.0 MPa. The tubes are subjected to a test pressure of 9 MPa. The working pressure in the oxygen tubes amounts to 20 MPa. The tubes are subjected to a test pressure of 25 MPa.

The working pressure in the space between the tubes and casing amounts to 0.07 MPa.

Technical characteristics of the preliminary heat exchanger intended for use in temperate conditions are as follows :

- The mass of the apparatus is 311 kg.
- The height of the tube winding is 293 mm.
- The height of the apparatus is 1605 mm.

- The outside heat exchange surface of :

- Air tubes is 7.1 m^2 and,
- Oxygen tubes is 1.4 m^2 .

When the apparatus is in operation, air to be cooled is passed through 8x1 mm diameter tubes and high pressure oxygen is passed through the 8x1.5 mm diameter tubes while gaseous nitrogen flows through the space between the tubes.

The hydraulic resistance in the inter-tubular space as determined at 20°C is 390 90 mm H₂O at air flow rate of $700 \text{ m}^3/\text{hr}$.

* Sub-cooler, A21

This sub-cooler is designed for subcooling oxygen by means of still liquid.

The sub-cooler is a coiled tube two-stream apparatus with 4 tubes coiled in 2 layers. The apparatus weighs 12.0 kg. The working pressure in the tubes is 0.07 MPa. The tubes are subjected to a test pressure of 0.2 MPa. The working pressure in the space between the tubes is 0.2 MPa. The test pressure applied in the space is 0.3 MPa.

The design temperature is within -186°C to -188°C . The

outside heat exchange surface of the 8x0.8 mm diameter tubes amounts to 0.41 m^2 .

When the sub-cooler is in operation, liquid oxygen is passed through the tubes while the still liquid flows through the space between the tubes.

* Sub-cooler, A6

This sub-cooler is designed for sub-cooling the still liquid and nitrogen reflux by the outgoing nitrogen. It is a coiled tube three-stream heat exchanger with two bundles of 8x0.8 mm diameter tubes which are parallel-coiled in four layers.

There are 8 tubes in each section of the apparatus (in still liquid section and nitrogen reflux section). The outside heat exchange surface area of the still liquid tubes is 0.85 m^2 , while the outside heat exchange surface area of the nitrogen reflux tubes is 2.12 m^2 .

The apparatus weighs 55 kg. The working pressure in the tubes with still liquid is 0.6 MPa, while the test pressure is 0.9 MPa. The working pressure in the tubes with nitrogen reflux is 0.6 MPa and the test pressure applied to these tubes is 0.9 MPa. The working pressure in the space between the tubes is 0.07 MPa while the test pressure is 0.2 MPa.

* Lower column, A7

The lower column is designed for the separation of air into oxygen-rich liquid, nitrogen reflux, and gaseous nitrogen.

The column is a cylindrical vessel accommodating 20 circular perforated plates. These plates are secured in special creases on the column casing by means of rolling them in.

- Working pressure = 6 MPa.
- Testing pressure = 9 MPa.
- Operational temperature = 93 K.
- Column capacity = 200 liter.
- Mass of column = 75 kg.
- Plate diameter = 0.4 m.

The column complies with the standards for construction and safe operation of high pressure vessels.

* Upper column, A9

The upper column is designed for the final separation of air into gaseous nitrogen and liquid oxygen.

The column is a cylindrical vessel accommodating 56

circular rectifying sieve plates which are fixed in special creases on the column casing by means of rolling them in. Installed in the column still a plate-fin condenser (called "packet" hereinafter).

- Working pressure in column	0.07 MPa
- Testing pressure	0.14 MPa
- Operational temperature	80 K
- Column capacity	900 liter
- Mass of column with packet	300 kg
- Plate diameter	0.5 m.

* Packet (plate-fin condenser), A8

The packet is built into the upper column still and is designed for condensing nitrogen incoming from the lower column owing to the oxygen boiling.

The packet consists of a number of 1 mm thick identical flat spacer plates silumin-coated on both sides, which are inter-laid with corrugated aluminum-foil packing of 0.2 mm thick (working portion) and 0.7 mm thick (distribution portion).

On the opposite ends of the spacer plates are lateral packing bars made of aluminum section. At top and bottom the packet is fitted with 4-mm thick cover plates coated inside

with silumin. The spacer and cover plates are made of an aluminum alloy, as well as the aluminum packing foil.

The channels of nitrogen are communicated by means of headers. The oxygen space channels are at the end faces of the packet so that oxygen passes through the packet from top to bottom.

- Working pressure in nitrogen space	0.6 MPa
- Testing pressure in nitrogen space	0.9 MPa
- Working pressure in oxygen space	.007 MPa
- Testing pressure in oxygen space	2.0 MPa
- Operational temperature	77 K
- Number of nitrogen space channels	16
- Number of oxygen space channels	17
- Mass of packet	76 kg

* Moisture separator, A1 & A2

The moisture separator is designed for removing moisture droplets from air before the preliminary heat exchanger and before the air purifier inlets.

The water separator is a welded cylindrical vessel into which the air flow is directed tangentially.

The capacity of the apparatus is 16 liters and it weighs

40 kg.

The working pressure in the separator amounts to 7.0 MPa while test pressure applied to the apparatus is 8.8 MPa.

When the apparatus is in operation, moisture is separated from the air due to the sharp change in the velocity and direction of the air flow.

* Liquid-oxygen pump, H1

The pump used is one of the Russian pump series 2НСГ. These pumps are designed for use in air separation plants or gasifiers to transfer super-cooled or compressed liquefied oxygen, nitrogen or argon.

- Mean delivery per charging cycle at a piston stroke of 30 mm, m^3/s (l/h) $(7.0 - 0.7) \times 10^{-5}$
 (252 ± 25)
- Delivery control range, % $(100 - 40) \pm 10$
- Mean discharge pressure per cycle, MPa 14.7
- Maximum discharge pressure, MPa 22.5 - 1.2
- Maximum power (at maximum delivery), kW 4.4

* Turboexpander, TA1

The expansion turbine type AT-0.6/4 incorporated in the air separation plant is designed for expansion cooling of

compressed air in a low-temperature refrigeration cycle. The air to be expanded must be free from mechanical admixtures and carbon dioxide.

- Refrigerating capacity, W 6400
- Inlet air pressure, MPa :
 - Working 3.96 ± 0.49
 - Maximum 6.28
- Air-handling capacity, kg/hr 600^{+60}_{-30}
- Outlet air pressure, MPa, 0.59
- Inlet air temperature, K ($^{\circ}\text{C}$) 173 ± 5
 (-100 ± 5)
- Outlet air temperature, K ($^{\circ}\text{C}$) 120 (-153)
- Rotor speed, thousands of rpm 150^{+25}_{-15}
- Cooling water consumption, kg/hr 3
- Cooling water pressure, MPa, 0.196

* Air purifier, A15 & A16

The air purifier is intended for removal of moisture, carbon dioxide and any other hydrocarbon from air to be distilled. It is designed to operate at the medium pressures (3.5 - 7 MPa).

The purifier may be incorporated in an air separation plant or in air complex purification installation.

An electric heater is provided with the purifier. This heater is intended for warming regenerating gas. When the air purifier is incorporated in an air separation plant, The heater is also used for warming the air or nitrogen used for warming up the separation unit.

Technical Data

1. Air purifier capacity, m^3/s 0.227 - 0.267
2. Working pressure, MPa 3.5 - 7.0
3. Air temperature at
 - purifier inlet, K ($^{\circ}$ C) 278 - 283
 - (5 - 10)
4. Content of CO_2 :
 - a) in purified gas, ppm 3 max.
 - b) at the end of adsorption cycle, in 8 hours on completion of parallel operation, ppm ... 20 max.
5. Dehydration level, dew point, K ($^{\circ}$ C) 203 max.
 - (-70) max.
6. Length of adsorber working cycle in air purification, with parallel operation taken into consideration, hr 9 max.
7. Length of desorption cycle, 2 hrs and 10 - 20 min.
8. Length of adsorber cooling cycle 4 hrs and 45 min. to 5 hrs

9. Duration of adsorbent operating

cycle prior to replacement, 2 years approx.

10. Temperature of regenerating gas

after it is raised by electric

heater, K ($^{\circ}$ C) 673 (400)

11. Temperature of regenerating gas

at the adsorber outlet by the end

of regeneration cycle, K ($^{\circ}$ C) 493 (220)

12. Regeneration flow rate, m^3/s :

a) At a temperature of 0 to 5° C

at the electric heater inlet 0.0472

b) At a temperature of 35 to 45° C

at the electric heater inlet 0.061 to 0.695

13. Electric heater consumed power, kW 30

14. Weight of zeolite used

as adsorbent, particle size 4 mm, kg 460

15. Main dimensions of air purifier, m :

a) Height 3.35

b) Width 1.845

c) Depth 2.05

16. Total weight of air purifier, kg 2565

including weight of zeolite 460 \pm 50

* Air compressor, K1

It is a stationary, reciprocating, direct action, four-stage, with perpendicular cylinders and with inter- and terminal coolers compressor.

Technical Data

1. Discharge pressure, MPa 7
2. Free; air delivery, m³/min 16 ± 5%
3. Power consumed, kW 190
4. Speed, rpm 500
5. Cooling water requirements

without terminal cooler with
entrance 15°C, liter/min 110
6. Recommended cooling water
requirement for terminal
cooler with entrance temp-
arature 15°C, liter/min 40
7. Number of cylinders 4
8. Cylinder bores, mm :

1st stage	500
2nd stage	270
3rd stage	145
4th stage	100
9. Piston stroke, mm 220
10. Stage outlet pressures, MPa :

1st stage	0.22 - 0.28
2nd stage	0.8 - 1.2
3rd stage	2.2 - 2.85
4th stage	7
11. Lubrication :
 - a) Crank mechanism circulation system with gear pump
 - b) Cylinders central force - feed

from a lubrication unit
incorporated with the
compressor

12. Oil capacity of compressor	
frame, liter	136
13. Force - feed delivery rate, g/hr	70
14. Overall dimensions, mm :	
a) length	2,670
b) width	1,880
c) height	2,560
15. Mass, kg	4,400
16. Automatics	protection and signaling for main compressor para- meters.
17. Drive	synchronous motor, direct coupled.
18. Synchronous motor :	
a) Rated power ,kW	200
b) Speed, rpm	500
c) Voltage, V	380
d) Mass, kg	1,660

Chapter Two

LITERATURE SURVEY

Oxygen is the most abundant element on the earth's surface. It occurs both as the free element and combined in innumerable compounds: It comprises 23% of the atmosphere by weight, 46% of the lithosphere, and more than 85% of the hydrosphere (~ 85% of the oceans and 88.81% of pure water).

2.1 Oxygen Discovery

In the 15th century, Leonardo da Vinci noted that air has several constituents, one of which supports combustion.⁽²⁾

The discovery of oxygen, however, is generally credited to C. W. Scheele and J. Priestley independently in 1773-4, though several earlier investigators had made pertinent observations without actually isolating and characterizing the gas⁽³⁻⁵⁾. Indeed, it is difficult to ascribe a precise meaning to the word *discovery* when applied to a substance so ubiquitous present as oxygen; particularly when (a) experiments on combustion and respiration were interpreted in terms of the phlogiston theory, (b) there was no clear consensus on what constituted "an element", and (c) the birth of Dalton's atomic theory was still far in the future.

Moreover, the technical difficulties before the mid-eighteenth century of isolating and manipulating gases compounded the problem still further, and it seems certain that several investigators had previously prepared oxygen without actually collecting it or recognizing it as a constituent of "common air".

Scheele, a pharmacist in Uppsala, Sweden, prepared oxygen at various times between 1771-3 by heating KNO_3 , $\text{Mg}(\text{NO}_3)_2$, Ag_2CO_3 , HgO and a mixture of H_3AsO_4 and MnO_2 . He called the gas vitriol air and reported that it was colorless, odorless and tasteless, and supported combustion better than common air, but the results did not appear until 1777 because of his publisher's negligence.

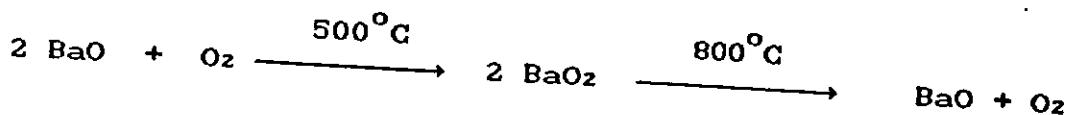
Priestley's classic experiment of using a "burning glass" to calcine HgO confined in a cylinder inverted over liquid mercury was first performed in Clone, England, on Monday, August 1, 1774; he related this to A. L. Lavoisier and others at a dinner party in Paris in October 1774 and published the results in 1775 after he had shown that the gas was different from nitrous oxide. Priestley's ingenious experiments undoubtedly established oxygen as a separate substance (dephlogisticated air) but it was Lavoisier's deep insight which recognizes the new gas as an element and as the key to our present understanding of the nature of combustion. This led to the overthrow of the phlogiston theory and laid the foundations of modern chemistry. Lavoisier named the

element "oxygène" in 1777 in the erroneous belief that it was an essential constituent of all acids (Greek οξυς, oxys, sharp, sour; γεινομαι, geinomai, I produce; i.e. acid forming).

H. Cavendish (1781) reported that water constituents of oxygen and hydrogen. The decomposition of water into its two constituents was first performed in 1800 by W. Nicholson and A. Carlisle through electrolysis.⁽⁷⁾

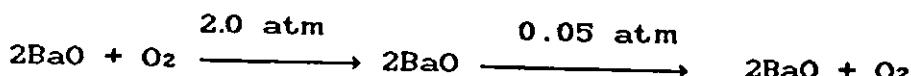
2.2 Old Production Techniques

Before the development of low-temperature distillation route oxygen was made by chemical methods. In the United Kingdom Brin's process was operated commercially from 1886 to 1906⁽⁷⁾. It depended on laboratory observations that barium oxide would react with atmospheric oxygen at about 500°C and that the peroxide so formed would decompose if the temperature was raised to about 800°C,



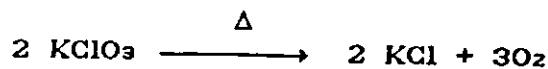
In commercial practice it was found that the barium oxide lost its power to react after a few cycles unless the incoming air was first freed from carbon dioxide, organic

matter and dust.⁽⁷⁾ It was also found that, instead of the expensive and time-consuming practice of raising and lowering the temperature by 300°C, the process could be operated at about 600°C using two different pressures :



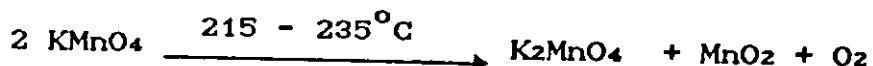
The gas produced by this process was impure by modern standards, usually containing 90-96% oxygen together with nitrogen and smaller amounts of argon and other gases. Occasionally, small-scale laboratory preparations of oxygen are required and the method chosen depends on the amount and purity required and the availability of services. Electrolysis of degassed aqueous electrolytes produces wet O₂, the purest gas being obtained from 30% potassium hydroxide solution using nickel electrodes. Another source is the catalytic decomposition of 30% aqueous hydrogen peroxide on a platinized nickel foil.

Many oxoacid salts decompose to give oxygen when heated. A convenient source is KClO₃ which releases oxygen when heated to 400-500°C according to the simplified equation⁽²⁾,



The decomposition is reduced to 150 °C in the presence of

MnO_2 but then the product is contaminated with up to 3% of ClO_2 . Small amounts of breathable oxygen for use in emergencies (e.g. failure of normal supply in aircraft or submarines) can be generated by decomposition of NaClO_3 in "Oxygen candles". The best method for the controlled preparation of very pure oxygen is the thermal decomposition of recrystallized pre-dried, degassed KMnO_4 in a vacuum line. $\text{Mn}^{(\text{VI})}$ and $\text{Mn}^{(\text{IV})}$ are both formed and the reaction can be formally represented as⁽²⁾,



Where electric power is exceptionally cheap, oxygen may be manufactured by the electrolysis of water, but the amount made by this process constitutes a very small percentage of total production in USA and Canada⁽⁸⁾.

By far, the most important method of manufacturing oxygen is by means of fractional distillation after the liquefaction of air. Liquid air is essentially a mixture of approximately one-fifth oxygen and four-fifths nitrogen. Nitrogen, having a lower boiling point, is allowed to escape, leaving oxygen in the liquid form, which may then be purified.

2.3 Gas Liquefaction

Air-separation plants are built for a wide variety of outputs: To produce one or more of the products in liquid form; and to make products of various purities. The precise practical method employed depends to some extent on the definition of these requirements.

An air-separation plant, however, always has three basic stages, namely purification, liquefaction and distillation. The other processes of prime importance is that of heat transfer, so as to keep losses of coldness to minimum. Among these four processes, liquefaction of a gas presented a serious challenge which was the first article in establishing the gases industry.

Michael Faraday (1845) was able to liquefy many gases for the first time, including ammonia, chlorine, sulphur dioxide and hydrogen sulfide⁽⁷⁾. He generated his gas in one arm of a sealed tube shaped like an inverted V and immersed the other arm in an appropriate cooling mixture, so that a combination of high pressure and low temperature was achieved in one piece of equipment.

Despite the efforts of other workers who used still high pressure there remained a number of gases which could not be converted to liquid form and these became known as "permanent gases". It was only when the concept of critical temperature

Was developed following Andrew's work on carbon dioxide in 1869 that the liquefaction of permanent gas became possible.⁽⁶⁾ The critical temperature is defined as that temperature above which it is impossible to liquefy a gas however great the applied pressure. It should be noted that the critical pressure is the pressure required to liquefy the gas at its critical temperature. At lower temperatures a lower pressure suffices.

Oxygen was first liquefied in 1877 by two workers quite independently using quite different methods.⁽⁷⁾

Cailletet (France) cooled oxygen to -29°C by bath of evaporating sulphur dioxide, compressed to a pressure of 200 atm and released the pressure suddenly. A mist of liquid oxygen was seen. He observed the same phenomenon when using carbon monoxide, acetylene and nitrous oxide but not with hydrogen.

Pictet (Switzerland) used a method now known as the "cascade method" which involves proceeding to lower and lower temperatures in a series of stages using different coolants. Thus he liquefied oxygen compressed to 320 atm by cooling it in liquid carbon dioxide at 140°C . The latter had in its turn been obtained by cooling in sulphur dioxide.

Wroblewski and *Olszewski* in the period 1880-90 were the first to obtain relatively large quantities of liquid oxygen.

They employed a cascade method sometimes combined with gas expansion.⁽⁷⁾

Another feature of work with liquefied gases is the difficulty of keeping any product obtained. The influx of heat from the surroundings posed serious problems for the workers in this field. The major advance was made by Dewar (1892) when he invented a vacuum flask to store liquefied gases originally designed by d'Arsonval in 1887.⁽⁷⁾

Production of oxygen on an industrial scale was pioneered by Linde company (1900) who built an air liquefier using ammonia for precooling⁽⁹⁾. Further development have been carried out by the same company in 1902 and 1907 when they erected the single and double-stage air separation column, respectively.

Other companies, Claude, Heylandt and Messer^(9&10) became involved in the air liquefaction and separation of air constituents since 1902.

The first commercial plants for oxygen production by the low temperature distillation method were started up almost simultaneously in England, Germany and France (1912) based on technical developments by Hampson, Von Linde and Claude respectively.⁽⁹⁾

Further improvements to the air liquefaction plants were

added by Linde Co. (1926) who in 1932 introduced expansion turbines for the first time in air separation plants⁽¹⁰⁾.

Kapitsa (1937-39) built a high efficiency expansion turbine permitting air separation on the basis of a low pressure cycle alone. Kapitsa's low-pressure cycle is the basis of all existing tonnage air separation plants.

Continued increase in efficiency and capacity resulted in plants capacities of 70,000 cubic meter per hour in the seventies⁽⁹⁾. In 1985, the world production of liquid oxygen to half million tons with a purity of around 99.5%⁽¹¹⁾

2.4 Heat Exchanger Network Optimization

In the oxygen plant under question, the liquefaction duty is actually performed through network of heat exchangers. Any possible optimizing of this network will support an optimum final design of the plant. Due to the rising costs of energy, the design of an optimal heat exchanger network between hot and cold process streams becomes an important problem. A heat exchanger network is defined as several heat exchangers arranged in series and/or parallel to affect the heat exchange between several hot and cold streams. The target was always to minimize the total area required for heat exchange using a thermodynamic approach. Other economic factors were not taken into consideration.

Early optimization work on heat exchanger networks was reviewed by Hendry et al.⁽¹²⁾ and Raghavan⁽¹³⁾. Some of the early authors used heuristic approaches for stream matching which do not guarantee optimality. A practical method of evaluating the heat exchange among multiple streams /in parallel and in series was developed by Chato et al.⁽¹⁴⁾. This can be used most effectively in the final design stage of a heat exchanger network. Lee et al.⁽¹⁵⁾ used a mathematical approach based on a branch and bound method to synthesize networks and find the optimum. This procedure requires too much time for large-scale problems.

Nishida et al.⁽¹⁶⁾ used an algorithm which they claimed give the least total heat transfer area and, by using a few heuristic rules, reduced it to a final form. Raghavan argued that this method did not guarantee optimality and proposed a Heat Availability Function (HAF) with which the maximum recoverable energy could be computed. An algorithm to determine an optimal network was also presented. In 1978, Linnhoff and Flower⁽¹⁷⁾ presented a method similar to that presented by Raghavan in that they proposed a method which first generated preliminary networks which gave maximum heat recovery and then a final network was evolved using the preliminary networks as starting points. Malhotra et al.⁽¹⁸⁾ used the Discrete Maximum Principle to minimize the total cost of heat exchanger chains in which one cold stream was heated by several hot auxiliary streams. In an extension to this work, Siddique et al.⁽¹⁹⁾ considered the optimization of

a heat exchanger train (as a network sometimes called) in which one cold stream to be heated by several hot streams using the same procedure as in their earlier work. Recently, Siddique et al.⁽²⁰⁾ extended their earlier work to include two cold streams. This method can only be applied to small-scale problems and is thus restrictive.

Parkinson et al.⁽²¹⁾ considered the optimal design of resilient heat exchanger networks. A resilient network is defined as one which can tolerate fluctuations in stream temperatures and flow rates. They presented an algorithm which synthesized networks resilient for all stream fluctuations and minimized network investment costs for the conditions of maximum energy recoverable. In an extension of their earlier work, Parkinson et al.⁽²²⁾ also solved the problem of resilient heat exchanger networks optimization using Monte Carlo simulation.

Hesselmann⁽²³⁾ presented an approach which incorporated the economic aspect of heat exchanger design. A functional relation between minimum investment costs and energy losses was determined and, by plotting the investment costs against energy losses, an optimal network could be determined.

The above approaches to the problem of heat exchanger network optimization rely on large computer algorithms and are time consuming. A hand application technique has been used by Linnhoff and Turner⁽²⁴⁾ and Hindmarsh et al.⁽²⁵⁾ to

optimize various process systems, including heat exchanger (or heat-recovery) systems. In 1980, Chato et al.⁽²⁶⁾ presented a simpler method based on the second law of thermodynamics, using the load curves for a particular problem. The method can be easily performed on a graph.

In this literature survey, the ideas and sometimes the expressions of the cited papers were used.

Chapter Three

METHOD OF ANALYSIS

It has been understood that oxygen production is compromised of four processes: compression, heat removal, liquefaction, and rectification of air. For these processes to be evaluated and analyzed, a large set of thermodynamic calculations are needed. The multiple parameters of plant circuit (like flow rate, pressure, temperature, product purity, product state, etc.) make it more complicated.

To perform necessary calculations for the plant parameters when operating under steady state, it has been thought of a computer program which is able to simulate the actual design of the plant as close as possible. Through the plant simulation program, a large set of runs could be performed in order to evaluate the circuit parameters under a set conditions of operation.

To get the required program, two options exist. The first one is to construct an independent, specific and comprehensive program which should meet all details given by the plant design. Although it is direct and attractive, this option calls for a tedious and complicated work because of the wide variety of processes and equipment. Even, the resulting program will still satisfy only the plant under

question , which in turn seems a simple gain when compared with the efforts behind.

The second option suggests to search for a program among several of them available in the market.; Fortunately, the Faculty of Engineering and Technology at the University of Jordan possesses and provide a quality computer software.

Finally, after the investigation of different technical softwares available at the faculty, a computer simulating program referred to as COMPUTER CODE has been chosen to perform the simulation of the oxygen plant. In the coming section, CODE is briefly described.

3.1 CODE Program

CODE is a steady state process simulator that calculates heat and material balances around process equipment. It is able to

1. prepare process design,
2. analyze design alternatives,
3. predict the effect of changes on plant operating conditions,
4. optimize energy consumption and
5. eliminate bottlenecks and increase throughput.

CODE contains a database with physical properties of 427 pure substances. The user can also create his own database containing other components by supplying the required physical properties.

Flowsheets containing up to 100 streams and 50 pieces of equipment can be simulated. With the aid of 16 built-in equipment modules, it is possible to model virtually all process plants. A brief description of each equipment module will follow.

3.1.1 CODE skills

These skills are listed in the CODE menu shown below.

CODE MENU	
A.> Input/Edit Data File
B.> Input Data Checking
C.> CODE Flowsheet Simulation
D.> Run Time Message
E.> Report Generation
F.> Pure Components Databank
G.> HP Plotter Driver
H.> Apple Laser Writer Driver
I.> List Data Files
J.> Type Files
K.> Tray Sizing Input
L.> Exit - Return to DOS

This menu is the interface with different phases of the package.

A. Input/Edit Data File

This is the user interface with the system. With Option A

one can create new problems, review results of problems already run, or build case studies or make modifications of previously run problems.

B. Input Data Checking

Once a problem has been created or modified, this phase of system is used to check or detect input errors in the problem data.

C. CODE Flowsheet Simulation

This is the computational phase of the program. Phase C reads the problem data file, performs the calculations, and records the results in the problem data file.

D. Run Time Message

This program allows to review any screen messages or error messages which may have occurred during a problem run. Phase D should be run each time when the problem calculations are performed.

E. Report Generation

This phase reads the problem output file and generates a report containing the problem results.

F. Components Databank

This program accesses the physical properties in the pure component databank. It allows inspection of the databank and the addition of new components and pseudo components.

This program provides an interface to the Hewlett-Packard 7400 series plotters. This phase is used to generate a graphical hard copy of the flow sheet outputs.

H. Apple Laser Writer Driver

This program provides an interface to the Apple Laser Writer. It is used to generate graphical hard copy flow sheet outputs.

I. List of Data Files

This program will list all data files (files with .DAT extension) on the default directory.

J. Type Files

This program will print a file to the screen. This option can be used to type out various files, such as Data, Result, or Report files.

K. Column-Tray Sizing

This program allows to enter data for tray sizing. It must be executed before the Report program.

L. Exit - Return to DOS

This program allows users to leave CODE and return to DOS.

3.1.2 Run procedure

1. Through phase A a data file is created. By the graphics option within this phase a set of equipment modules are selected, laid down and then connected to each other according to the problem being dealt with. Both equipment and streams are automatically numbered. Necessary information are provided by using the spreadsheets option. Local menus are so helpful to indicate the required information either for equipment or streams. Of course, the user should begin with specifying the components handled by his process(es) and thermal options of solution.
2. Phase B is used to survey errors accompanying step # 1. If errors are assigned, phase A is recalled to correct them or their causes. When phase B shows a well refined data file, it becomes possible to move further to the next step.
3. Phase C is selected and the required data file is assigned. Calculations will be performed upon a number of loops depending on how far entered data can help accuracy.
4. Phase D is then loaded to display errors detected during performing the calculations.
5. Using phase E a report file is generated. There is a variety of report files : comprehensive, stream details only, equipment details only, both stream and equipment

details or what is referred to as the USOR report.

3.1.3 Equipment modules

The following list introduces the equipment modules available in CODE.

ABSR : Absorber-stripper module using absorption and stripping factor method of Edmister.

ADBF : Two-phase equilibrium flash module with a third product stream containing a free water phase in hydrocarbon systems.

COLM : Comprehensive, rigorous multistage equilibrium routine using a modified simultaneous corrections algorithm. COLM can handle up to 300 stages and is relatively fast for problems with few components. It may be used for simulating absorbers, and fractionators.

CTRL : Flow sheet controller with feed forward and feedback control.

DISC : Short-cut distillation rating or design module. In the design mode, three options are available

for feed tray location. In the rating mode, the fractional recovery of the light key component is computed.

DIST : Component splitter module.

DVDR : Stream and component divider module. It divides one input stream into two to six streams with flow or proportional control and arbitrary component separation.

ENTM : Isentropic compressor or expander for vapor or gas.

EVLP : Phase envelope module which generates dew point, bubble point, and condensing curves.

FHTR : Fired heater module controlling either output temperature or heat addition.

HXER : Multi-pass heat exchanger module with ten modes. Change of phase is allowed and single- or two-stream cases can be used.

MIXR : Rigorous mixing point module with flash equilibrium at the output.

MMIX : Simple stream adder module. It should always be followed in the layout by an ADBF if thermal

conditions are important.

PUMP : Liquid pump or multistage polytropic vapor compressor module.

REAC : Simple stoichiometric reactor module with heat of reaction.

TOWR : Fast, comprehensive , multistage equilibrium routine using a state-of-the-art "inside-out" algorithm. This module should be used instead of COLM to save run-time. TOWR is limited to 100 stages. It may be used for simulating absorbers, strippers, reboiled absorbers, and fractionators.

VALV : Downstream pressure controller module.

The modules have been used in simulating the plant are : DVDR, ENTM, HXER, MIXR, PUMP, TOWR and VALV.

3.2 Plant Simulation

This section describes the simulation steps with reference to Figure 3.1.

Prior to the implementation of plant simulation the following assumptions have been established :

- a) The plant operates under steady-state conditions.
- b) Air is assumed to be a binary mixture of only nitrogen and oxygen with molar ratios of 81% and 19%, respectively. This means that other constituents of air (water vapor, carbon dioxide other hydrocarbons, argon, helium, etc.) were neglected based on their very small total contribution to an air sample when compared with that of the two major constituents (N_2 and O_2).
- c) Also, air is assumed to follow the laws of perfect gas when necessary for the simulating program.
- d) Refrigeration losses to ambient were neglected along the heat transfer blocks of the simulated layout.
- e) The whole quantity of air fed to the compressor is processed by all the rest of equipment; i.e. no purging losses are allowed.

Figure 3.2 shows the scheme of equipment connection.

- Air compressor

In the plant description, its operation has been reported to be normal with a volumetric flow rate of air supplied for separation ranging from $750 \text{ m}^3/\text{hr}$ to $1056 \text{ m}^3/\text{hr}$. A flow rate of 40 mol/hr ($961.62 \text{ m}^3/\text{hr}$ at 20°C and 1.0 atm) has been selected. Under temperature of 15.6°C and pressure of 1.0 atm , this is equivalent to $947.608 \text{ m}^3/\text{hr}$. Both values are in good agreement with the range specified before.

The four-staged, inter-cooled compressor is simulated using the two equipment modules ENTM and HXER.

The temperature increase of the compressed air due to deviation from isothermal compression has been limited to the final (fourth) stage of compression.

According to Figure 3.1, ENTM # 1 through ENTM # 4 stand for the respective four compression stages. Heat exchanger modules HXER # 5 through HXER # 7 simulate the respective three inter-coolers of the compressor, while HXER # 8 represents the terminal cooler.

In fact, the compression process could be simulated as a one-staged process without altering the final state of air at compressor delivery.

- Preliminary heat exchanger

Through this exchanger, air is precooled to its

purification temperature (10 - 12°C). Moisture separators have not been included in the analysis because moisture (water vapor) was not assumed as a constituent of air.

According to the operating principle of this exchanger, direct heat exchange only occurs between the hot stream (air) and one cold stream (nitrogen). The other cold stream (oxygen) is not allowed to exchange heat directly with air; its effect is to sustain the temperature of nitrogen as long as the heat transfer mechanism is valid.

Three units of equipment module HXER are connected to simulate the function of the preliminary heat exchanger.

- Air purifier

Following the assumption that air is a binary mixture of only nitrogen and oxygen, air purifier is automatically dropped off. The only effect produced by this equipment is the temperature rise of air across the purification process by 2°C. This effect is considered in the analysis. Equipment module HXER # 13 is used to provide the required temperature rise.

- Heat exchanger/upper half

For this exchanger, the part before the branching of air stream will be termed as upper half while that part after the branching will be termed as lower half.

Through the upper half of the heat exchanger air is to be cooled to the inlet temperature of the turboexpander.

Two units of equipment module HXER (HXER # 11 HXER # 12) are arranged to represent a three-stream heat exchanging unit. A divider module (DVDR # 14) precedes the formed unit to split the air stream into two streams. Splitting ratio complies with thermodynamic equilibrium which requires that air (as a single stream) has a unique temperature at any specified location of the heat exchanger.

- Heat exchanger/lower half

Through this section, part of the processed air (40%) is allowed to exchange heat with the two cold streams (N_2 and O_2). A divider module (DVDR # 17) precedes the exchanger section to partition the air stream. The rest of the air (60%) is directed to the turboexpander (ENTM # 18). A divider module (DVDR # 19) is again used to split the main hot stream in order to have the arrangement of three-stream heat-exchanging section.

Beyond this section of the heat exchanger, the outgoing hot stream is throttled by equipment module VALV # 23.

The outlet pressures of both turboexpander and throttling valve comply with the requirements of the separation unit

- Lower column

An equipment module, TOWR # 24, is used to simulate the lower column of the separation unit. The selected mode of operation guarantees the distillation fraction assigned by the manufacturer.

- Sub-cooler A21

This sub-cooler is also a three-stream equipment. Its function is to sub-cool each of the two streams (liquid

nitrogen, liquid oxygen-riched air) leaving the lower column. HXER # 25 and HXER # 26 are the simulating modules.

Across both outlet streams, two valve modules (VALV # 28 and VALV # 29) throttle the running streams to the pressure range required by the upper column follows.

- Sub-cooler A6

For sub-cooling the main product (liquid oxygen) emerging from the upper column by the incoming liquid oxygen-riched air, HXER # 27 exists to simulate the required sub-cooler.

- Upper column

A tower module (TOWR # 30) is used to represent the upper rectifying column. The selected operating mode guarantees the flow rate of the final product as assigned by the manufacturer.

A pump module PUMP # 31 is used across the sub-cooled liquid oxygen stream for the latter to be compressed to the required handling pressure.

The two streams outgoing from the upper column (gaseous nitrogen and liquid oxygen) are the same cold streams encountered by the heat exchangers later.

HXER # 33 & HXER # 34 are being used as temperature controllers to isolate the liquefaction block (preliminary heat-exchanger, heat exchanger and turbo-expander) from temperature variations that may occur at the exit of the rectification block whenever some parameters of the latter are altered.

A divider module, DVDR # 32, is located across the nitrogen stream at the entrance to the preliminary heat-exchanger. This divider corresponds to an actual one in the plant layout design. Its function is to make part of the nitrogen stream bypass the preliminary heat-exchanger for the outlet temperature of the precooled air not to violate its assigned range (5 - 7°C).

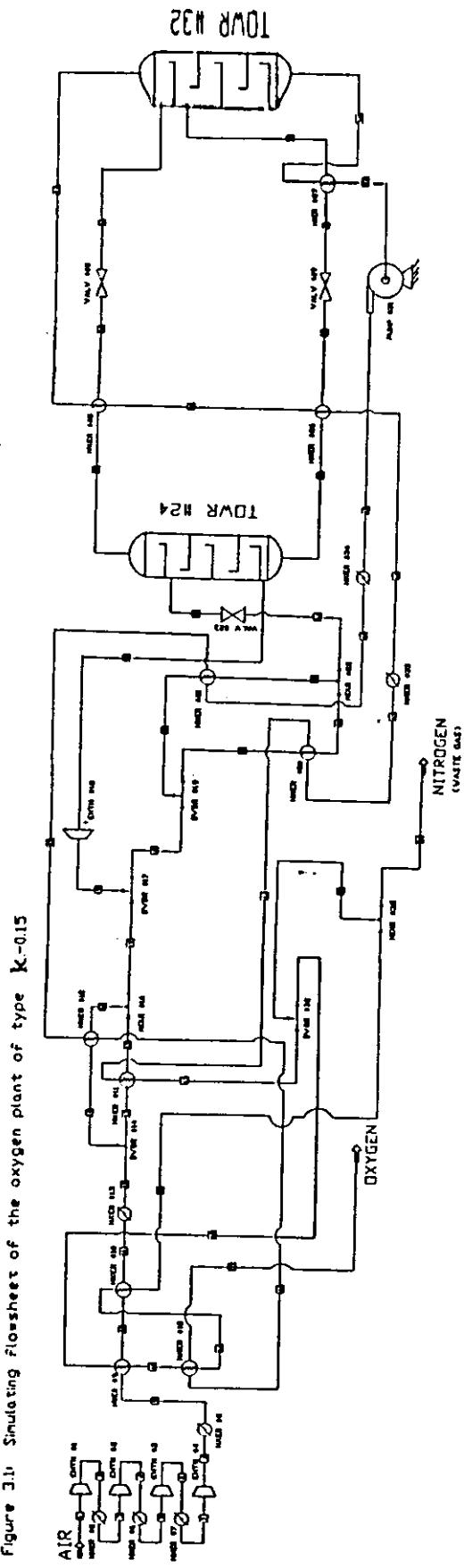


Figure 3.1: Simulating flowsheet of the oxygen plant of type k-0.15

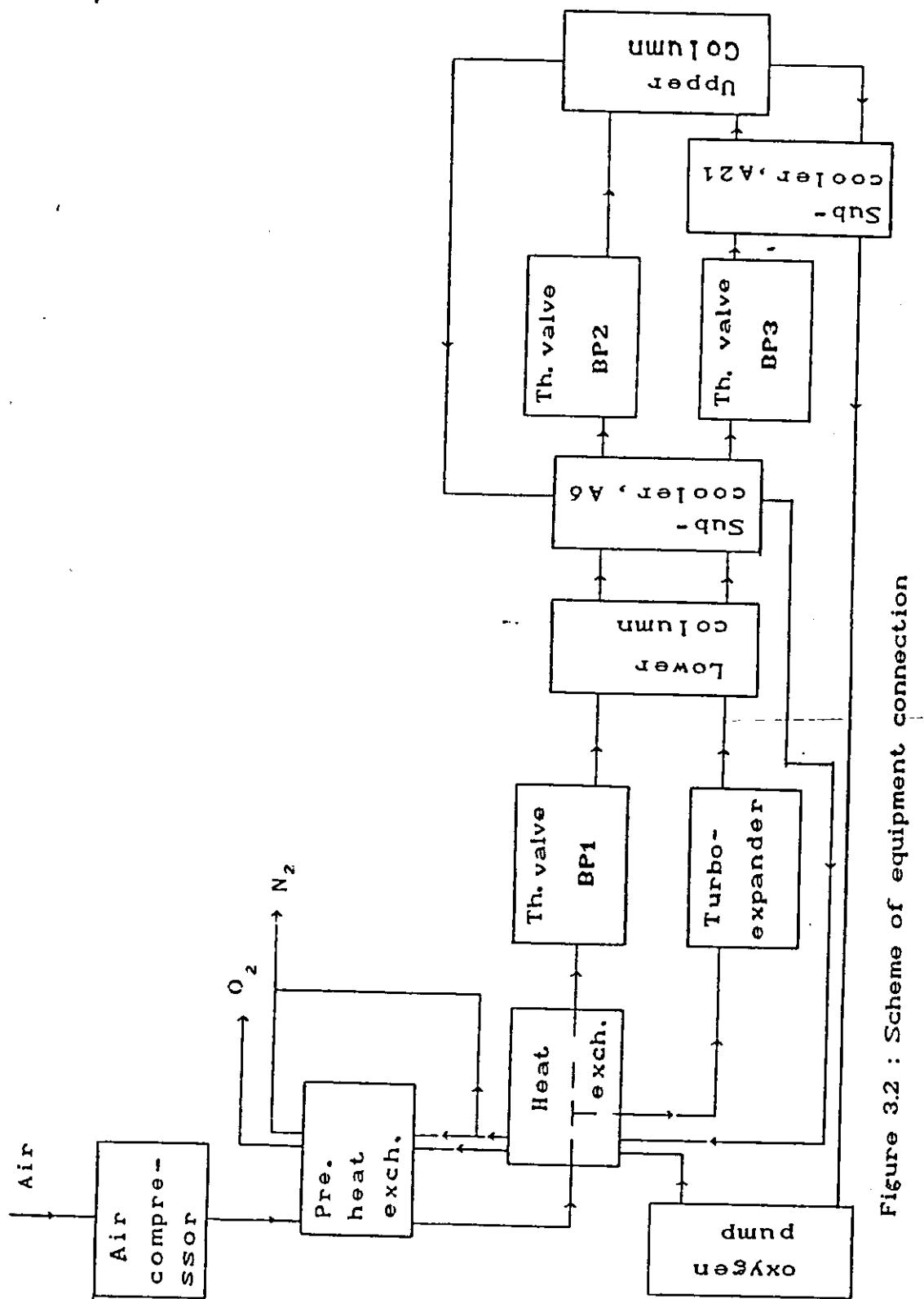


Figure 3.2 : Scheme of equipment connection

Chapter Four

INVESTIGATION OF THE EXISTING PLANT

4.1 Introduction

4.1.1 Optimum process design

Air-separation process may be considered as a series of unit operations. The various components are closely integrated by the process designer and optimized to suit the particular requirements and to suit available energy source(s).

The important factors which influence the selection of the process within the oxygen plant are

- energy consumption,
- product quantity,
- investment cost,
- product purity,
- product state (temperature, pressure, phase)
- continuity of product supply,
- plant flexibility,
- maintenance cost and
- permissible plant dimension.

These factors remain in continuous conflict when searching for an optimum design. The primary factor

determining the cost of the product is the energy cost.

As to our plant, energy is lost due to the following causes

- a) low efficiencies of energized equipment (compressor, pump, electric heater)
- b) heat in-leak to cold blocks due to the imperfect insulation.
- c) incomplete heat transfer resulting from imperfect heat exchanger network.

To improve the efficiencies of the compressor, the pump and the electric heater, it is necessary to increase the capital cost of the plant. Also, to improve the efficiency of the insulation, more expenditure is necessary. However, increasing the efficiency of the heat exchanger network does not necessarily require higher cost.

The plant circuit shows that heat is being exchanged across three locations within the plant. These locations are

1. compressor inter-coolers,
2. sub-coolers contained in the air-separation unit and
3. cooling and liquefaction block (preliminary heat exchanger, heat exchanger and turbo-expander).

A compressor inter-cooler is a water-cooled air cylinder.

Its effectiveness depends on inlet temperature of the cooling water which in turn is determined by the flow rate of water.

For the sub-coolers contained in the air-separation unit, the configuration of streams is controlled by the operation principle of the rectifying columns and so their efficient operation is an integrated part of an effective rectification block.

Since the circuit arrangement does not employ an independent refrigerating system for the liquefaction of air, the cooling and liquefaction block becomes of most importance to focus on. Within this block, expansion of air by the turbo-expander provides the necessary cooling effect. After the separation of air into liquid oxygen and gaseous nitrogen, a heat exchanger network is used to utilize the coolness within the separation products to cool the incoming air.

The following section is an energy integration analysis to investigate the present layout of the plant and to suggest improvements that may be necessary.

4.2 Energy Integration Analysis

4.2.1 Method of approach

The starting point for an energy integration analysis is

the calculation of the minimum heating and/or cooling requirements for a heat exchanger network. These calculations can be performed given the input and output streams without having to specify any heat exchanger network. Similarly, we can calculate the minimum number of heat exchangers required to obtain the minimum energy requirements without having to specify a network. Then the minimum energy requirements and the minimum number of heat exchangers provide targets for the subsequent design of heat exchanger network.

Two well-known approaches are available to estimate the minimum requirements of energy; they are

1. Composite Curves and
2. Problem Table.

In the first approach a single composite of all hot and a single composite of all cold streams are produced in the temperature-enthalpy (T-H) diagram, and handled just in the same way as a two-stream problem. This method will not be used because it would require graph paper and cut and paste approach (for sliding the graphs relative to one another) which would be messy and imprecise.

The second approach which has been published by Linnhoff and Flower (1978)⁽¹⁷⁾, is used for considering the energy

targets algebraically. This approach will be adopted because of its simplicity and applicability to computer handling.

4.2.2. Stream definition

Figure 4.1 shows the connection scheme within the stream-equipment layout of the cooling and liquefaction block in the plant. Also, design-temperature ranges at the equipment boundaries are shown.

Figure 4.2 illustrates the formation of streams according to which the coming analysis is based.

Considering the fluid type, three streams are formulated, they are air, oxygen and nitrogen. The first one is hot stream while the other two are cold streams. Physically, the hot stream, H, 1 is splitted into two parts at the middle of the heat exchanger, A4. The first part, H2, moves through the turboexpander while the other part, H3, is throttled by valve BP1.

The oxygen stream conserves itself as one cold stream, C1. The nitrogen stream, C2, is divided into two parts. The first part travels through the preliminary heat exchanger. The second part bypasses the exchanger such that the outlet temperature of incoming air is controlled. The latter part is excluded from the formed streams. The part which goes through the exchanger is chosen to be the third cold stream,

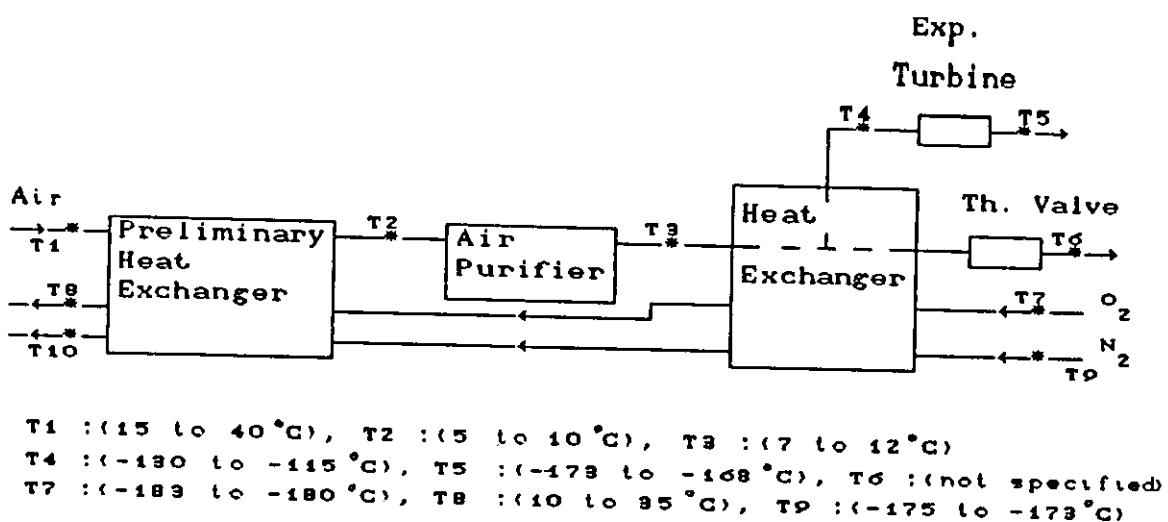


Figure 4.1 : stream-equipment layout

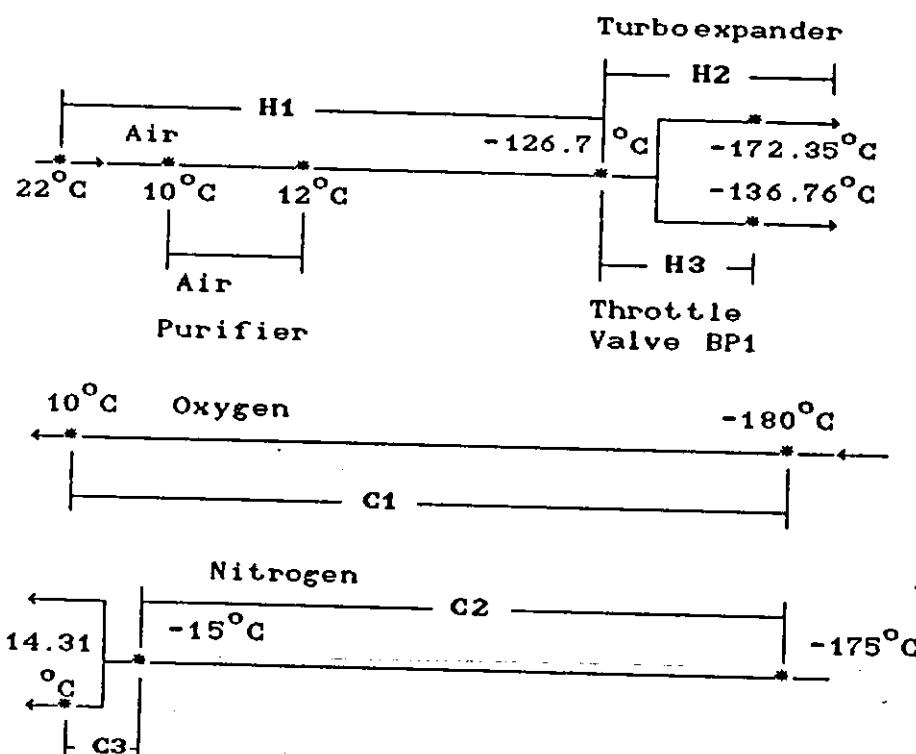


Figure 4.2 : Stream formation

C3.

Using the flowsheet shown in Figure 3.1, hot and cold streams are defined as follows :

Hot stream no.1; H1 (-22.0 to -126.69°C)

Comprises air streams numbered in the flowsheet as 9, 10, 11, 12, 13, 14, 15, 16 and 17.

Hot stream no.2; H2 (-126.69 to -172.35°C)

Comprises air streams numbered as 19 and 26.

Hot stream no.3; H3 (-126.69 to -136.76°C)

Comprises air streams numbered as 18, 20, 21, 22, 23 and 24.

Cold stream no.1; C1 (-180.0 to 10.0°C)

Comprises oxygen streams numbered as 42, 45, 46 and 51.

Cold stream no.2; C2 (-175.0 to -15.0°C)

comprises nitrogen streams numbered as 43, 44, 47.

Cold stream no.3; C3 (-15.0 to 14.31°C)

Comprises nitrogen streams numbered as 48, 49, 36 and 38.

The temperature-enthalpy diagram can be used to represent the thermal characteristics of process streams, as illustrated in Figure 4.3. When a differential heat flow dQ

is added to a process stream, it will increase its enthalpy (II) by $CP \cdot \Delta T$, where,

CP = heat capacity flow rate (kW/K)

= mass flow rate, \dot{m} (kg/s) * specific heat, C_p (kJ/kg.K)

ΔT = differential temperature change

Recalling the assumption of air as an ideal gas and assuming that CP = constant then, the following balance becomes possible,

$$\dot{m}^0 C_p \Delta T = CP \cdot \Delta T \quad (4.1)$$

Hence dQ will be the same as ΔH .

Then CP may be calculated as follows,

$$CP_i = \Delta H_i / \Delta T_i \quad (4.2)$$

where i denotes the stream number.

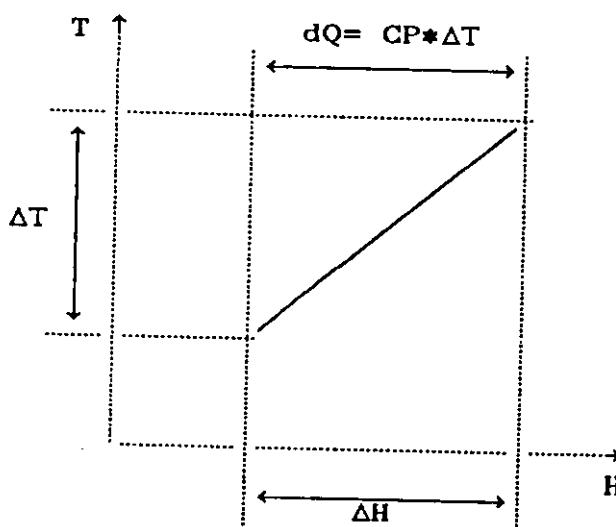


Figure 4.3 temperature — enthalpy diagram

Table (4.1) shows the streams defined before along with their supply and target temperatures, enthalpies and heat capacity flow rates.

TABLE 4.1 - STREAM DATA

Stream No & Type	T_s ($^{\circ}$ C)	Enthalpy at T_s (kW)	T_t ($^{\circ}$ C)	Enthalpy at T_t (kW)	C_p (kW/ $^{\circ}$ C)
H1, hot	22.0	222.82	-126.69	161.10	0.4151
H2, hot	-126.69	96.66	-172.35	89.94	0.1472
H3, hot	-126.69	64.44	-136.76	59.63	0.4776
C1, cold	-180.00	16.72	10.00	38.55	0.1149
C2, cold	-175.00	130.99	-15.00	173.98	0.2687
C3, cold	-15.00	57.41	14.31	59.96	0.0870

4.2.3 Calculation of minimum energy requirements

A very simple way of incorporating the second law of thermodynamics into the energy integration analysis was presented by hohmann⁽²⁷⁾, Umeda et al⁽²⁸⁾ and Linnhoff and Flower⁽²⁹⁾. Their analysis will be followed.

Initially, a minimum driving force (temperature difference), ΔT_{min} should be specified. The economical value of ΔT_{min} is usually estimated by trading off energy against capital cost. However, an experience value of ΔT_{min} will be adopted. For a refrigeration system the experience

(78)

value of ΔT_{\min} is 5°C . Now, we can establish two temperature scales on a graph, one for hot streams and the other for cold streams, which are shifted by 5°C . See Figure (4.2).

Then the stream data are plotted on this graph. Next we establish a series of temperature intervals that correspond to the heads and tails of the arrows, i.e., the inlet (supply temperature, T_s) and the outlet (target temperature, T_t) of the hot and cold streams as given in Table (4.1). Temperature intervals are superimposed.

Setting up the intervals in this way guarantees that full heat interchange within any interval is possible since the driving force is already adequate. Hence, each interval will have either a net surplus or a net deficit of heat as dictated by enthalpy balance, but never both. This is shown in Table 4.2. Knowing the stream population in each interval (from Figure 4.4), enthalpy balances can easily be calculated for each according to

$$\Delta H_i = (T_i - T_{i+1})(\sum C_p_{\text{cold}} - \sum C_p_{\text{hot}})_i \quad \dots \quad (4.3)$$

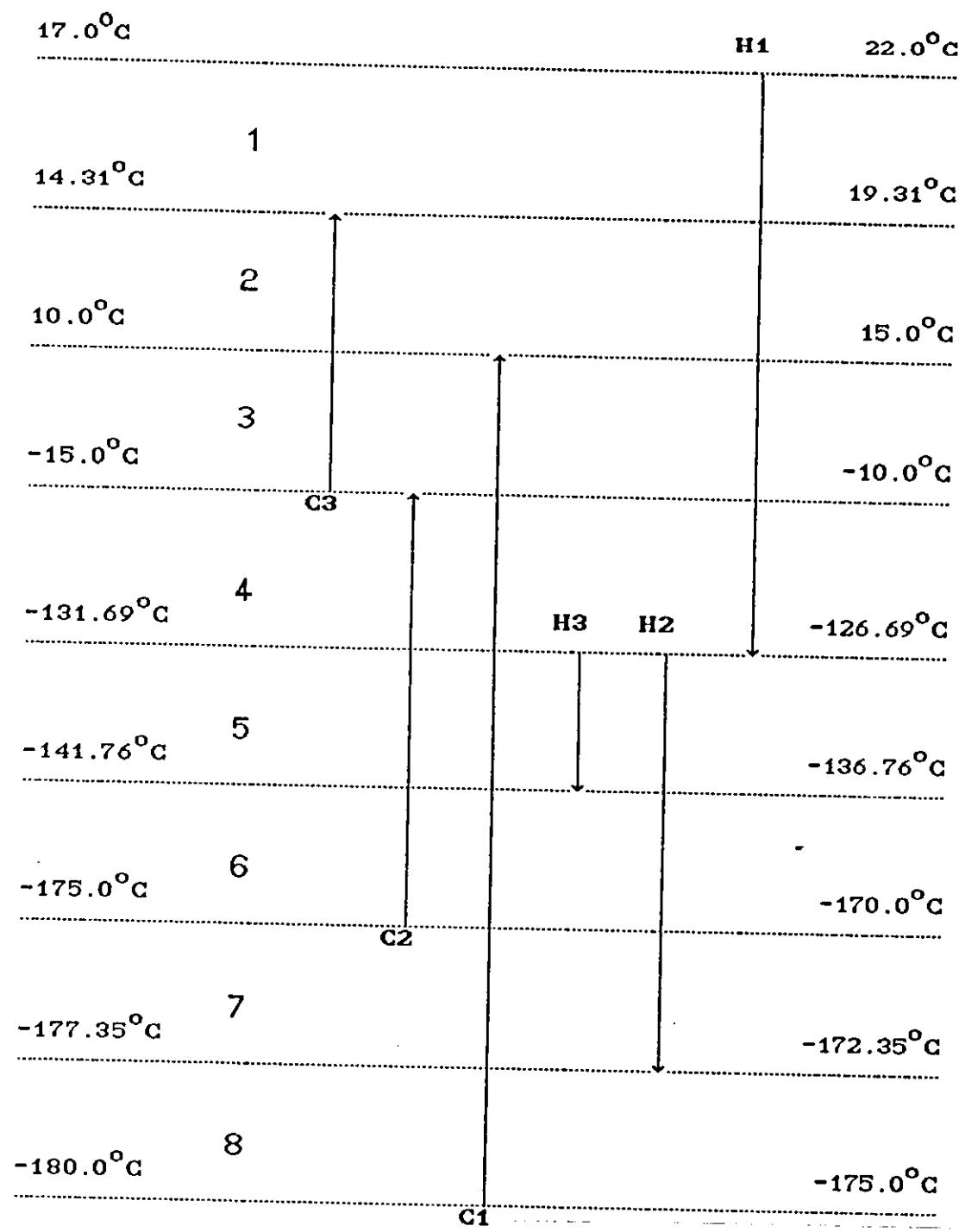


Figure 4.4 – Temperature interval analysis

for any interval i . The last column in Table 4.2 indicates whether an interval is in heat surplus or heat deficit. It

would be possible to produce a feasible network design based on the assumption that all "surplus" intervals rejected heat to cold utility, and all "deficit" intervals received heat from hot utility.

Now, considering the results shown in Table 4.2, Figure 4.3 explains what is called **HEAT CASCADE PRINCIPLE** by which it is possible to evaluate the minimum requirements of energy for a specified process..

TABLE 4.2 - TEMPERATURE INTERVAL ANALYSIS

Interval No. (i)	$T_i - T_{i+1}$ (°C)	$\Sigma CP_{\text{cold}} - \Sigma CP_{\text{hot}}$ (kW/°C)	ΔH_i (kW)	Surplus or Deficit
17.0°C				
14.31°C	1 2.69	- 0.4151	-1.1166	Deficit
10.0°C	2 4.31	- 0.3281	-1.4141	Surplus
-15.0°C	3 25.0	- 0.2132	-5.3300	Surplus
-131.69°C	4 116.69	- 0.0315	-3.6757	Surplus
-141.76°C	5 10.07	- 0.2412	-2.4289	Surplus
-175.0°C	6 33.24	+ 0.2364	+7.8579	Deficit
-177.35°C	7 2.35	- 0.0323	-0.0759	Surplus
-180.0°C	8 2.65	+ 0.1149	+0.3045	Surplus

Assuming that no heat is supplied to the hottest interval (1) from hot utility, a surplus amount of 1.1166 kW is cascaded into interval (2). Interval (2) has a 1.4141 kW surplus which will be joined with the income from interval

- (1) to produce a heat flow of 2.5307 kW passed to interval
(3). The cascading process continues through the rest of intervals as shown in Figure 4.5.

By completing the cascading process, the minimum energy requirements are shown to be 5.8788 kW cold.

Appendix B shows the results of the temperature interval analysis for values of ΔT_{min} ranging from 0.0 to 10.0°C .

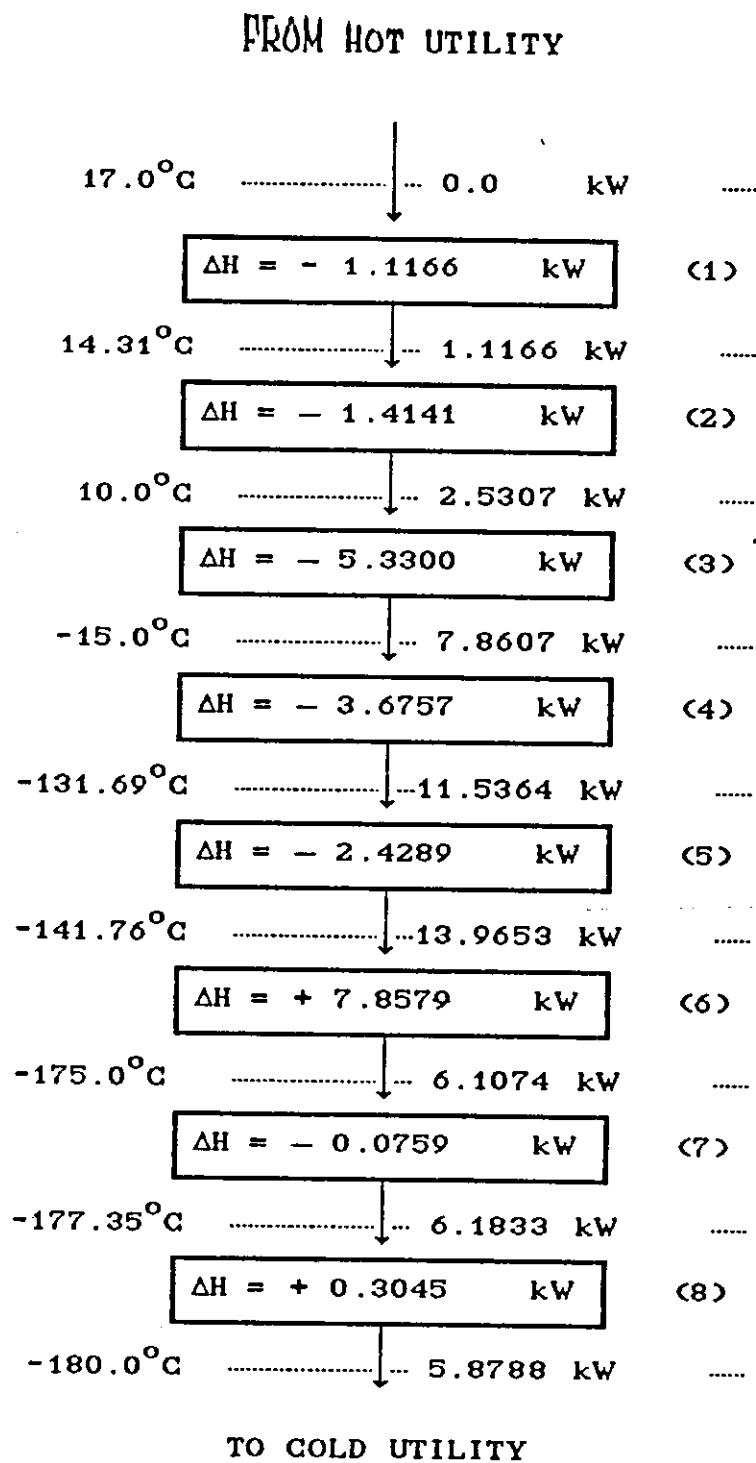


Figure 4.5 - Heat cascading

4.3 Maximum Energy Recovery Design

There are other characteristics of the optimum heat exchanger network. An optimum network should use the minimum area of heat transfer. Also, the number of employed heat exchangers within such network should be minimum. These two characteristics could be investigated for a certain design by using an analysis called MAXIMUM ENERGY RECOVERY (MER) design.

4.3.1 Stream matching / exchanger network

The ruling criterion , is to maximize the individual loads of matches (exchangers) in order to have the minimum number of operating units. This aspect is simply imposed by the fact that the capital cost increases as the number of employed units increases.

Figure 4.6 illustrates the working streams and their enthalpies. The actual cooling utility is also shown.

H 1 air 61.725 kW	H 2 air 6.72 kW	H 3 air 4.805 kW
C 1 oxygen 21.83 kW	C 2 nitrogen 43.00 kW	C 3 nitrogen 2.35 kW

Turbo - expander 5.8764 kW 6.4 kW min rated

Figure 4.6 – Stream enthalpy

A simplification of the configuration shown in Figure 4.6 is possible; that is to let both C2 and C3 be one stream, say C2. This unification is justified as follows

- a) physically, C2 and C3 are one stream (both are nitrogen and also have the same pressure) and
- b) outlet temperature of C2 is the same as the inlet temperature of C3.

When the plant is under operation, the partitioning of nitrogen stream into two streams (C2 & C3), Figure 1.1, occurs at inlet to the preliminary heat exchanger.

In order to sustain thermal equilibrium, the newly formed stream C2 must have

- a) a total enthalpy equivalent to the summation of the

individual enthalpies of both C2 and C3 and

- b) a target temperature similar to that of the two streams when mixed at the exit from the preliminary heat exchanger, (1.2°C). Then,

$$\text{Enthalpy of C2} = \text{Enthalpy of C2} + \text{Enthalpy of C3}$$

$$\begin{aligned} &= (\text{CP} * \Delta T)_{\text{C2}} + (\text{CP} * \Delta T)_{\text{C3}} \\ &= (0.2687)(175 - 15) + (0.087)(15 - 12) \\ &= 45.341 \text{ kW} . \end{aligned}$$

The resulting heat capacity flow rate of the new stream will be

$$\begin{aligned} \text{CP} &= \frac{\Delta H}{\Delta T} = \frac{45.341 \text{ kW}}{(175 + 1.2)^{\circ}\text{C}} \\ &= 0.2573 \text{ kW}/^{\circ}\text{C} \end{aligned}$$

Now, the streams configuration may be redrawn as illustrated by Figure 4.7.

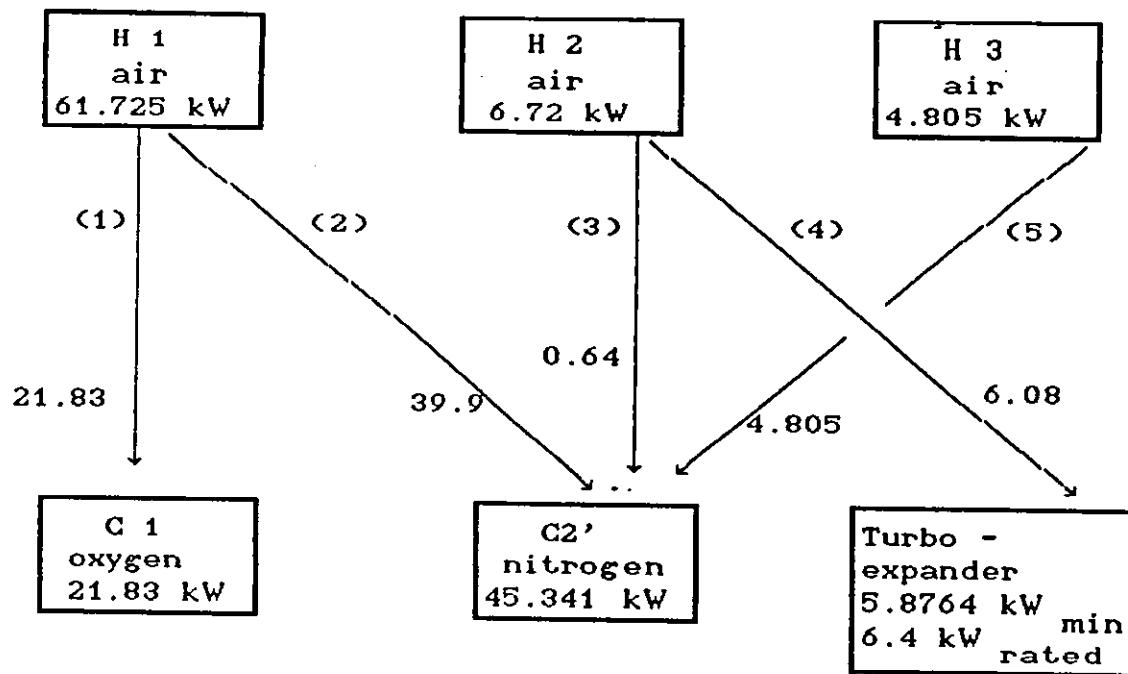


Figure 4.7 – Stream matches

The Figure also illustrates 5 matches started from the cold end (oxygen stream) with their loads.

In this configuration, it is noticed that match (3) is relatively so small to deserve an independent heat exchanger. H3 could be cooled further if the load of match (3) is reloaded upon match (5). This in turn, will leave the cooling load required by H2 handled by only one match; that is match (4). What we have truly done is that we increased the minimum cooling requirement a bit. There is nothing being sacrificed, since the rated capacity of the available cooling utility (turboexpander) allows for such an increase,

recall that its rated capacity is 6.4 kW min.

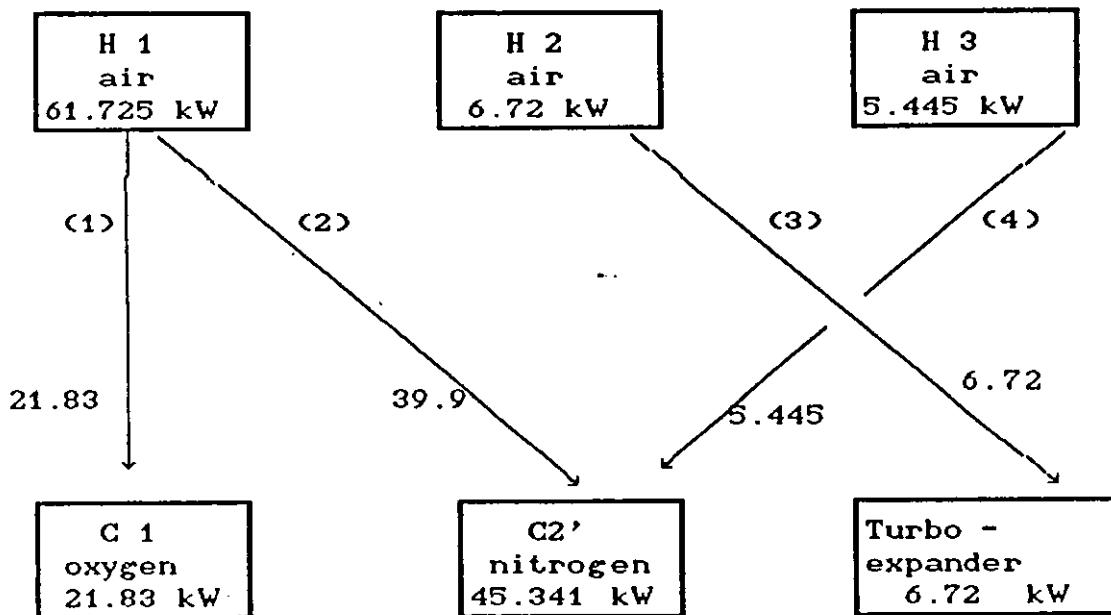


Figure 4.8 -Final configuration

Figure 4.9 shows the final configuration of stream matches and the relevant loads. This configuration indicates that three heat exchangers are needed plus a cooling utility.

It is a common practice to start a design at its most restrictive location. The hot air stream H1 should be interrupted at a certain point; that is, air purifier with a predetermined temperature interval. A temperature increase of 2°C across the air purifier is reported by the manufacturer. The inlet temperature to this unit ranges from 5°C to 10°C .

As a result of this restriction, it becomes necessary to implement matching of streams starting from the hot end bearing in mind that both matches (1) and (2) will be upon two stages to allow for cooling interruption imposed by the air purifier.

Match 1/ 1st stage : H1 is let to exchange heat with C1 such that the outlet temperature of H1 permits a feasible matching with C2; the minimum temperature difference across the warm end of a match (a heat exchanger) should be equal (or even greater than) $\Delta T_{min} = 5^{\circ}\text{C}$ as presumed at the beginning of analysis. This match stage will be denoted by 1-1.

Match 2/ 1st stage : H1 matches with C2 after it has been cooled through match 1-1. This match stage will be denoted by 2-1.

Along the match stages; 1-1 & 2-1, H1 is to be cooled from its supply temperature down to 10°C which is the maximum permissible value of inlet temperature of air to the purifier. The accompanying temperature drop is chosen to be divided equally among the two match stages.

Supply temperature of C1 is

$$\langle T_s \rangle_{C1} = \langle T_t \rangle_{C1} - \frac{\langle CP \rangle_{H1} * \Delta T}{\langle CP \rangle_{C1}}$$

$$= 10 - \frac{0.4151 * 6}{0.1149}$$

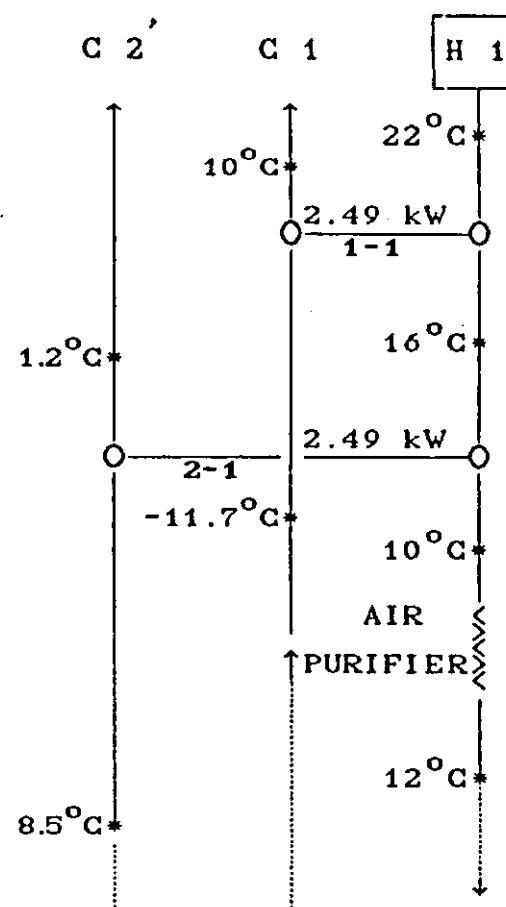
$$= - 11.7^\circ C$$

Supply temperature of C2' is

$$\langle T_s \rangle_{C2'} = \langle T_t \rangle_{C2'} - \frac{\langle CP \rangle_{H1} * \Delta T}{\langle CP \rangle_{C2'}}$$

$$= 1.2 - \frac{0.4151 * 6}{0.2573}$$

$$= - 8.5^\circ C$$



See Figure 4.8.

Figure 4.9 Matches 1 and 2/
first stage

By the end of this stage we have :

Supply temperature of H1 = 12°C,

Target temperature of C1 = - 11.7°C and

Target temperature of C2' = - 8.5°C.

The load left to the 2nd stage of match 1, denoted by 1-2, will be $21.83 - 2.49 = 19.34$ kW. Similarly, the load of the 2nd stage of match 2; denoted by 2-2, will be

$$39.0 + 2.40 = 37.41 \text{ kW. See Figure 4.10.}$$

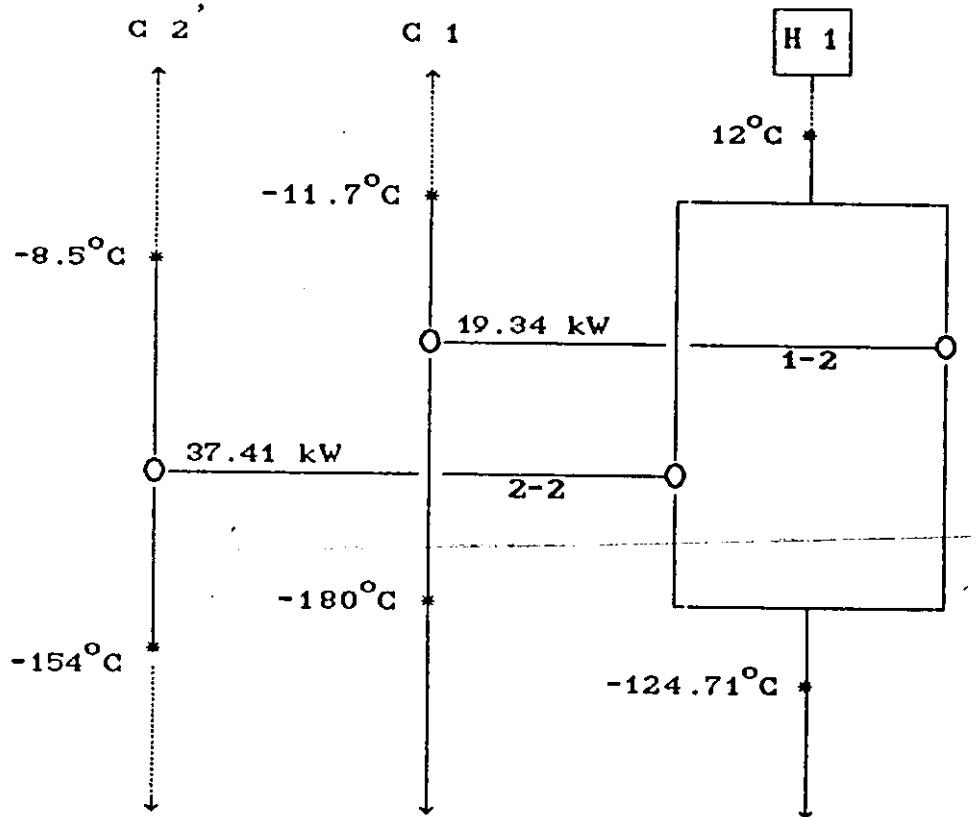


Figure 4.10 - Matches 1 and 2/ second stage

For stream H1 it has been splitted because air stream (H1) actually exchanges heat in parallel with both oxygen stream (C1) and nitrogen stream (C2'). On the other hand, the ratio of splitting should keep air temperature always having the same value of temperature within both resulting parts.

The ratio of air stream splitting is determined in accordance with the decided partial loads. Then :

$$H_1\text{-part for match 1-2} = \frac{19.34}{19.34 + 37.41}$$

$$= 34\% H_1$$

$$\therefore H_1\text{-part for match 2-2} = 1.0 - 0.34$$

$$= 66\% H_1.$$

Now, target temperature of H_1 is

$$\begin{aligned} \langle T_t \rangle &= T_s - \frac{\Delta H}{CP} \\ &= 12.0 - \frac{19.34 + 37.41}{0.4151} \\ &= -124^{\circ}\text{C}, \end{aligned}$$

which confirms with technical characteristics of the turbo-expander; since it includes that the inlet temperature of air to the expander should be within $(-130$ to $-115)^{\circ}\text{C}$.

Supply temperature of C_1 is

$$\begin{aligned} \langle T_s \rangle_{C1} &= \langle T_t \rangle_{C1} - \frac{\langle \Delta H \rangle_{\text{match 1-2}}}{\langle CP \rangle_{C1}} \\ &= -11.7 - \frac{19.34}{0.1149} \\ &= -180^{\circ}\text{C} \end{aligned}$$

Supply temperature of C_2 is

$$\begin{aligned} \langle T_s \rangle_{C2} &= \langle T_t \rangle_{C2} - \frac{\langle \Delta H \rangle_{\text{match 2-2}}}{\langle CP \rangle_{C2}} \\ &= -8.5 - \frac{37.41}{0.2573} \\ &= -154.0^{\circ}\text{C.} \end{aligned}$$

Now, match 4 must handle a load of 5.445 kW still available in stream C2.

Target temperature of H3 is

$$\langle T_t \rangle_{H3} = \langle T_s \rangle_{H3} - \frac{\langle \Delta H \rangle_{H3}}{\langle CP \rangle_{H3}}$$

$$= -124.71 - \frac{5.445}{0.4776}$$

$$= -136.11^\circ\text{C}$$

Supply temperature of C2 is

$$\langle T_s \rangle_{C2'} = \langle T_t \rangle_{C2'} - \frac{\Delta H}{\langle CP \rangle_{C2'}}$$

$$= -154.0 - \frac{5.44}{0.2573}$$

$$= -175.0^\circ\text{C}$$

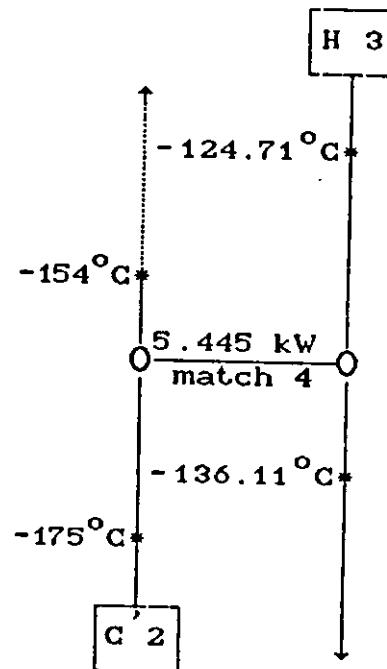


Figure 4.11 — Match 4

A process flowsheet has been established to check the resulting configuration of streams and the relevant matches ; See Figure 4.12.

This flowsheet was exposed to the simulation program of CODE. The results obtained are listed in the Appendix C.

4.4 Column-Tray Sizing

As the simulating program CODE comprises the facility capable for sizing distillation columns, it has been possible to extend our investigation towards the air-separation unit.

A flowsheet segment representing only the double-column rectifying unit is shown in Figure 4.13.

For this segment, the tray sizing calculations were performed using the program phase "Tray Sizing Input". Such calculations aim to estimate the minimum diameters of the distillation columns under operation. The method being used is the Smith-Dresser-Ohlswager technique⁽³⁰⁾.

The program prompts the user to enter the following data: Tray spacing, down comer area, weir length and weir height; then it calculates a diameter for each tray.

The results obtained and shown in Appendix D reveal that:

- The estimated diameter of lower column ≈ 0.30 m.
- The estimated diameter of upper column ≈ 0.40 m.

But the actual diameters of lower and upper columns within the air-separation unit are 0.4 m and 0.5 m, respectively. This means that both of the rectification columns employed in the plant are capable to separate an additional amount of air into oxygen and nitrogen. Also, this

introduces a possibility of expanding the production capacity of the plant while still using the existing air-separation unit. Of course, such expansion should also consider the rated capacities of other existing equipment.

To evaluate the allowance which has already been noticed within both of the rectification columns, CODE program was used. The flow rate of air incoming to the separation unit was increased by increments and then the program phase "Column-Tray Sizing" was run. When the calculations have given column diameters similar to the actual design, a flow rate of air approaching 1.7 times the operating value was permitted. See the results as obtained in the Appendix E.

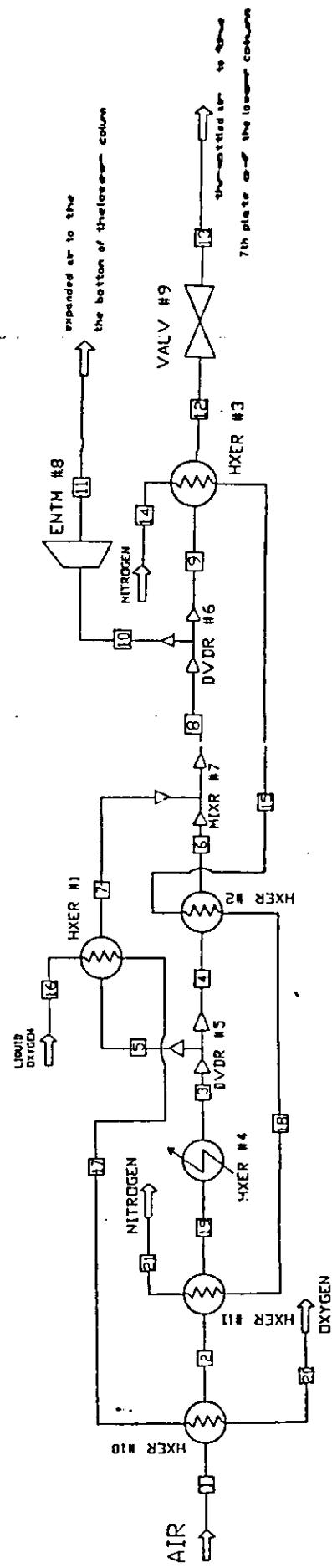
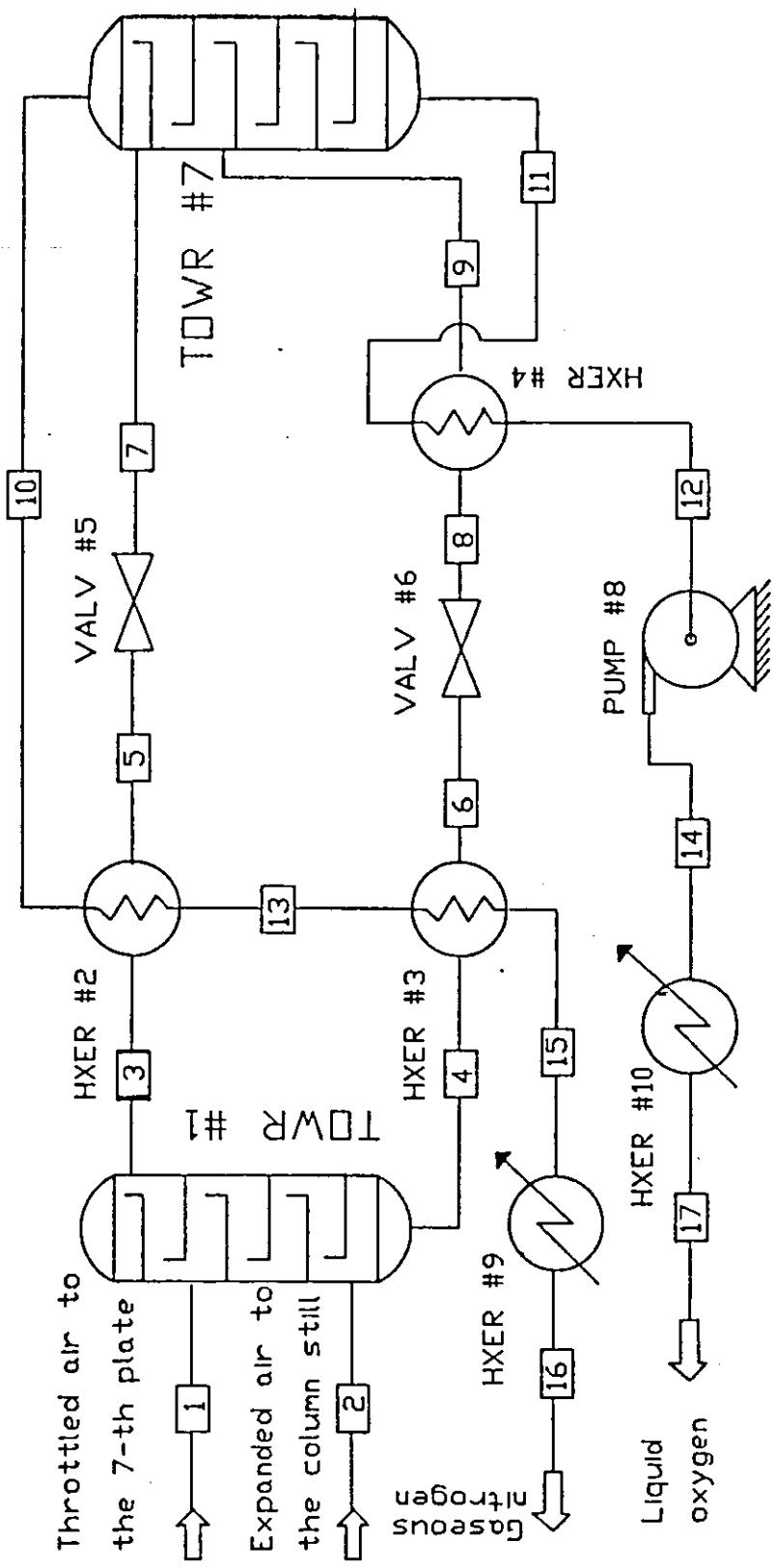


Figure 4.12 : Optimized exchanger-network of the oxygen plant

Figure 4.13 : Tray-sizing of the air-rectification columns



Chapter Five

DISCUSSION OF RESULTS

5.1 Base Design

5.1.1 Plant simulation

The results obtained for the simulating flowsheet, shown in Figure 3.1 which are listed in Appendix A, comply with the technical data of the plant as given by the manufacturer. Oxygen purity is only 2% less than stated in the plant certificate. —This deviation does not affect the energy requirements of the circuit; Oxygen purity depends only on the operation characteristics of the rectifying columns. According to the module TOWR, its routine comprises several options that may produce values of purity as high as desired by the user. However, this requires larger number of computer runs and hence longer time necessary for the calculations to converge.

In regard to the modules simulating the heat exchangers, the most important parameter is the logarithmic mean temperature difference (LMTD) which is denoted in all results by "corrected delta T." The value substituted for the coefficient of heat transfer ($300 \text{ MJ/hr.m}^2 \cong 83.33 \text{ W/m}^2$) was not determined in advance; It is only an assumed value. The target, here, is to estimate the product value of the overall coefficient of heat transfer, U , times the heat transfer

area, A, i.e. "UA" corresponding to a specified LMTD and presumed heat duty, Q, for each heat exchanger.

Values of pressure viewed in all results refer to the absolute pressures.

5.1.2 Evaluation of the heat exchanger network

The question that should be answered in this discussion is whether the heat transfer area used in the heat exchanger network is minimum. To answer this question, one should consider the process flowsheet which has been predicted by the energy integration analysis, Figure 4.12. The total area of heat transfer calculated for this network is then compared with that already exists within the actual design of the plant assuming that the coefficients of heat transfer are kept the same.

The scheme to be applied is as follows:

1. The "UA" product of all actual heat exchanger network are are estimated using the general equation of heat transfer,

$$Q = U * A * \text{LMTD} \quad \dots\dots\dots(5.2)$$

where,

Q : Load of the exchanger, kW,

U : Overall coefficient of heat transfer of the
exchanger, $\text{W/m}^2 \cdot ^\circ\text{C}$,

A : Heat-transfer area of the tube side , m^2 and

LMTD : Logarithmic mean temperature difference across the heat exchanger, °C.

LMTD is known for any heat-exchanging section and Q is obtained from the results of the program calculations. Then, the product "UA" is determined by the equation (5.2). See Table 5.1.

2. Using the same equation, the corresponding "UA" products for the predicted flowsheet in Figure 4.11 can be evaluated using results found in Appendix B. See Table 5.2.
 3. The values of the overall coefficient of heat transfer for the actual design are evaluated as follows

$$U_{actual} = \frac{\text{"UA"}_{actual}}{A_{actual}} \dots\dots\dots(5.3)$$

where A_{actual} is recalled from the technical data given for the plant. See Table 5.3.

4. Then the resulting values of U in step 3 and the corresponding "UA"'s in Table 5.2 are substituted in the

following equation to find the predicted area,

$$A_{predicted} = \frac{"UA"_{predicted}}{U_{actual}} \dots \quad (5.4)$$

See Table 5.4.

TABLE 5.1 : "UA" FOR ACTUAL DESIGN

Section	Simulating Module	Q (kW)	LMTD (°C)	"UA" (W/°C)
<u>Heat Exchanger/</u>				
- <u>upper half</u> :				
* Air - nitrogen	HXER # 11	39.58	30.11	1314
* Air - oxygen	HXER # 12	17.90	33.76	530
- <u>lower half</u> :				
* Air - nitrogen	HXER # 20	3.41	37.11	92
* Air - oxygen	HXER # 21	1.40	41.59	34
<u>Preliminary Heat-Exchanger/</u>				
* Air - nitrogen	HXER # 9	2.35	18.32	128
* nitrogen - oxygen	HXER # 15	- 2.52†	1.45	1737
* Air - nitrogen	HXER # 10	2.72	7.37	370
$\Sigma =$				4205

† the negative sign is imposed because at this section of heat exchanger, the nitrogen works as a hot stream which is opposite to the nitrogen function throughout the rest of sections.

TABLE 5.2 : "UA" FOR PREDICTED FLOWSHEET

<u>Section</u>	<u>Simulating Module</u>	<u>Q (kW)</u>	<u>LMTD (°C)</u>	<u>"UA" (W/°C)</u>
<u>Heat Exchanger/</u>				
- <u>upper half</u> :				
* Air - nitrogen	HXER # 2	40.00	23.20	1724
* Air - oxygen	HXER # 1	9.87	36.01	274
- <u>lower half</u> :				
* Air - nitrogen	HXER # 3	5.10	32.58	157
<u>Preliminary Heat-Exchanger/</u>				
* Air - nitrogen	HXER # 11	2.38	15.51	153
* Air - oxygen	HXER # 10	1.96	19.04	103
<u>Σ</u>				2411

TABLE 5.3 : "U" FOR ACTUAL DESIGN

<u>Section</u>	<u>"UA"</u> (W/ $^{\circ}$ C)	<u>A</u> (m 2)	<u>U</u> (W/m 2 . $^{\circ}$ C)
<u>Heat Exchanger/</u>			
- <u>upper half</u> :			
* Air - nitrogen	1314	4.58	286.9
* Air - oxygen	530	2.29	231.4
- <u>lower half</u> :			
* Air - nitrogen	92	4.58	20.1
* Air - oxygen	34	2.29	14.8
<u>Preliminary Heat-Exchanger/</u>			
* Air - nitrogen	128 + 370 = 498	7.1	70.1
* nitrogen - oxygen	1737	1.4	1240.7

TABLE 5.4 : HEAT TRANSFER AREA FOR THE PREDICTED DESIGN

Section	U (W/m ² . °C)	"UA" (W/°C)	A (m ²)
<u>Heat Exchanger/</u>			
- <u>upper_half</u> :			
* Air - nitrogen	286.9	1724	6.0
* Air - oxygen	231.4	274	1.18
- <u>lower_half</u> :			
* Air - nitrogen	20.1	157	7.8
* Air - oxygen	14.8	0.0	0.0
<u>Preliminary Heat-Exchanger/</u>			
* Air - nitrogen	70.1	153	2.18
* Air - oxygen	x†	103	x
$\Sigma =$		17.16 + x	

† x stands for the unknown value of the heat-transfer coefficient between air and oxygen since in the actual design of the plant there is no match between these two streams.

For air-oxygen match suggested within the preliminary heat-exchanger, a coefficient of heat transfer similar to that of air-oxygen match within the upper half of the actual heat exchanger is employed ($U = 231.4 \text{ W/m}^2 \cdot ^\circ\text{C}$). Also, the geometry of tubes is suggested to remain invariant i.e. tube-in-tube. It is now desired to evaluate the dimensions of these tubes which will suit both the coefficient of heat transfer and the suggested geometry.

When two different fluids, say F and G, flow according to the pattern shown in Figure 5.1, the coefficient of heat transfer U ruling the process of heat transfer between these fluids is estimated as follows⁽³¹⁾, assuming that thermal resistance of the tube wall is negligible when compared to the thermal resistance of the fluid film,

$$U = \frac{h_{io} * h_o}{h_{io} + h_o} \quad \dots \dots \quad (5.5)$$

where,

h_o : film coefficient of heat transfer for fluid F and

h_{io} : film coefficient of heat transfer for fluid G.

Actually h_{io} is the modified value of the coefficient (h_i) which is based on the inside diameter of the inner tube such that,

$$h_{io} = h_i * \frac{d_i}{d_o} \quad \dots \dots \quad (5.6)$$

where, referring to Figure 5.1,

h_{io} : Modified film coefficient of heat transfer, $\text{W/m}^2 \cdot ^\circ\text{C}$,

h_i : Film coefficient of heat transfer based on the inside diameter of the inner tube, $\text{W/m}^2 \cdot ^\circ\text{C}$,

d_i : Inside diameter of the inner tube, m and

d_o : Outside diameter of the inner tube, m.

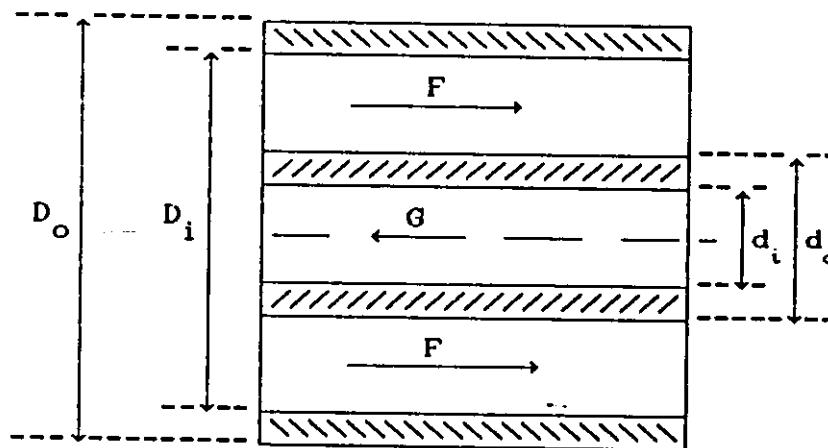


Figure 5.1 Tube-in-tube section

For turbulent flow of gases inside tubes, the film coefficient of heat transfer h can be expressed as a function of both Prandtl and Reynolds numbers as follows⁽³²⁾

$$h = 0.022 * \text{Pr}^{0.6} * \text{Re}^{0.8} \quad (5.7)$$

To obtain the required value of U ($231.4 \text{ W/m}^2 \cdot ^\circ\text{C}$), equation (5.5) is used. To determine the appropriate heat transfer coefficients h_o and h_{io} resulting from the suggested geometry (tube-in-tube), one can use equation (5.7)

whereby the Prandtl number is known at the corresponding range of temperature. The Reynolds number in the predicted section can be now found. As the Reynolds number is known, one can choose a combination of flow conditions and geometry to satisfy the calculated Reynolds number.

Let's denote the parameters related to the upper half of heat exchanger A4 by "1" and those related to the preliminary heat exchanger A3 by "2".

Then, recalling the necessary values from Table 5.5 and substituting into equation 5.6 will give

$$\Pr_1^{0.6} * \text{Re}_1^{0.8} = \Pr_2^{0.6} * \text{Re}_2^{0.8}$$

For air,

$$(1.141)^{0.6} * (6.60866E4)^{0.8} = (0.759)^{0.6} * (\text{Re}_2)^{0.8}$$

$$\therefore \text{Re}_{z, \text{air}} = 8.97 \text{ E4}$$

TABLE 5.5 : Pr & Re NUMBERS FOR AIR AND OXYGENHeat Exchanger/ Upper Half

<u>Stream</u>	<u>Prandtl Number</u>	<u>Reynolds Number</u>
Air	1.141	6.60866 E4
O ₂	1.010	1.94540 E4

Preliminary Heat-Exchanger

<u>Stream</u>	<u>Prandtl Number</u>
Air	0.759
O ₂	1.107

For O₂,

$$(1.010)^{0.6} * (1.9454E4)^{0.8} = (1.107)^{0.6} * (Re_{2,O_2})^{0.8}$$

$$\therefore Re_{2,O_2} = 1.811E4$$

For the flow inside tube, the Reynolds number is expressed as follows

$$Re = \frac{\rho v d}{\mu} \quad \dots\dots\dots(5.8)$$

where,

Re : Reynolds number,

v : Flow velocity, m/s,

d : Tube diameter, m and

μ : Dynamic viscosity of the flowing fluid Pa.s.

But for the flow inside annular space (tube-in-tube), an equivalent diameter, De , is substituted instead of tube diameter, d . For heat transfer, De may be given as in Reference 32,

$$De = \frac{D_i^2 - d_o^2}{d_o^2} \quad \dots\dots\dots(5.9)$$

in terms of the dimensions shown in Figure 5.1, where

D_i : The inside diameter of the outer tube, m.

Now,

$$Re_{z,air} = \frac{\rho v De}{\mu} = \frac{(53.1)(v)(De)}{1.811E-5} = 8.97 E4$$

$$\therefore (v.De)_{air} = 0.030 \quad \text{and,}$$

$$Re_{z,o} = \frac{(304.35)(v)(d_i)}{1.98E-5} = 1.811E4$$

$$\therefore (vd_i)_o = 1.1782E-3$$

Let's have for oxygen, 19 tubes of 5x1 mm diameter (similar to the upper-half section within the heat exchanger A4). This will yield a flow velocity of

$$v = \frac{1.1782E-3}{d_i (= 3E-3)} = 0.39 \text{ m/s.}$$

This value is less than that of the actual design (0.5 m/s).

It is noted that oxygen tubes are selected to have the same thickness as in the heat exchanger since the working pressure is assumed similar to the actual design.

To maintain air velocity as in the actual design (5.0 m/s),

$$De = \frac{0.030}{5.0} = 6.0 E-3 \text{ m} = 6.0 \text{ mm.}$$

Recall that $De = \frac{D_i^2 - d_o^2}{d_o^2}$ where d_o , here, is the outside

diameter of the inner (oxygen) tube i.e. 5 mm.

$$\therefore 5.64 = \frac{D_i^2 - 5^2}{5^2} \text{ then,}$$

$$D_i = 13.22 \cong 13 \text{ mm}$$

In order to calculate the thickness of the outer (air) tube, the equation of hoop stresses has been used ⁽³³⁾. The necessary properties of copper were obtained ⁽³⁴⁾. The thickness was calculated to be 1.0mm. This means that the

diameter of the air tube is 15x1 mm.

Since U_{air-O_2} has already been assumed to be 231.4 W/ m² °C and the product of "UA" was determined as 103 W/ °C, then the heat-transfer area required for air-O₂ section will be

$$A_{air-O_2} = \frac{103}{231.4} = 0.445 \text{ m}^2.$$

Hence, the length of the air tube should be

$$L = \frac{0.445}{(19)(\pi)(5.E-3)} = 1.491 \cong 1.5 \text{ m}.$$

Finally a bundle of straight, copper tube-in-tube arrangement would be necessary. This will result in a total area of heat transfer pointing to $17.16 + 0.445 = 17.605 \text{ m}^2$. This resembles a reduction in heat transfer area of about 21% of the existing design (22.24 m².)

5.1.3 Inspection of the air-separation unit/ rectifying columns

Using the program phase named "Tray Sizing Input", it has been possible to investigate the maximum rectification capacity of the two columns within the air-separation unit.

The sizing of a column is based on the flooding diameter. It is the maximum diameter beyond which flooding occurs i.e. the column will function improperly.

The lower and upper column diameters needed to rectify the nominal flow rate of the incoming air have been calculated to be 0.3 m and 0.4 m, respectively. These diameters are less than those within the existing design. The flow rate of air has then been incrementally raised until both lower and upper columns approach their actual diameters (0.4 m and 0.5 m for the lower and upper columns, respectively). The corresponding value of air flow-rate amounts to 70% above the nominal flow rate.

Through plant details, the maximum tolerance provided for design capacities of equipment was noticed to be less than or equal to 10%. If a similar tolerance is assumed for the air-separation unit, it will have a capacity allowance of 60%. This availability indicates the possibility of increasing the plant productivity. Of course, the plant equipment will respond to the new productivity according to the rated capacity of each. Tracking the plant circuit, the following steps are needed for increasing the productivity of the plant by 60% :

1. Installing another air compressor which is able to deliver 60% of the air flow-rate suggested for the existing compressor at the same outlet pressure. The

resulting two outputs could then join each other in a final single line.

2. Replacement of the air purifier with a new one whose cylinders can accommodate the purification of the extra amount of air flow-rate.
3. Replacement of the expansion turbine with a new one whose liquefaction (refrigeration) duty is high enough to fit the running target.
4. Replacement of the liquid-oxygen pump with a new one capable to pump 60% over the rated capacity of the present pump.
5. Replacement of the existing pipe-network with a new one which is capable to withstand 1.6 times the nominal flow rates.
6. Instruments will be independent of any variation. This is due to the constancy of the properties being measured.
7. A reserve of 21% in the heat transfer area has been estimated to exist within the heat-exchanger network if operated according to the configuration predicted before.

Along with these steps, neither new nor additional civil

work will be needed.

5.2. Economical Comparison of the Existing and the Predicted Designs

Table 5.6 illustrates the elements of capital cost of a new oxygen plant based on 1988 prices⁽¹⁾. The Table also shows the extra capacity for the major equipment within the plant over that given by the manufacturer. This capacity has been evaluated such that the production capacity of the existing plant could be expanded by 60%.

Table 5.7 illustrates the cost of expanding the production capacity of the existing plant by 60% of its nominal value and regarding to the steps listed in section 5.1.4.

TABLE 5.6 : COST ELEMENTS OF A NEW OXYGEN PLANT OF TYPE
K-0.15 AND THE EXTRA CAPACITY WITH RESPECT
RESPECT TO 60% EXPANSION

Cost element	Cost	Extra Capacity
* Air compressor	\$ 100 ,000	0 .0%
* Air purifier	\$ 10 ,000	0 .0%
* preliminary heat exchanger,A3	\$ 15 ,000	35 .0%
* Heat exchanger ,A4	\$ 20 ,000	0 .0%
* Turbo-expander	\$ 10 ,000	0 .0%
* Liquid-oxygen pump	\$ 15 ,000	0 .0%
* Rectifying columns	\$ 50 ,000	100 .0%
* Two subcoolers	\$ 15 ,000	0 .0%
* Electric system	\$ 9 ,000	100 .0%
* Piping system	\$ 16 ,000	0 .0%
* Civil construction	\$ 75 ,000	100 .0%
Σ Cost	\$ 335 ,000	

TABLE 5.7 : COST OF EXPANDING THE
PRODUCTION CAPACITY OF
THE EXISTING PLANT BY
60%

Cost element	Cost
* Air compressor	\$ 85,000
* Air purifier	\$ 12,000
* preliminary heat exchanger,A3	\$ 16,000
* Heat exchanger ,A4	\$ 23,000
* Turbo-expander	\$ 12,000
* Liquid-oxygen pump	\$ 18,000
* Rectifying columns	\$ 0,000
* Two subcoolers	\$ 18,000
* Electric system	\$ 0,000
* Piping system	\$ 25,000
* Civil construction	\$ 0,000
Σ Cost	\$ 209,000

Table 5.8 shows the annual costs for both the existing and expanded plants⁽¹⁾.

Now, if JISICO has a future plan demanding more oxygen gas but, less than or equal 1.6 times the production capacity of the existing plant, then two alternatives will rise.

TABLE 5.8 : ANNUAL COST FOR THE EXISTING AND EXPANDED PLANTS

Cost item	Cost value	
	Existing plant	Expanded plant
Electric power	\$ 10,000	\$ 16,000
Man-power	\$ 12,000	\$ 12,000
Maintenance	\$ 8,000	\$ 12,800
Σ	\$ 30,000	\$ 40,000

These are,

- Alternative 1 : To expand the production capacity of the existing plant by 60%
- Alternative 2 : To install a new plant similar to the existing one.

Upon having a decision to choose either of the two alternatives, lets assume that the company will adopt a simple aggressive policy, and choose a plan for the optimistic sales forecast. This level of analysis is of the least order of complexity⁽³⁵⁾.

The analysis can be summarized as follows :

- (1) The annual costs are estimated for both alternatives 1 & 2.
- (2) The cash necessary to start each alternative is annualized. The calculations are based on 10% interest rate and 20 years life time.
- (3) The summation of values estimated in (1) and (2) is divided by the annual amount of production to estimate cost of unit production as expected along the plant life time.

Table 5.9 illustrates this analysis.

TABLE 5.9 : ALTERNATIVE - DECISION ANALYSIS

Description	Alternative (1)	Alternative (2)
(a) Annual cost :		
- Electric power	\$ 16,000	\$ 20,000
- Man-power	\$ 12,000	\$ 24,000
- Maintenance	\$ 12,800	\$ 16,000
Σ	\$ 40,000	\$ 50,000
(b) Cash for starting	\$ 209,000	\$ 335,000
(c) Annualized cash	\$ 24,549	\$ 39,349
(d) Σ (a) & (c)	\$ 64,549	\$ 89,349
(e) Annual production of oxygen, m^3	(1.6)(1.23E6)*	(2.0)(1.23E6)
(f) Cost of unit production	\$ 0.0327	\$ 0.0363

Alternative 1 is noticed to cost about 94.5% of the alternative 2 cost.

* The annual production of the plant is 1,230,000 m^3 of gaseous oxygen⁽¹⁾.

The estimated cost of unit production does not consider the deterioration factor for either equipment or currency. It only indicates the alternatives costs relative to each other.

Chapter Six

CONCLUSION

The following statements constitute the conclusion of the research :

- (1) The oxygen plant under consideration satisfies the condition of the minimum energy requirements by 91.8%. Recall that the rated capacity of the turboexpander is 6.4 kW while the minimum cold requirement has been estimated to be 5.8788 kW.
- (2) The heat exchanger network employed within the plant slightly deviates from the optimum configuration. The heat transfer area of the preliminary heat exchanger A3, could be reduced by 21% if a stream configuration similar to that within the heat exchanger A4 was applied.
- (3) The air-separation unit/rectifying columns are capable to handle 60% more flow rate of air than the nominal flow rate. This extra capacity does not consider the (evaporation) losses within practice, it stands for ideal process.
- (4) The existing instrumentation of the plant will need few

modifications to accommodate the additional rates.

- (5) The cost of equipment replacement has been estimated to be \$ 209,000 based on the 1988 prices. While erecting a new similar plant will cost \$ 335,000.
- (6) The cost analysis presented shows that the cost of unit production if the existing plant is expanded is about 90.0% of the cost if a new similar plant is installed next to the existing one. If the alternative of plant expansion was decided upon, the operation of the existing plant would be stalled to implement the necessary equipment replacement. This may impose additional cost such that expansion becomes not profitable.
- (7) The results obtained in this work indicate that expanding the existing plant may not afterall be favourable.

REFERENCES

1. Catalogues and drawings of air-separation plant type K-0.15, Chemmashexport, Odessa, USSR, 1988.
2. N. N. Greenwood and E. Earshaw, *Chemistry of Elements*, Pergamon Press, Ltd., 1984.
3. J. W. Mellor, *A Comprehensive Treatise on Inorganic and Theoretical Chemistry*, Vol.1, pp. 344-51. Longmans, Green, 1922. History of the discovery of oxygen.
4. M. E. Weeks, *Discovery of the Elements*, 6th edn., pp. 209-23, Journal of Chemical Education, Easton. Pa. 1956. (Oxygen).
5. J. R. Partington, *A History of Chemistry*, Vol.3, Macmillan, London, 1962; Scheele and discovery of oxygen (pp. 219-22).
6. A. L. Lavoisier, *La Traite' Elementaire de Chemie*, Paris 1789, Translated by R. Kerr, *Elements of Chemistry*, London, 1790; facsimile reprint by Dover Publication, Inc., New York, 1965.
7. Norman Booth (ed), *Industrial Gases*, Pergamon Press Ltd., Oxford, 1973.
8. Pamphlet G-4, Compressed Gas Association, Inc., 6th edn., 1972, New York, N. Y. 10036 (pp. 4).
9. A. Arkharov, I. Marfenina, Y. Mikulin, *Theory and Design of Cryogenic Systems*, Mir Publishers, 1981.

10. G. G. Haselden (ed), *Cryogenic Fundamentals*, Academic Press Inc., London, 1971.
11. B. A. Hands (ed), *Cryogenic Engineering*, Academic Press Inc., London, 1986.
12. J. E. Hendry, D. F. Rudd and J. D. Seader, *Synthesis in the Design of Chemical Processes*, A.I.Ch.E.Jl., Vol.19, pp. 1-15, 1973.
13. S. Raghavan, *Heat Exchanger Network Synthesis : A thermodynamic approach*. Ph.D. thesis, Purdue University, West Lafayette, IN (1977).
14. J. C. Chato, R. J. Laverman and J. M. Shah, *Analysis of Parallel Flow, Multi-Stream Heat Exchangers*, Int. J. Heat Mass Transfer, Vol. 14, pp. 1691-1703, 1971.
15. K. F. Lee, A. H. Masso and D. F. Rudd, *Branch and Bound Synthesis of Integrated Process Designs*, Ind. Eng. Chem. Fundam., Vol. 9, pp. 48-58, 1970.
16. N. Nishida, Y. A. Liu and L. Lapidus, *Studies in Chemical Process Design and Synthesis. III : A simple and practical approach to the optimal synthesis of heat exchanger networks*, A. I. Ch. E. Jl., Vol. 23, pp. 77-93, 1977.
17. B. Linnhoff and J. R. Flower, *Synthesis of Heat Exchanger Networks*, A. I. Ch. E. Jl., Vol. 24, pp. 633-654, 1978.
18. A. Malhotra, O. P. Chawla and S. H. Mullick, *Economic Optimization of Heating Chains by Discrete Maximum Principle*, Reg. J. Energy Heat Mass Transfer, Vol. 1, pp. 142-152, 1979.

19. A. R. Siddiqui, A. Malhotra and O. P. Chawla, *Economic Optimization of Heat Exchanger-Cooler Train*, *Reg. J. Energy Heat Mass Transfer*, Vol. 3, pp. 39-47, 1981.
20. A. R. Siddiqui, A. Malhotra and O. P. Chawla, *Optimization of a Heat Exchanger Chain Consisting of Two Cold Streams*, *Eng. Optim.*, Vol. 7, pp. 157-166, 1984.
21. A. R. Parkinson, J. S. Liebman, C. O. Pedersen and A. B. Templeman, *The Optimal Design of Resilient Heat Exchanger Networks*, *A. I. Ch. E. Symp. Ser. 78*, No. 214, pp. 85-98, 1982.
22. A. R. Parkinson, J. S. Liebman and C. O. Pedersen, *Principles of Heat Exchangers Network Design*, manuscript submitted for publication.
23. K. Hesselmann, *Optimization of Heat Exchanger Networks, Second Law Aspects of Thermal Design*, Vol. HTD-33, pp. 95-99. ASME, New York, 1984.
24. B. Linnhoff and J. A. Turner, *Heat-Recovery Networks : New insights yield big savings*, *Chem. Engng.*, Vol. 88(22), pp. 56-70, 1981.
25. E. Hindmarsh, D. Boland and D. W. Townsend, *Maximizing Energy Savings for Heat Engines in Process Plants*, *Chem. Engng.* Vol. 92(3), pp. 38-47, 1985.
26. John C. Chato and C. Damianides, *Second-Law-Based Optimization of Heat Exchanger Network Using Load Curves*, *Int. Jl. Heat Mass Transfer*, Vol. 23, No. 8, pp. 1079-1086, 1986. K-0.15, Chemmashexport, Odessa, USSR, 1988.
27. E. C. Hohmann, *Optimum Networks for Heat Exchange*, Ph.D. Thesis, University of Southern California, 1971.

28. T. Umeda, J. Itoh, and K. Shiroko, *Chem. Eng. Prog.*, 74 (9):70, 1978.
29. B. Linnhoff. ...et al.). *A User Guide on Process Integration for the Efficient Use of Energy*, London: The Institution of Chemical Engineers, 1982.
30. R. B. Smith, T. Dresser, and S. Ohlswager, *Hydrocarbon Processing*, 40, No. 5, pp 183, 1963
31. Donald Q. Kern, *Process Heat Transfer*, McGraw-Hill Inc., 24th edn., 1988.
32. W. M. Kays, *Convective Heat and Mass Transfer*, McGraw-Hill Inc., 1966.
33. Ferdinand P. Beer and E. Russell Johnston, *Mechanics of Materials*, McGraw-Hill Ryerson Limited, 1985.
34. R. E. Gackenbach, *Materials Selection for Process plants*, Reinhold Publishing Corporation, 1960.
35. L. M. Rose, *Engineering Investment Decisions : Planning under uncertainty*, Elsevier Scientific Publishing Company, 1st edn., 1976.

APPENDIXES

Appendix A

Appendix B

Appendix C

Appendix D

Appendix E

Appendix A

Temperature interval analysis

$$\Delta T = 0.0^{\circ}\text{C}$$

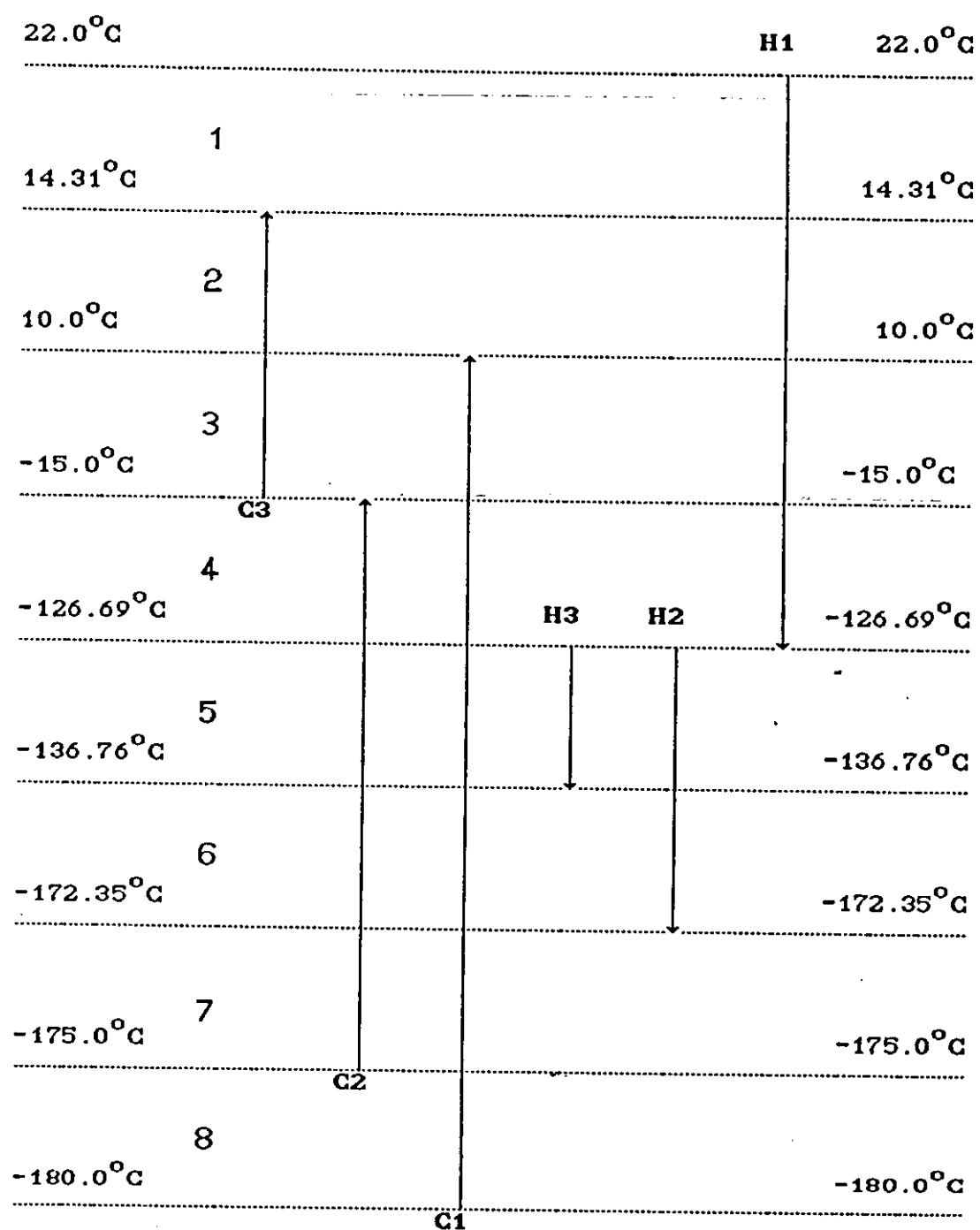


Figure A.1 : Temperature interval analysis

TABLE A.1 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 0.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma CP_{\text{cold}} - \Sigma CP_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
22.0	1	7.69	- 0.4151	-3.1921	Surplus
14.31	2	4.31	- 0.3281	-1.4141	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	111.69	- 0.0315	-3.5182	Surplus
-126.69	5	10.07	- 0.2412	-2.4289	Surplus
-136.76	6	35.59	+ 0.2364	+8.4135	Deficit
-172.35	7	2.65	+ 0.3836	+1.0165	Deficit
-175.0	8	5.0	+ 0.1149	+0.5745	Deficit
-180.0					
			$E_{\min} = \Sigma$	5.8788	

$$\Delta T \approx 1.0^\circ\text{C}$$

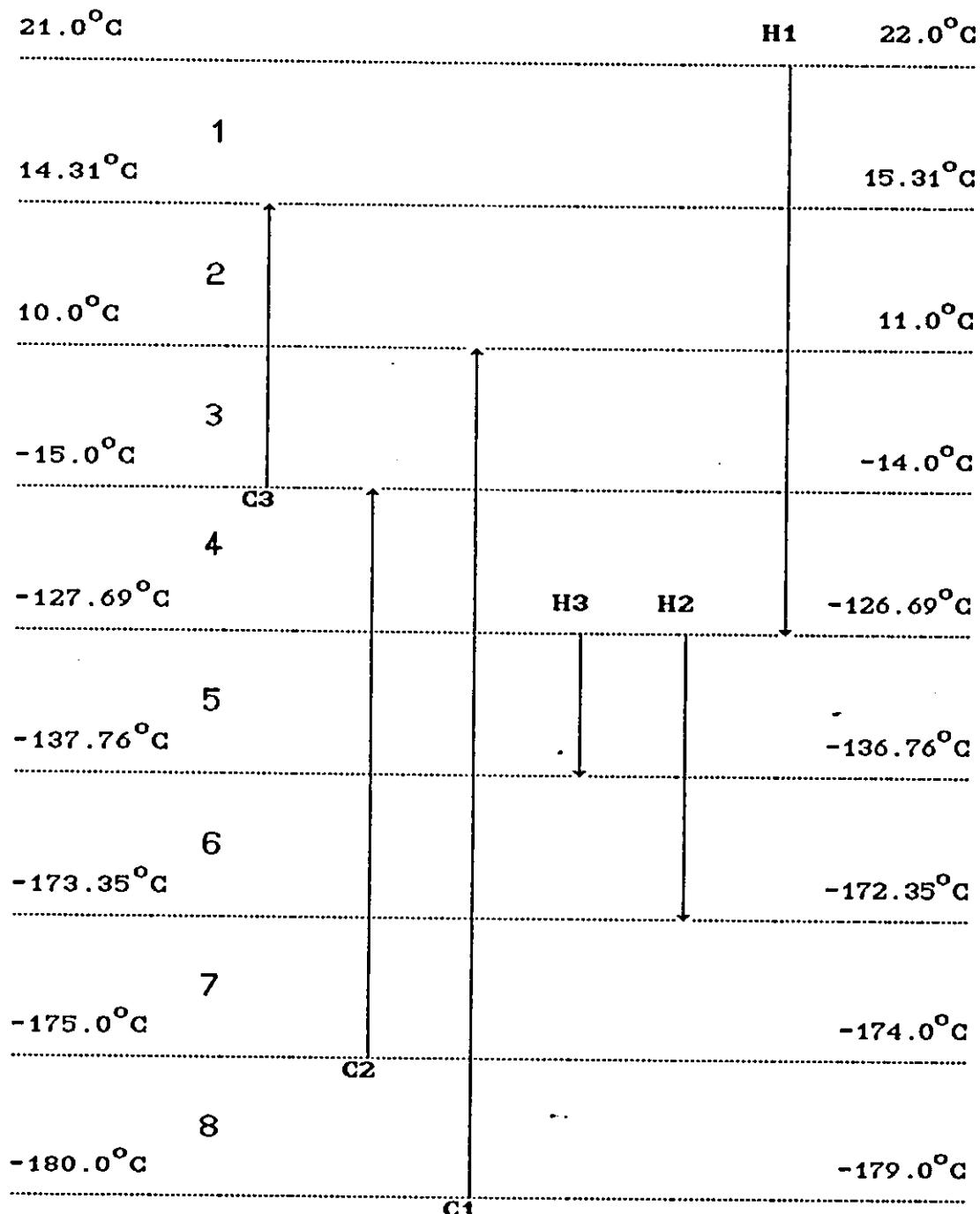


Figure A.2 : Temperature interval analysis

TABLE A.2 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 1.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma CP_{\text{cold}} - \Sigma CP_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
21.0	1	6.69	- 0.4151	-2.7770	Surplus
14.31	2	4.31	- 0.3281	-1.4141	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	111.69	- 0.0315	-3.5497	Surplus
-127.69	5	10.07	- 0.2412	-2.4289	Surplus
-137.76	6	35.59	+ 0.2364	+8.4135	Deficit
-173.35	7	2.65	+ 0.3836	+0.6329	Deficit
-175.0	8	5.0	+ 0.1149	+0.5745	Deficit
-180.0					
			$E_{\min} = \Sigma$	5.8788	

$$\Delta T = 2.0^{\circ}\text{C}$$

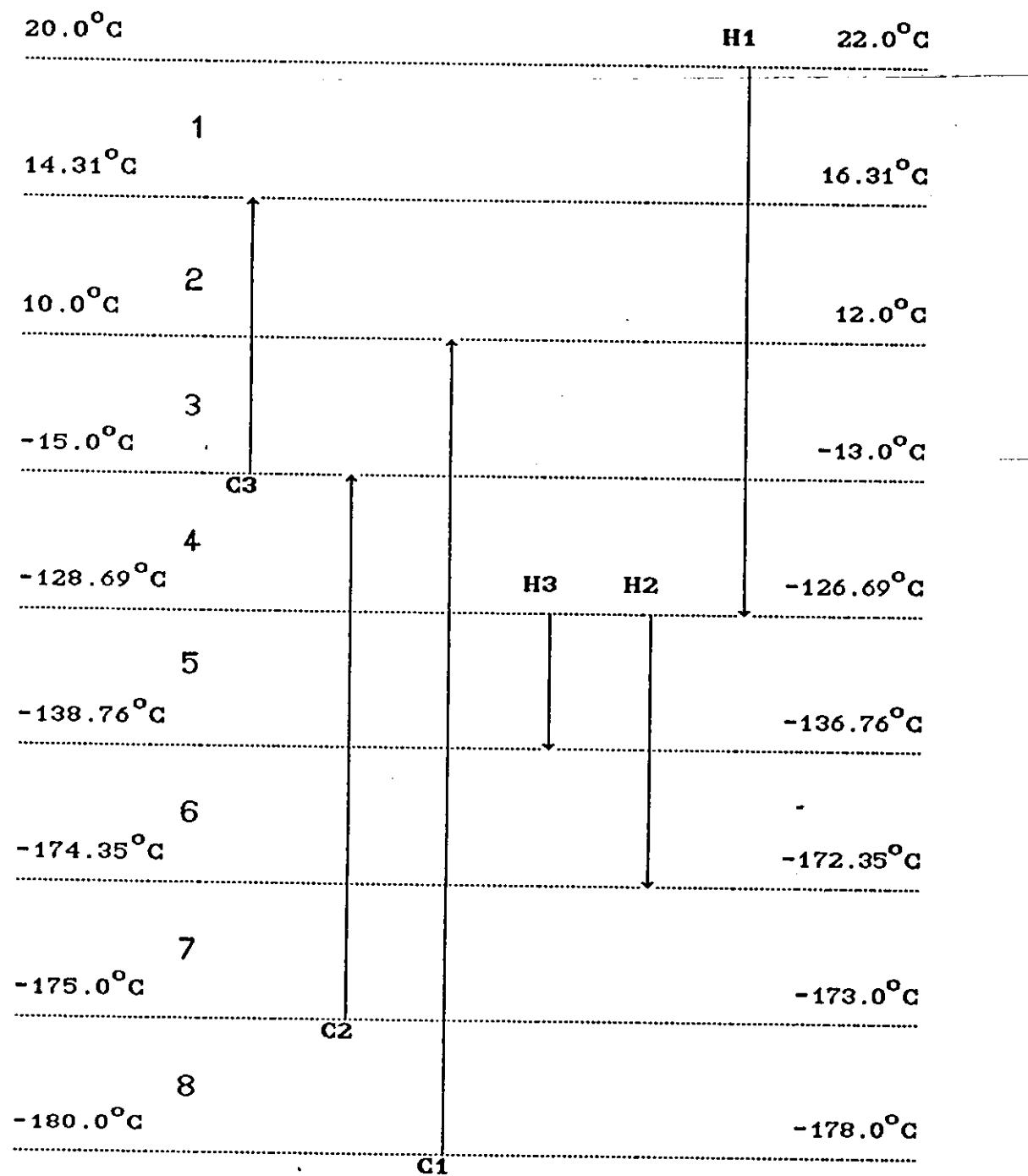


Figure A.3 : Temperature interval analysis

TABLE A.3 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 2.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma CP_{\text{cold}} - \Sigma CP_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
20.0					
14.31	1	5.69	- 0.4151	-2.3619	Surplus
10.0	2	4.31	- 0.3281	-1.4141	Surplus
-15.0	3	25.0	- 0.2132	-5.33	Surplus
-128.69	4	113.69	- 0.0315	-3.5182	Surplus
-138.76	5	10.07	- 0.2412	-2.4289	Surplus
-174.35	6	35.59	+ 0.2364	+8.4135	Deficit
-175.0	7	0.65	+ 0.3836	+0.2647	Deficit
-180.0	8	5.0	+ 0.1149	+0.5745	Deficit
$E_{\min} = \Sigma$					5.8634

$$\Delta T = 3.0^{\circ}\text{C}$$

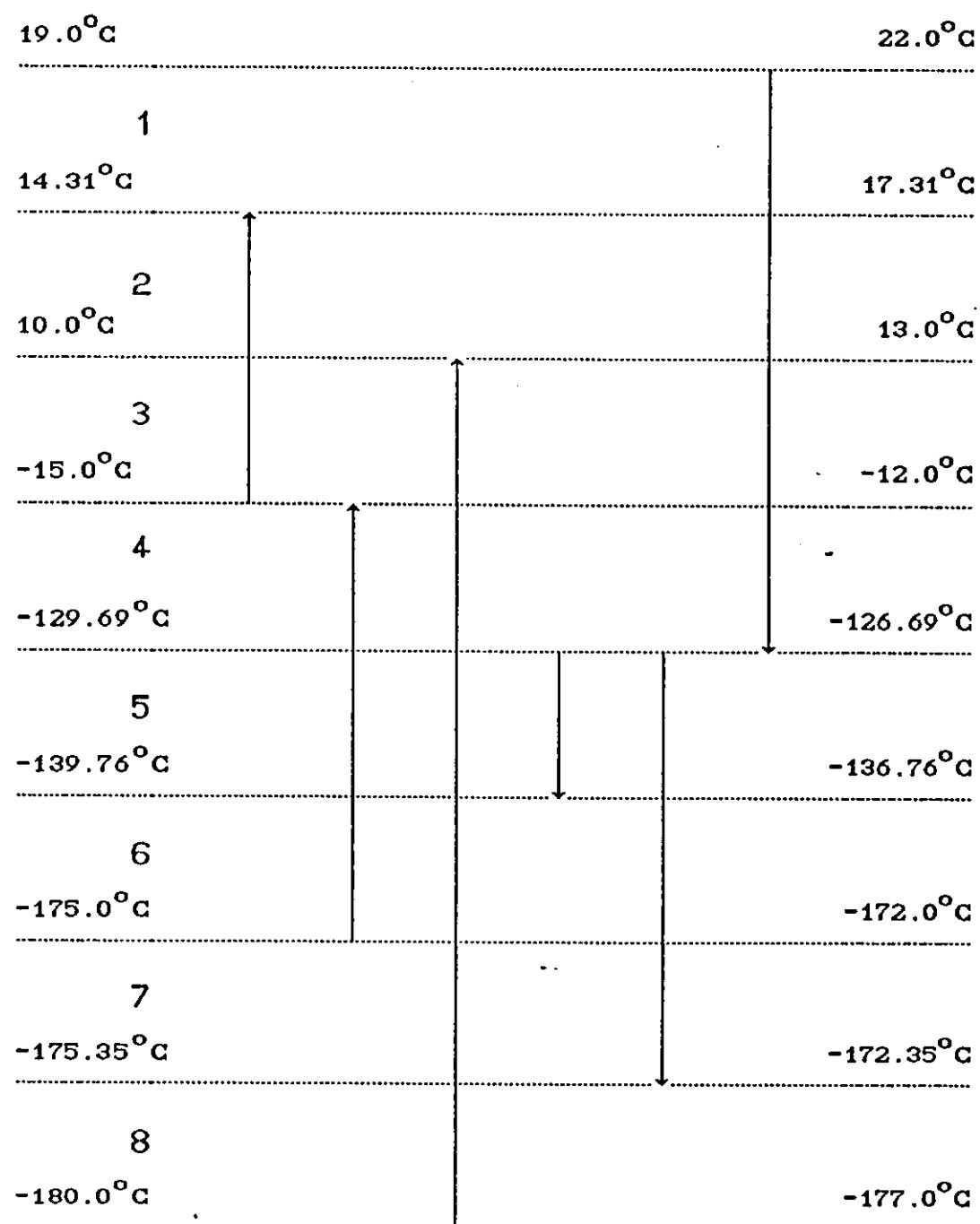


Figure A.4 : Temperature interval analysis

TABLE A.4 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 3.0^{\circ}\text{C}$

T_i $(^{\circ}\text{C})$	T_{i+1} $(^{\circ}\text{C})$	ΣCP_{cold} - ΣCP_{hot} ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
19.0	1	4.69	- 0.4151	-1.9468 Surplus
14.31	2	4.31	- 0.3281	-1.4141 Surplus
10.0	3	25.0	- 0.2132	-5.33 Surplus
-15.0	4	114.69	- 0.0315	-3.6127 Surplus
-129.69	5	10.07	- 0.2412	-2.4289 Surplus
-139.76	6	35.24	+ 0.2364	+8.3307 Deficit
-175.0	7	0.35	- 0.0323	-0.0113 Surplus
-175.35	8	4.65	+ 0.1149	+0.5343 Deficit
-180.0				
			$E_{\min} = \Sigma$	5.8788

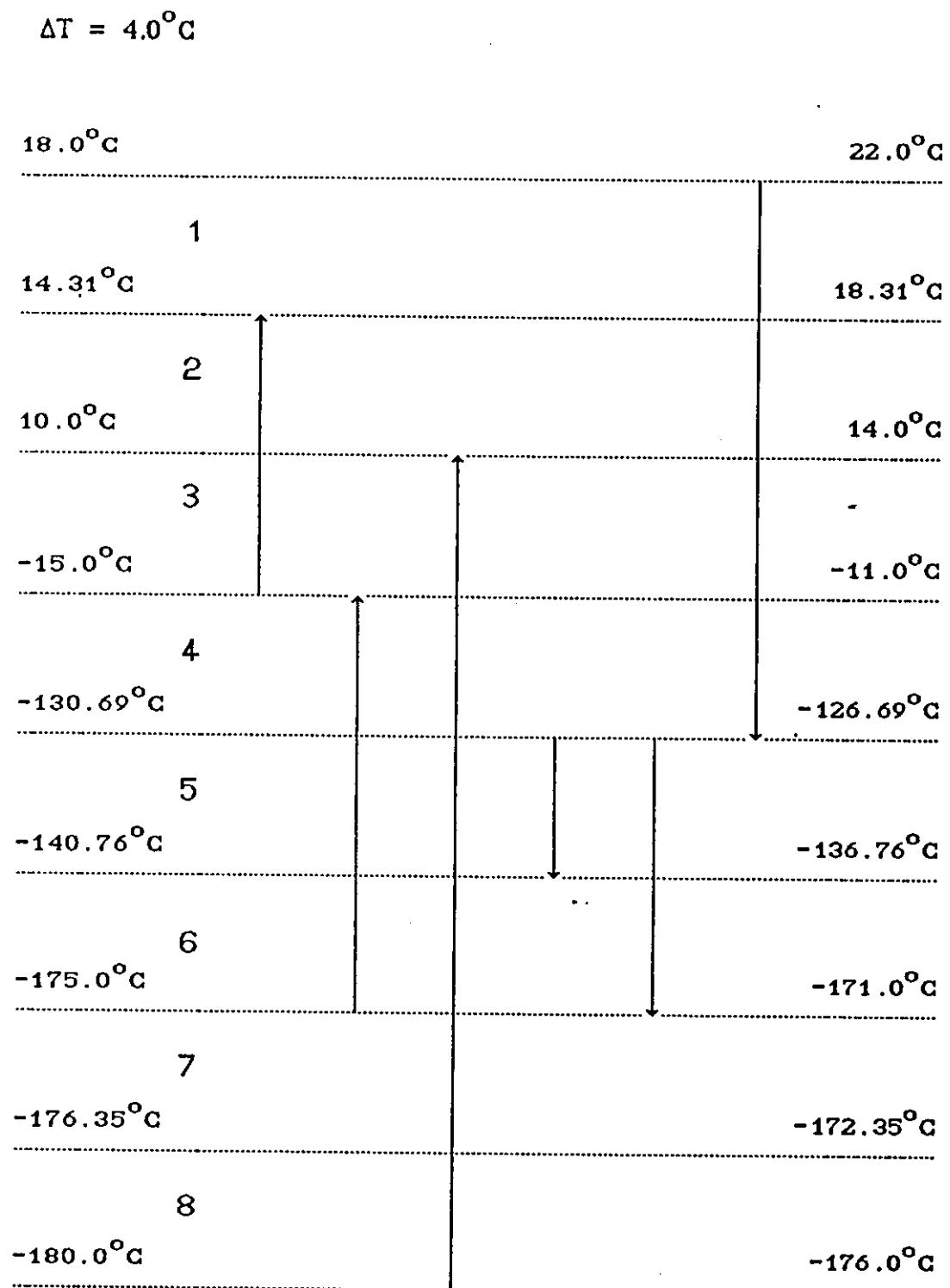


Figure A.5 : Temperature interval analysis

TABLE A.5 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 4.0^{\circ}\text{C}$

$T_i - T_{i+1}$ $(^{\circ}\text{C})$	$\Sigma CP_{\text{cold}} -$ ΣCP_{hot} $(\text{kW}/^{\circ}\text{C})$	ΔE_i (kW)	Surplus or Deficit
18.0			
14.31	3.69	- 0.4151	-1.5317 Surplus
10.0	4.31	- 0.3281	-1.4141 Surplus
-15.0	25.0	- 0.2132	-5.33 Surplus
-130.69	115.69	- 0.0315	-3.6442 Surplus
-140.76	10.07	- 0.2412	-2.4289 Surplus
-175.0	34.24	+ 0.2364	+8.0943 Deficit
-176.35	1.35	- 0.0323	-0.0436 Surplus
-180.0	3.65	+ 0.1149	+0.4194 Deficit
$E_{\min} = \Sigma$			5.8788

$$\Delta T = 5.0^{\circ}\text{C}$$

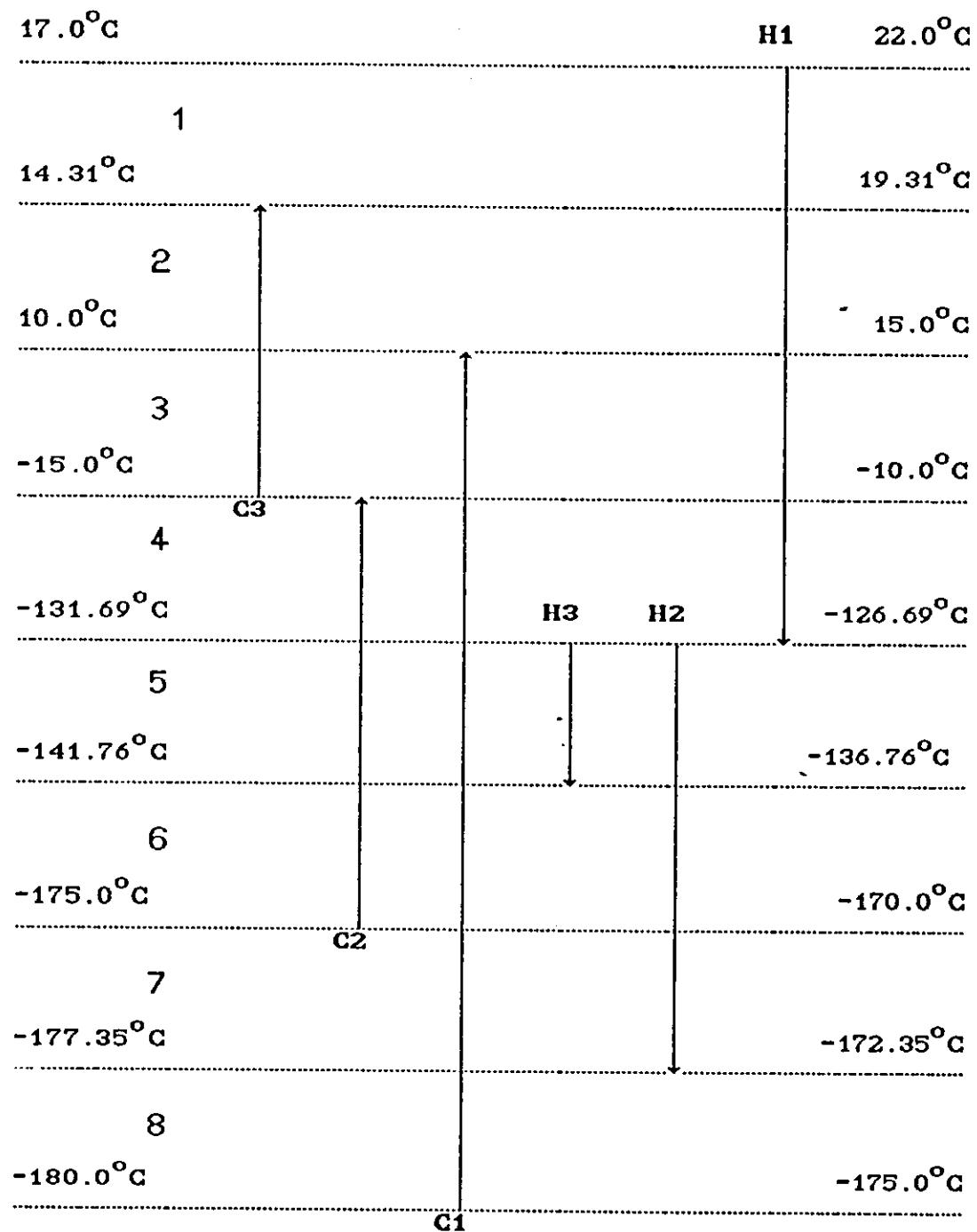


Figure A.6 : Temperature interval analysis

TABLE A.6 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 5.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma \text{CP}_{\text{cold}} - \Sigma \text{CP}_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
17.0					
14.31	1	2.69	- 0.4151	-1.1166	Surplus
10.0	2	4.31	- 0.3281	-1.4141	Surplus
-15.0	3	25.0	- 0.2132	-5.33	Surplus
-131.69	4	116.69	- 0.0315	-3.6757	Surplus
-141.76	5	10.07	- 0.2412	-2.4289	Surplus
-175.0	6	33.24	+ 0.2364	+7.8579	Deficit
-177.35	7	2.35	- 0.0323	-0.0759	Surplus
-180.0	8	2.65	+ 0.1149	+0.3045	Deficit
$E_{\min} = \Sigma$				5.8788	

$$\Delta T = 6.0^{\circ}\text{C}$$

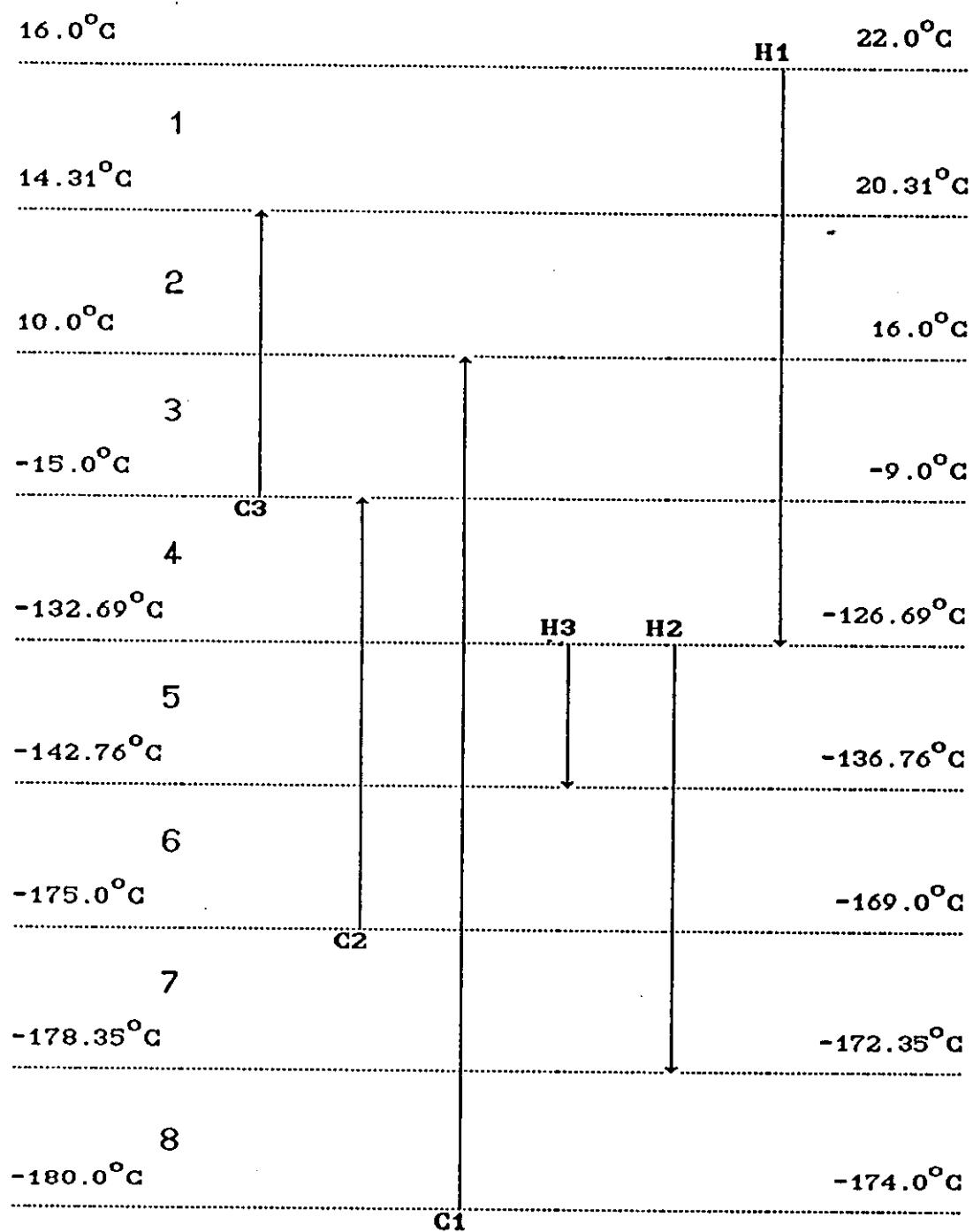


Figure A.7 : Temperature interval analysis

TABLE A.7 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 6.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma \frac{CP_{\text{cold}}}{CP_{\text{hot}}} =$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
16.0	1	1.69	- 0.4151	-0.7015	Surplus
14.31	2	4.31	- 0.3281	-1.4141	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	117.69	- 0.0315	-3.7072	Surplus
-132.69	5	10.07	- 0.2412	-2.4289	Surplus
-142.76	6	32.24	+ 0.2364	+7.6215	Deficit
-175.0	7	3.35	- 0.0323	-0.1082	Surplus
-178.35	8	1.65	+ 0.1149	+0.1896	Deficit
-180.0					
			$E_{\min} = \Sigma$	5.8788	

$$\Delta T = 7.0^\circ\text{C}$$

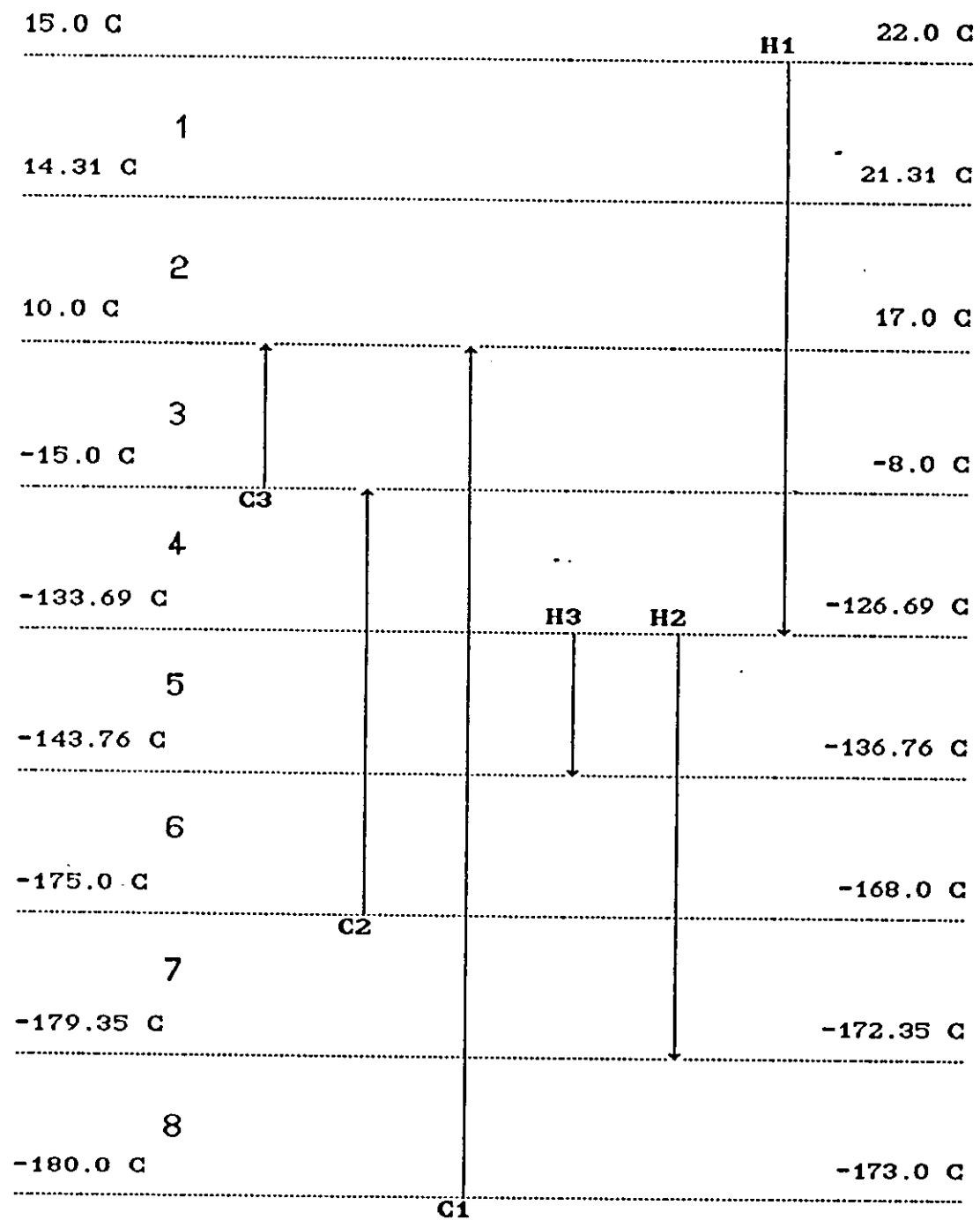


Figure A.8 : Temperature interval analysis

TABLE A.8 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 7.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma \text{CF}_{\text{cold}} - \Sigma \text{CP}_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
15.0	1	0.69	- 0.4151	-0.2864	Surplus
14.31	2	4.31	- 0.3281	-1.4141	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	118.69	- 0.0315	-3.7387	Surplus
-133.69	5	10.07	- 0.2412	-2.4289	Surplus
-143.76	6	31.24	+ 0.2364	+7.3851	Deficit
-175.0	7	4.35	- 0.0323	-0.1405	Surplus
-179.35	8	0.65	+ 0.1149	+0.0747	Deficit
-180.0					
			$E_{\min} = \Sigma$	5.8788	

$$\Delta T = 8.0^{\circ}\text{C}$$

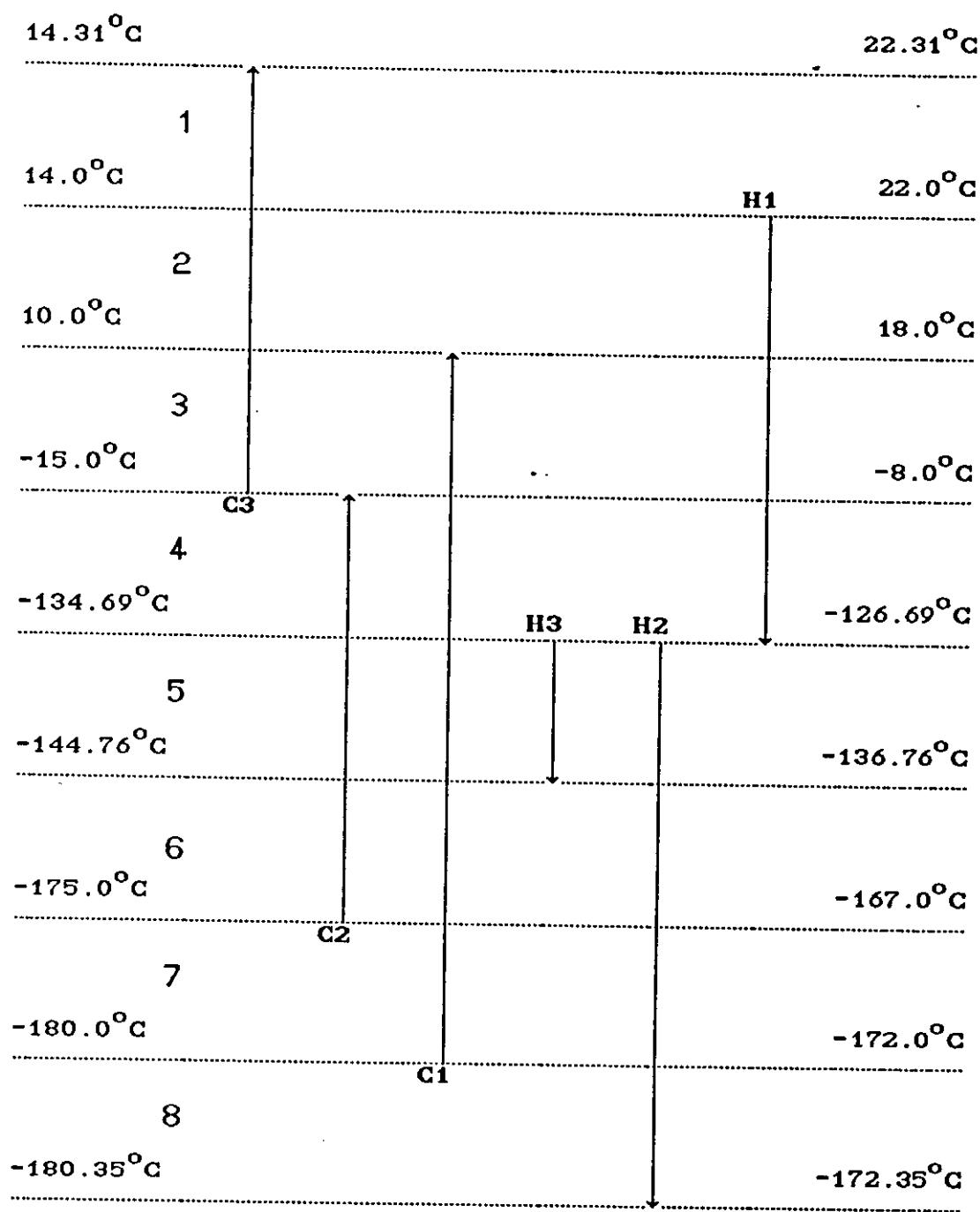


Figure A-9 : Temperature - interval analysis

TABLE A.9 : TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 8.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma \text{CP}_{\text{cold}} - \Sigma \text{CP}_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
14.31	1	0.31	+ 0.0870	+0.0270	Deficit
14.0	2	4.0	- 0.3281	-1.3124	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	119.69	- 0.0315	-3.7702	Surplus
-134.69	5	10.07	- 0.2412	-2.4289	Surplus
-144.76	6	30.24	+ 0.2364	+7.1487	Deficit
-175.0	7	5.0	- 0.0323	-0.1615	Surplus
-180.0	8	0.35	- 0.4151	-0.1453	Surplus
-180.35					
			$E_{\min} = \Sigma$	5.9726	

$$\Delta T = 9.0^{\circ}\text{C}$$

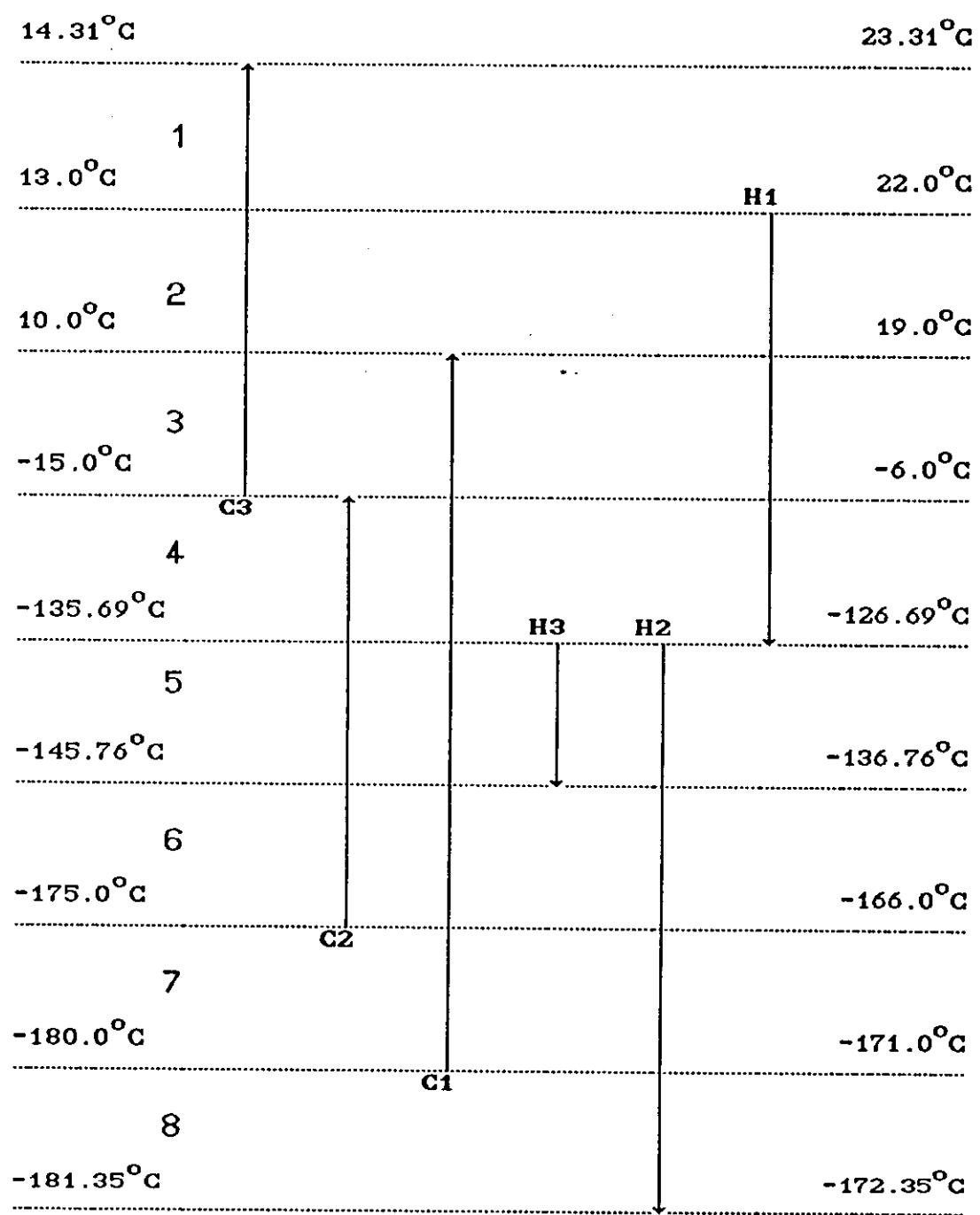


Figure A-10 : Temperature - interval analysis

TABLE A.10: TEMPERATURE-INTERVAL ANALYSIS, $\Delta T = 9.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma CP_{\text{cold}} - \Sigma CP_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
14.31	1	1.31	+ 0.0870	+0.1140	Deficit
13.0	2	3.0	- 0.3281	-0.9843	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	120.69	- 0.0315	-3.8017	Surplus
-135.69	5	10.07	- 0.2412	-2.4289	Surplus
-145.76	6	29.24	+ 0.2364	+6.9123	Deficit
-175.0	7	5.0	- 0.0323	-0.1615	Surplus
-180.0	8	1.35	- 0.4151	-0.5604	Surplus
-181.35					
			$E_{\min} = \Sigma$	6.2405	

$$\Delta T = 10.0^{\circ}\text{C}$$

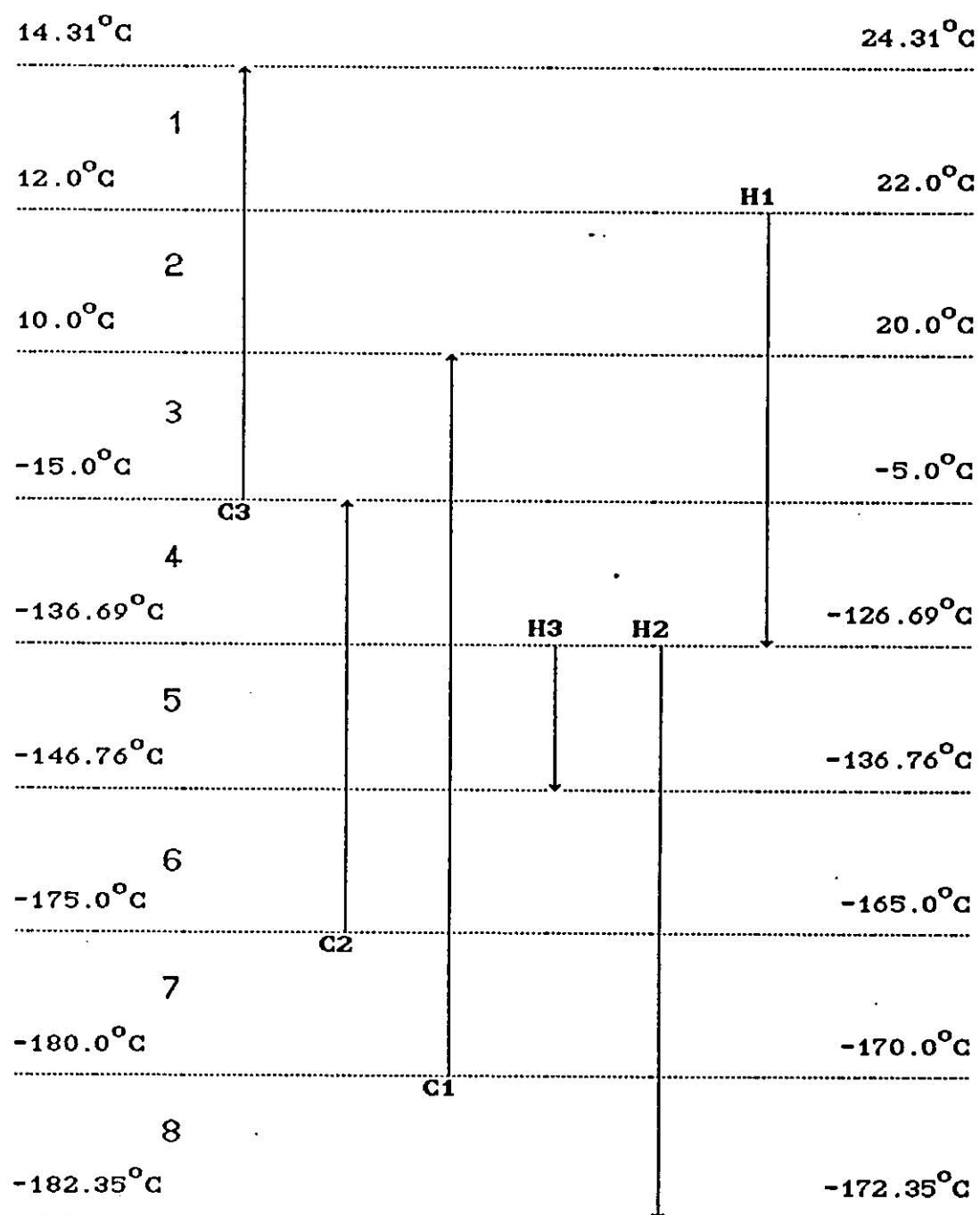


Figure A-11 : Temperature - interval analysis

TABLE A.11: TEMPERATURE-INTERVAL ANALYSIS, $\Delta T=10.0^{\circ}\text{C}$

$^{\circ}\text{C}$	Interval No. (i)	$T_i - T_{i+1}$ ($^{\circ}\text{C}$)	$\Sigma \text{CP}_{\text{cold}} - \Sigma \text{CP}_{\text{hot}}$ ($\text{kW}/^{\circ}\text{C}$)	ΔH_i (kW)	Surplus or Deficit
14.31	1	2.31	+ 0.0870	+0.2010	Deficit
14.0	2	2.0	- 0.3281	-0.6562	Surplus
10.0	3	25.0	- 0.2132	-5.33	Surplus
-15.0	4	121.69	- 0.0315	-3.8332	Surplus
-136.69	5	10.07	- 0.2412	-2.4289	Surplus
-146.76	6	28.24	+ 0.2364	+6.6759	Deficit
-175.0	7	5.0	- 0.0323	-0.1615	Surplus
-180.0	8	2.35	- 0.4151	-0.9755	Surplus
-182.35					
			$E_{\min} = \Sigma$	6.5084	

Appendix B

Computer results of the simula-
ting flowsheet of the oxygen
plant of type K-0.15

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986
 All Rights Reserved.

TOPOLOGY

Equipment Stream Numbers

1	ENTM	1	-2	0	0
2	ENTM	3	-4	0	0
3	ENTM	5	-6	0	0
4	ENTM	7	-8	0	0
5	HXER	2	-3	0	0
6	HXER	4	-5	0	0
7	HXER	6	-7	0	0
8	HXER	8	-9	0	0
9	HXER	9	48	-10	-49
10	HXER	10	36	-11	-38
11	HXER	13	44	-15	-47
12	HXER	14	45	-16	-46
13	HXER	11	-12	0	0
14	DVDR	12	-13	-14	0
15	HXER	46	49	-51	-36
16	MIXR	15	16	-17	0
17	DVDR	17	-18	-19	0
18	ENTM	19	-26	0	0
19	DVDR	18	-20	-21	0
20	HXER	43	20	-44	-23
21	HXER	42	21	-45	-22
22	MIXR	23	22	-24	0
23	VALV	24	-25	0	0
24	TOWR	25	26	-27	-28
25	HXER	27	34	-29	-39
26	HXER	28	39	-30	-40
27	HXER	31	35	-33	-37
28	VALV	29	-32	0	0
29	VALV	30	-31	0	0
30	TOWR	32	33	-34	-35
31	PUMP	37	-41	0	0
32	DVDR	47	-48	-50	0
33	HXER	40	-43	0	0
34	HXER	41	-42	0	0
35	MIXR	38	50	-52	0

Stream Connections

Stream	Equipment	From	To
1		0	1
2		1	5
3		5	2
4		2	6
5		6	3
6		3	7

CODE REPORT

Stream Connections

138

Stream	Equipment	
From	To	
7	7	4
8	4	8
9	8	9
10	9	10
11	10	13
12	13	14
13	14	11
14	14	12
15	11	16
16	12	16
17	16	17
18	17	19
19	17	18
20	19	20
21	19	21
22	21	22
23	20	22
24	22	23
25	23	24
26	18	24
27	24	25
28	24	26
29	25	28
30	26	29
31	29	27
32	28	30
33	27	30
34	30	25
35	30	27
36	15	10
37	27	31
38	10	35
39	25	26
40	26	33
41	31	34
42	34	21
43	33	20
44	20	11
45	21	12
46	12	15
47	11	32
48	32	9
49	9	15
50	32	35
51	15	0
52	35	0

COMPONENTS 2

ID numbers 46, 47,
CODE REPORT

THERMODYNAMICS

Kvalue option: Peng-Robinson
Enthalpy option: Peng-Robinson

Density option! API method

139

MISCELLANEOUS

Recycle calculations are converged.

Recycle equipment list (KE2): 13, 14, 17, 18, 19, 33, 20, 11,
 31, 34, 21, 22, 23, 24, 12, 15,
 25, 26, 28, 29, 27, 30,

Streams used in conv. routine (KE4): (0)
 ()=Delay factor

Preferred cut stream list (KE3): 11, 17, 40, 37, 34, 35,

Convergence tolerances,	Error
Flowrates:	.00100000
Vapor fraction:	.00100000
Temperature:	.00100000
Pressure:	.00100000
Enthalpy:	.00100000
Flash calcs:	.00005000

Max. loops in recycle calc.: 30
 in flash calcs: 75

CORE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

*** DIVIDERS ***	14	17	19	32
Equipment no.				
External name				
Fraction/flow	1 .68750	.40000	.71000	.33000
	2 .31250	.60000	.29000	.67000
	3 .00000	.00000	.00000	.00000
	4 .00000	.00000	.00000	.00000
	5 .00000	.00000	.00000	.00000
	6 .00000	.00000	.00000	.00000
Mode	.0	.0	.0	.0
Stream #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
Comp. ID #	.0	.0	.0	.0
.....				

*** MIXERS-W/FLASH ***	16	22	35
Equipment no.			
External name			
.....			

*** VALVES ***			
.....			

Equipment no.	23	28	29	
External name				140
Outlet pressure bars	6.6132	1.4006	1.4572	
EXCHANGER/CONDENSERS				
Equipment no.	5	6	7	8
External name				
Heat transfer coeff.	.00000	.00000	.00000	.00000
Area	.00000	.00000	.00000	.00000
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	5.0	5.0	5.0	5.0
Min. delta T or T-out	20.00	20.00	20.00	22.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	-158.52	-183.56	-164.72	-106.13
Water usage, DM3/hr	.00000	.00000	.00000	.00000
Corrected delta T	.00	.00	.00	.00
Equipment no.	9	10	11	12
External name				
Heat transfer coeff.	300.00	300.00	300.00	300.00
Area	1.5385	4.4356	15.772	6.3643
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	3.0	3.0	3.0	3.0
Min. delta T or T-out	10.00	1.00	25.00	28.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	-8.4539	-9.8046	-142.49	-64.454
Water usage, DM3/hr	.00000	.00000	.00000	.00000
Corrected delta T	18.32	7.37	30.11	33.76
Equipment no.	13	15	20	21
External name				
Heat transfer coeff.	.00000	300.00	300.00	300.00
Area	.00000	20.847	1.1028	.40436
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	5.0	3.0	3.0	3.0
Min. delta T or T-out	10.00	1.00	36.00	40.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	3.0318	9.0809	12.279	5.0450
Water usage, DM3/hr	.00000	.00000	.00000	.00000
Corrected delta T	8.63	1.45	37.11	41.59
Equipment no.	25	26	27	33
External name				
Heat transfer coeff.	300.00	300.00	300.00	.00000
Area	5.7545	5.8885	.53237	.00000
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	3.0	3.0	3.0	5.0
Min. delta T or T-out	6.00	3.00	7.00	-175.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	-10.610	-7.1362	1.3775	.15206

Water usage, DM3/hr	.00000	100000	100000	.00000	111
Corrected delta T	6.15	4.04	8.63	8.63	
Equipment no.	34				
External name					
Heat transfer coeff.	.00000				
Area	.00000				
Number of shells	.0				
Shell passes	.0				
Tube passes	.0				
Mode	5.0				
Min. delta T or T-out	-180.00				
Delta P, stream 1	.00000				
Delta P, stream 2	.00000				
Q, stream 1 MJ/hr	1.1303				
Water usage, DM3/hr	.00000				
Corrected delta T	18.32				

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986

All Rights Reserved.

*** PUMPS/COMPRESSORS ***

Equipment no.	31			
External name				
Number of stages	.0			
Work capacity MJ/hr	.00000			
Outlet pressure bars	190.00			
Power type:	.0			
(+) steam				
(O) electricity				
(-) fuel gas				
H, steam out KJ / kg	.00000			
Fuel usage, M3 /hr	.00000			
Water usage, DM3/hr	.00000			
Steam usage Tonnes/hr	.00000			
Kilowatt usage	1.1632			

*** ENTROPY MACHINES ***

Equipment no.	1	2	3	4
External name				
Mode	.0	.0	.0	.0
Outlet pressure bars	2.7100	8.2300	22.300	43.600
Adiabatic efficiency	.70000	.70000	.70000	.70000
Theoretical hp	41.181	47.389	41.786	26.671
Actual horsepower	58.831	67.699	59.694	38.101
Entropy in KJ/deg C	1140.9	812.34	438.03	94.006
Entropy out, ideal	1140.9	812.33	438.02	93.919
Entropy out, actual	1257.2	940.77	555.43	177.31
Equipment no.	18			
External name				
Mode	.0			
Outlet pressure bars	6.6065			
Adiabatic efficiency	.85000			
Theoretical hp	-10.517			
Actual horsepower	-8.9396			
Entropy in KJ/deg C	-733.70			
Entropy out, ideal	-733.72			

Entropy out, actual -693.70

142

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986

All Rights Reserved.

*** RIGOROUS TOWERS ***

Equipment no.	24	30
External name		
Number of stages	21.0	57.0
Feed 1, stage #	15.0	1.0
Feed 2, stage #	21.0	17.0
Feed 3, stage #	.0	.0
Feed 4, stage #	.0	.0
Feed 5, stage #	.0	.0
Sidestream # 1 stage	.0	.0
Sidestream # 2 stage	.0	.0
Sidestream # 3 stage	.0	.0
Sidestream # 4 stage	.0	.0
Cond. pressure bars	6.7131	1.4006
Cond. delta P bars	.00000	.00000
Colm. delta P bars	.19400E-01	.19850
Condenser type	.0	1.0
Condenser mode	4.0	.0
Value of cond. spec.	16.000	.00000
Cond comp 1 position	1.0	.0
Cond comp 2 position	.0	.0
Cond. deg. subcooled	1.00	.00
Reboiler mode	.0	4.0
Val. of reboiler spec	.00000	7.6000
Rebr comp 1 position	.0	2.0
Rebr comp 2 position	.0	.0
Damping ratio	.00000	1.0000
Sidestream 1 mode	.0	.0
Sidestream 2 mode	.0	.0
Sidestream 3 mode	.0	.0
Sidestream 4 mode	.0	.0
Sidestream # 1 spec	.00000	.00000
Sidestream # 2 spec	.00000	.00000
Sidestream # 3 spec	.00000	.00000
Sidestream # 4 spec	.00000	.00000
Sidestrm 1 comp posn	.0	.0
Sidestrm 2 comp posn	.0	.0
Sidestrm 3 comp posn	.0	.0
Sidestrm 4 comp posn	.0	.0
Cond. duty MJ/hr	-163.51	.00000
Rebr duty MJ/hr	.00000	153.21
Est. dist rate Kgmol/hr	16.000	.00000
Est. reflux Kgmol/hr	.00000	.00000
Est. side draw rate 1	.00000	.00000
Est. side draw rate 2	.00000	.00000
Est. side draw rate 3	.00000	.00000
Est. side draw rate 4	.00000	.00000
Est. temp stg 1 C	-178.00	-193.00
Est. temp stg N C	-174.00	-183.00

CODEREPOR

*** Stream no. 1 ***

	All Vapor
Temperature deg C.	20.0000
Pressure bars.	1.01325
Enthalpy MJ/hr	812.759
Entropy MJ/hr*K	1.14091
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	1.19674
Viscosity centipoise180405E-01
Thermal cond. cal/cm*s*K628029E-04
Specific heat kJ/kg*K	1.01667
Z factor999419
m3/hr (15.6 deg C & 1 atm)	947.608
Vol. flowrate m3/hr	961.641
	Vapor
	mole
	fraction
Nitrogen810000
Oxygen190000

*** Stream no. 2 ***

	All Vapor
Temperature deg C.	154.633
Pressure bars.	2.71000
Enthalpy MJ/hr	970.718
Entropy MJ/hr*K	1.25721
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	2.19114
Viscosity centipoise239542E-01
Thermal cond. cal/cm*s*K907807E-04
Specific heat kJ/kg*K	1.02916
Z factor	1.00045
m3/hr (15.6 deg C & 1 atm)	947.608
Vol. flowrate m3/hr	525.220
	Vapor
	mole
	fraction
Nitrogen810000
Oxygen190000

CODE REPORT

*** Stream no. 3 ***

	All Vapor
Temperature deg C.	20.0000
Pressure bars.	2.71000
Enthalpy MJ/hr	812.202
Entropy MJ/hr*K812340
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	3.20381
Viscosity centipoise180405E-01
Thermal cond. cal/cm*s*K628029E-04
Specific heat kJ/kg*K	1.02047
Z factor998463

MJ/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr
 Vapor mole fraction
 Nitrogen .810000
 Oxygen .190000

947.608
 359.206
 Vapor
 flowrate
 kgmol/hr
 32.4000
 7.60000

*** Stream no. 4 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr

All Vapor
 174.770
 8.23000
 993.971
 .940765
 28.7703
 1150.81
 40.0000
 6.34707
 .247552E-01
 .957600E-04
 1.03568
 1.00172
 947.608
 181.317
 Vapor mole fraction
 Nitrogen .810000
 Oxygen .190000

CODE REPORT

*** Stream no. 5 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr

All Vapor
 20.0000
 8.23000
 810.408
 .438028
 28.7703
 1150.81
 40.0000
 9.75871
 .180405E-01
 .628029E-04
 1.03134
 .995489
 947.608
 117.928
 Vapor mole fraction
 Nitrogen .810000
 Oxygen .190000

*** Stream no. 6 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr
 Entropy MJ/hr*K
 Ave. mol. wt.

All Vapor
 156.639
 22.3000
 970.685
 .555435
 28.7703

Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m ³	17.8700
Viscosity centipoise240468E-01
Thermal cond. cal/cm*s*K912667E-04
Specific heat kJ/kg*K	1.04437
Z factor	1.00422
m ³ /hr (15.6 deg C & 1 atm) .	947.608
Vol. flowrate m ³ /hr	64.3682
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen810000
Oxygen190000

CODE REPORT

*** Stream no. 7 ***

Temperature deg C.	A11 Vapor
Pressure bars.	20.0000
Enthalpy MJ/hr	22.3000
Entropy MJ/hr*K	805.964
Ave. mol. wt.940061E-01
Total flow kg/hr.	28.7703
kgmol/hr	1150.81
Density kg/m ³	40.0000
Viscosity centipoise	26.6192
Thermal cond. cal/cm*s*K180657E-01
Specific heat kJ/kg*K628029E-04
Z factor	1.05903
m ³ /hr (15.6 deg C & 1 atm) .	.988871
Vol. flowrate m ³ /hr	947.608
Vapor	43.2331
mole	Vapor
fraction	flowrate
Nitrogen	kgmol/hr
Oxygen810000

*** Stream no. 8 ***

Temperature deg C.	A11 Vapor
Pressure bars.	107.532
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	908.265
Ave. mol. wt.177305
Total flow kg/hr.	28.7703
kgmol/hr	1150.81
Density kg/m ³	40.0000
Viscosity centipoise	39.4742
Thermal cond. cal/cm*s*K220565E-01
Specific heat kJ/kg*K799859E-04
Z factor	1.06555
m ³ /hr (15.6 deg C & 1 atm) .	1.00399
Vol. flowrate m ³ /hr	947.608
Vapor	29.1540
mole	Vapor
fraction	flowrate
Nitrogen	kgmol/hr
Oxygen810000

CODE REPORT

*** Stream no. 9 ***

All Vapor

Temperature deg C.	22.0000
Pressure bars.	43.6000
Enthalpy MJ/hr	802.137
Entropy MJ/hr*K	-.138655
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	52.0371
Viscosity centipoise	.182234E-01
Thermal cond. cal/cm*s*K	.631572E-04
Specific heat kJ/kg*K	1.09814
Z factor	.982313
m3/hr (15.6 deg C & 1 atm)	947.608
Vol. flowrate m3/hr	22.1156
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .810000	32.4000
Oxygen .190000	7.60000

*** Stream no. 10 ***

All Vapor

Temperature deg C.	15.3137
Pressure bars.	43.6000
Enthalpy MJ/hr	793.683
Entropy MJ/hr*K	-.167679
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	53.4060
Viscosity centipoise	.179041E-01
Thermal cond. cal/cm*s*K	.619794E-04
Specific heat kJ/kg*K	1.10248
Z factor	.979320
m3/hr (15.6 deg C & 1 atm)	947.608
Vol. flowrate m3/hr	21.5487
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .810000	32.4000
Oxygen .190000	7.60000

CODE REPORT

*** Stream no. 11 ***

All Vapor

Temperature deg C.	7.59766
Pressure bars.	43.6000
Enthalpy MJ/hr	783.879
Entropy MJ/hr*K	-.202191
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	55.0894
Viscosity centipoise	.175316E-01
Thermal cond. cal/cm*s*K	.606436E-04
Specific heat kJ/kg*K	1.10845
Z factor	.975487

m3/hr (15.6 deg C & 1 atm) .	947.608
Vol. flowrate m3/hr	20.8902
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen .810000	32.4000
Oxygen .190000	7.60000

147

*** Stream no. 12 ***

Temperature deg C.	All Vapor
Pressure bars.	10.0000
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	786.911
Ave. mol. wt.	- .191324
Total flow kg/hr.	28.7703
kgmol/hr	1150.81
Density kg/m3	40.0000
Viscosity centipoise	54.5527
Thermal cond. cal/cm*s*K176482E-01
Specific heat kJ/kg*K610568E-04
Z factor	1.10737
m3/hr (15.6 deg C & 1 atm) .	.976727
Vol. flowrate m3/hr	947.608
Vapor	
mole	
fraction	
Nitrogen .810000	21.0957
Oxygen .190000	32.4000

CODE REPORT

*** Stream no. 13 ***

Temperature deg C.	All Vapor
Pressure bars.	10.0000
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	541.001
Ave. mol. wt.	- .131536
Total flow kg/hr.	28.7703
kgmol/hr	791.184
Density kg/m3	27.5000
Viscosity centipoise	54.5527
Thermal cond. cal/cm*s*K176482E-01
Specific heat kJ/kg*K610568E-04
Z factor	1.10752
m3/hr (15.6 deg C & 1 atm) .	.976727
Vol. flowrate m3/hr	651.480
Vapor	
mole	
fraction	
Nitrogen .810000	14.5033
Oxygen .190000	22.2750

*** Stream no. 14 ***

Temperature deg C.	All Vapor
Pressure bars.	10.0000
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	245.910
Ave. mol. wt.	- .597889E-01

Vapor	
flowrate	
kgmol/hr	
Nitrogen .810000	28.7703
Oxygen .190000	5.22500

Total flow kg/hr.	359.629	
kgmol/hr	12.5000	148
Density kg/m3	54.5527	
Viscosity centipoise176482E-01	
Thermal cond. cal/cm*s*K610568E-04	
Specific heat kJ/kg*K	1.10704	
Z factor976727	
m3/hr (15.6 deg C & 1 atm)	296.127	
Vol. flowrate m3/hr	6.59242	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen .810000	10.1250	
Oxygen .190000	2.37500	
CODE REPORT		
*** Stream no. 15 ***		
Temperature deg C.	All Vapor -126.812	
Pressure bars.	43.6000	
Enthalpy MJ/hr	398.507	
Entropy MJ/hr*K	-.842187	
Ave. mol. wt.	28.7703	
Total flow kg/hr.	791.184	
kgmol/hr	27.5000	
Density kg/m3	166.452	
Viscosity centipoise106017E-01	
Thermal cond. cal/cm*s*K409996E-04	
Specific heat kJ/kg*K	2.37539	
Z factor619385	
m3/hr (15.6 deg C & 1 atm)	651.480	
Vol. flowrate m3/hr	4.75330	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen .810000	22.2750	
Oxygen .190000	5.22500	
*** Stream no. 16 ***		
Temperature deg C.	All Vapor -126.440	
Pressure bars.	43.6000	
Enthalpy MJ/hr	181.456	
Entropy MJ/hr*K	-.380657	
Ave. mol. wt.	28.7703	
Total flow kg/hr.	359.629	
kgmol/hr	12.5000	
Density kg/m3	164.835	
Viscosity centipoise106149E-01	
Thermal cond. cal/cm*s*K410453E-04	
Specific heat kJ/kg*K	2.33661	
Z factor623874	
m3/hr (15.6 deg C & 1 atm)	296.127	
Vol. flowrate m3/hr	2.18179	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen .810000	10.1250	
Oxygen .190000	2.37500	
CODE REPORT		

*** Stream no. 17 ***

	All Vapor
Temperature deg C.	-126.696
Pressure bars.	43.6000
Enthalpy MJ/hr	579.963
Entropy MJ/hr*K	-1.22284
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	165.942
Viscosity centipoise106058E-01
Thermal cond. cal/cm*s*K410139E-04
Specific heat kJ/kg*K	2.36246
Z factor620796
m3/hr (15.6 deg C & 1 atm)	947.608
Vol. flowrate m3/hr	6.93515
	Vapor
	mole
	fraction
Nitrogen810000
Oxygen190000
	Vapor
	flowrate
	kgmol/hr
	32.4000
	7.60000

*** Stream no. 18 ***

	All Vapor
Temperature deg C.	-126.695
Pressure bars.	43.6000
Enthalpy MJ/hr	231.985
Entropy MJ/hr*K	-.489127
Ave. mol. wt.	28.7703
Total flow kg/hr.	460.325
kgmol/hr	16.0000
Density kg/m3	165.937
Viscosity centipoise106058E-01
Thermal cond. cal/cm*s*K410140E-04
Specific heat kJ/kg*K	2.36314
Z factor620808
m3/hr (15.6 deg C & 1 atm)	379.043
Vol. flowrate m3/hr	2.77413
	Vapor
	mole
	fraction
Nitrogen810000
Oxygen190000
	Vapor
	flowrate
	kgmol/hr
	12.9600
	3.04000

CODE REPORT

*** Stream no. 19 ***

	All Vapor
Temperature deg C.	-126.695
Pressure bars.	43.6000
Enthalpy MJ/hr	347.978
Entropy MJ/hr*K	-.733690
Ave. mol. wt.	28.7703
Total flow kg/hr.	690.488
kgmol/hr	24.0000
Density kg/m3	165.937
Viscosity centipoise106058E-01
Thermal cond. cal/cm*s*K410140E-04
Specific heat kJ/kg*K	2.36291
Z factor620808

m³/hr (15.6 deg C & 1 atm)
 Vol. flowrate m³/hr 4.16120

	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.810000	19.4400
Oxygen	.190000	4.56000

*** Stream no. 20 ***

Temperature deg C.	All Vapor
Pressure bars.	-126.695
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	164.710
Ave. mol. wt.	-.347280
Total flow kg/hr.	28.7703
kgmol/hr	326.831
Density kg/m ³	11.3600
Viscosity centipoise	165.937
Thermal cond. cal/cm*s*K106058E-01
Specific heat kJ/kg*K410140E-04
Z factor	2.36313
m ³ /hr (15.6 deg C & 1 atm)620808
Vol. flowrate m ³ /hr	269.121
	Vapor	1.96963
	mole	
Nitrogen	.810000	Vapor
Oxygen	.190000	flowrate

CODE REPORT

*** Stream no. 21 ***

Temperature deg C.	All Vapor
Pressure bars.	-126.695
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	67.2758
Ave. mol. wt.	-.141847
Total flow kg/hr.	28.7703
kgmol/hr	133.494
Density kg/m ³	4.64000
Viscosity centipoise	165.937
Thermal cond. cal/cm*s*K106058E-01
Specific heat kJ/kg*K410140E-04
Z factor	2.36258
m ³ /hr (15.6 deg C & 1 atm)620808
Vol. flowrate m ³ /hr	109.922
	Vapor	.804498
	mole	
Nitrogen	.810000	Vapor
Oxygen	.190000	flowrate

*** Stream no. 22 ***

Temperature deg C.	All Vapor
Pressure bars.	-136.781
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	62.2307
Ave. mol. wt.	-.177787
	28.7703	

Total flow kg/hr	133.494
kgmol/hr	4.64000
Density kg/m ³	267.904
Viscosity centipoise106456E-01
Thermal cond. cal/cm*s*K397891E-04
Specific heat kJ/kg*K	7.77801
Z factor412963
m ³ /hr (15.6 deg C & 1 atm)	109.922
Vol. flowrate m ³ /hr498299
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .810000	3.75840
Oxygen .190000	.881600

CODE REPORT

*** Stream no. 23 ***

Temperature deg C.	All Vapor
Pressure bars.	-136.752
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	152.431
Ave. mol. wt.	-.434724
Total flow kg/hr.	28.7703
kgmol/hr	326.831
Density kg/m ³	11.3600
Viscosity centipoise	267.071
Thermal cond. cal/cm*s*K106420E-01
Specific heat kJ/kg*K397925E-04
Z factor	7.70562
m ³ /hr (15.6 deg C & 1 atm)414162
Vol. flowrate m ³ /hr	269.121
Vapor mole fraction	1.22378
Nitrogen .810000	Vapor flowrate kgmol/hr
Oxygen .190000	9.20160

*** Stream no. 24 ***

Temperature deg C.	All Vapor
Pressure bars.	-136.760
Enthalpy MJ/hr	43.6000
Entropy MJ/hr*K	214.662
Ave. mol. wt.	-.612499
Total flow kg/hr.	28.7703
kgmol/hr	460.325
Density kg/m ³	16.0000
Viscosity centipoise	267.300
Thermal cond. cal/cm*s*K106430E-01
Specific heat kJ/kg*K397916E-04
Z factor	7.72551
m ³ /hr (15.6 deg C & 1 atm)413833
Vol. flowrate m ³ /hr	379.043
Vapor mole fraction	1.72216
Nitrogen .810000	Vapor flowrate kgmol/hr
Oxygen .190000	12.9600

CODE REPORT

*** Stream no. 25 ***

152

	Overall	Vapor	Liquid
Temperature deg C.	-172.366		
Pressure bars.	6.61325		
Vapor fraction .	.817598		
Enthalpy MJ/hr .	214.662	187.666	27.0110
Entropy MJ/hr*K .	-.474266	-.269560	-.204705
Ave. mol. wt. .	28.7703	28.6655	29.2409
Total flow kg/hr .	460.325	374.990	85.3375
kgmol/hr .	16.0000	13.0816	2.91843
Density kg/m3 .		26.7024	1406.48
Viscosity centipoise .		.690154E-02	.103313
Thermal cond. cal/cm*s*k .		.357732E-04	.505024E-03
Specific heat kJ/kg*K .		1.25170	2.17381
Z factor .		.847052	
m3/hr (15.6 deg C & 1 atm) .		309.974	
Vol. flowrate m3/hr .		14.0435	
Surface tension dyne/cm. .			5.67519
S. G. (60/60) .			.915033
m3/hr (15.6 deg C & 1 atm) .			.934078E-01
Vol. flowrate m3/hr .			.606751E-01
	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.83630	.69196	10.9401
Oxygen	.16370	.30804	2.14142

*** Stream no. 26 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.347		
Pressure bars. .	6.60647		
Vapor fraction .	.832393		
Enthalpy MJ/hr .	323.794	286.507	37.2739
Entropy MJ/hr*K .	-.693828	-.411175	-.282652
Ave. mol. wt. .	28.7703	28.6730	29.2536
Total flow kg/hr .	690.488	572.812	117.675
kgmol/hr .	24.0000	19.9774	4.02257
Density kg/m3 .		26.6790	1407.54
Viscosity centipoise .		.690418E-02	.103845
Thermal cond. cal/cm*s*k .		.357657E-04	.504985E-03
Specific heat kJ/kg*K .		1.24986	2.17363
Z factor .		.847282	
m3/hr (15.6 deg C & 1 atm) .		473.210	
Vol. flowrate m3/hr .		21.4708	
Surface tension dyne/cm. .			5.69054
S. G. (60/60) .			.915375
m3/hr (15.6 deg C & 1 atm) .			.128755
Vol. flowrate m3/hr .			.836040E-01
	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.83443	.68876	16.6698
Oxygen	.16557	.31124	3.30767
CODE REPORT			1.25197

*** Stream no. 27 ***

	All Liquid
Temperature deg C.	-175.941
Pressure bars.	6.71314

Enthalpy MJ/hr	152.101
Entropy MJ/hr*K	-1.12251
Ave. mol. wt.	28.0883
Total flow kg/hr.	449.412
kgmol/hr	15.9999
Density kg/m3	1329.92
Viscosity centipoise947307E-01
Thermal cond. cal/cm*s*K510531E-03
Specific heat kJ/kg*K	2.34647
Surface tension dyne/cm.	4.74317
S. G. (60/60)883885
m3/hr (15.6 deg C & 1 atm)509248
Vol. flowrate m3/hr337928
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen981097
Oxygen189026E-01

153

*** Stream no. 28 ***

Temperature deg C.	All Liquid
Pressure bars.	-172.151
Enthalpy MJ/hr	6.73254
Entropy MJ/hr*K	222.843
Ave. mol. wt.	-1.68107
Total flow kg/hr.	29.2250
kgmol/hr	701.402
Density kg/m3	,24.0001
Viscosity centipoise	1402.47
Thermal cond. cal/cm*s*K102868
Specific heat kJ/kg*K504666E-03
Surface tension dyne/cm.	2.18357
S. G. (60/60)	5.60304
m3/hr (15.6 deg C & 1 atm)914607
Vol. flowrate m3/hr768091
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen695936
Oxygen304064

CODE REPORT

*** Stream no. 29 ***

Temperature deg C.	All Liquid
Pressure bars.	-186.647
Enthalpy MJ/hr	6.71314
Entropy MJ/hr*K	141.492
Ave. mol. wt.	-1.23802
Total flow kg/hr.	28.0883
kgmol/hr	449.412
Density kg/m3	15.9999
Viscosity centipoise	1445.97
Thermal cond. cal/cm*s*K123409
Specific heat kJ/kg*K527857E-03
Surface tension dyne/cm.	2.09301
S. G. (60/60)	6.98401
m3/hr (15.6 deg C & 1 atm)883885
Vol. flowrate m3/hr509248
	.310805

	Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen	.981097	15.6975
Oxygen	.189026E-01	.302440

*** Stream no. 30 ***

	All Liquid	
Temperature deg C.	-176.939	
Pressure bars.	6.73254	
Enthalpy MJ/hr	215.707	
Entropy MJ/hr*K	-1.75344	
Ave. mol. wt.	29.2250	
Total flow kg/hr.	701.402	
kgmol/hr	24.0001	
Density kg/m ³	1454.69	
Viscosity centipoise	.113785	
Thermal cond. cal/cm*s*K	.512414E-03	
Specific heat kJ/kg*K	2.07018	
Surface tension dyne/cm.	6.67743	
S. G. (60/60)	.914607	
m ³ /hr (15.6 deg C & 1 atm)	.768091	
Vol. flowrate m ³ /hr	.482172	
	Liquid	
	mole	
	fraction	
Nitrogen	.695936	16.7025
Oxygen	.304064	7.29756

CODE REPORT

*** Stream no. 31 ***

	Overall	Vapor	Liquid	
Temperature deg C.	-190.027			
Pressure bars.	1.45725			
Vapor fraction	.138541			
Enthalpy MJ/hr	215.707	46.6821	168.992	
Entropy MJ/hr*K	-1.73074	-.409657E-01	-1.68978	
Ave. mol. wt.	29.2250	28.5267	29.3371	
Total flow kg/hr	701.402	94.8512	606.546	
kgmol/hr	24.0001	3.32499	20.6751	
Density kg/m ³		6.34647	1590.31	
Viscosity centipoise		.566974E-02	.161615	
Thermal cond. cal/cm*s*K		.339249E-04	.533619E-03	
Specific heat kJ/kg*K		1.10370	1.84549	
Z factor		.949507		
m ³ /hr (15.6 deg C & 1 atm)		78.6264		
Vol. flowrate m ³ /hr		14.9457		
Surface tension dyne/cm.			9.92894	
S. G. (60/60)			.917617	
m ³ /hr (15.6 deg C & 1 atm)			.662037	
Vol. flowrate m ³ /hr			.381406	
	Vapor	Liquid	Vapor	
	mole	mole	flowrate	
	fraction	fraction	kgmol/hr	
Nitrogen	.87111	.66782	2.89644	13.8073
Oxygen	.12889	.33218	.428552	6.86778

*** Stream no. 32 ***

	Overall	Vapor	Liquid
Temperature deg C.	-192.652		

Pressure bars. . .	1.40055			
Vapor fraction . .	.668986E-01			
Enthalpy MJ/hr . .	141.492	14.7390	126.654	
Entropy MJ/hr*K . .	-1.23346	-.138828E-01	-1.21957	
Ave. mol. wt. . .	28.0883	28.0396	28.0918	
Total flow kg/hr . .	449.412	30.0129	419.398	
kgmol/hr . .	15.9999	1.07037	14.9296	
Density kg/m3		6.29450	1503.80	
Viscosity centipoise541943E-02	.148024	
Thermal cond. cal/cm*s*K336861E-04	.537568E-03	
Specific heat kJ/kg*K		1.13005	1.97940	
Z factor947868		
m3/hr (15.6 deg C & 1 atm)		24.9376		
Vol. flowrate m3/hr		4.76818		
Surface tension dyne/cm.			8.31469	
S. G. (60/60)883978	
m3/hr (15.6 deg C & 1 atm)475188	
Vol. flowrate m3/hr278895	
Nitrogen	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr
Oxygen99332 .98024	.98024 .99332	1.06322 .715335E-02	14.6345 .295055

CODE REPORT

*** Stream no. 33 ***				
	Overall	Vapor	Liquid	
Temperature deg C.	-190.008			
Pressure bars.	1.45725			
Vapor fraction148894			
Enthalpy MJ/hr	217.084	50.2428	166.874	
Entropy MJ/hr*K	-1.71320	-.440183E-01	-1.66918	
Ave. mol. wt.	29.2250	28.5310	29.3463	
Total flow kg/hr	701.402	101.955	599.446	
kgmol/hr	24.0001	3.57346	20.4266	
Density kg/m3		6.33689	1590.93	
Viscosity centipoise567168E-02	.161666	
Thermal cond. cal/cm*s*K339457E-04	.533592E-03	
Specific heat kJ/kg*K		1.10343	1.84441	
Z factor949532		
m3/hr (15.6 deg C & 1 atm)		84.6209		
Vol. flowrate m3/hr		16.0893		
Surface tension dyne/cm.			9.94126	
S. G. (60/60)917866	
m3/hr (15.6 deg C & 1 atm)654110	
Vol. flowrate m3/hr376793	
Nitrogen	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr
Oxygen87004 .66549	.66549 .87004	3.10905 .464419	13.5938 6.83284

*** Stream no. 34 ***

	All Vapor		
Temperature deg C.	-192.647		
Pressure bars.	1.40055		
Enthalpy MJ/hr	453.661		
Entropy MJ/hr*K	-.421952		
Ave. mol. wt.	28.0407		
Total flow kg/hr.	908.518		

	kgmol/hr	32.4000
Density kg/m ³	6.19011
Viscosity centipoise541992E-02
Thermal cond. cal/cm*s*K339356E-04
Specific heat kJ/kg*K	1.13027
Z factor947875
m ³ /hr (15.6 deg C & 1 atm)	767.562
Vol. flowrate m ³ /hr	146.771
	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.993053	32.1749
Oxygen	.694701E-02	.225083

CODE REPORT

*** Stream no. 35 ***

	All Liquid	
Temperature deg C.	-179.524
Pressure bars.	1.59905
Enthalpy MJ/hr	58.1288
Entropy MJ/hr*K	-.604788
Ave. mol. wt.	31.8810
Total flow kg/hr.	242.295
kgmol/hr	7.60000
Density kg/m ³	1708.81
Viscosity centipoise175474
Thermal cond. cal/cm*s*K517224E-03
Specific heat kJ/kg*K	1.63250
Surface tension dyne/cm.	12.0762
S. G. (60/60)985094
m ³ /hr (15.6 deg C & 1 atm)246347
Vol. flowrate m ³ /hr141794
	Liquid	Liquid
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.296160E-01	.225082
Oxygen	.970384	7.37492

*** Stream no. 36 ***

	All Vapor	
Temperature deg C.	-17.0000
Pressure bars.	1.40055
Enthalpy MJ/hr	206.062
Entropy MJ/hr*K234685
Ave. mol. wt.	28.0407
Total flow kg/hr.	299.812
kgmol/hr	10.6920
Density kg/m ³	1.84668
Viscosity centipoise157820E-01
Thermal cond. cal/cm*s*K571445E-04
Specific heat kJ/kg*K	1.04649
Z factor998569
m ³ /hr (15.6 deg C & 1 atm)	253.296
Vol. flowrate m ³ /hr	162.355
	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.993056	10.6178
Oxygen	.694353E-02	.742405E-01

CODE REPORT

*** Stream no. 37 ***

Temperature deg C.	All Liquid
Pressure bars.	-183.027
Enthalpy MJ/hr	1.59905
Entropy MJ/hr*K	56.7509
Ave. mol. wt.	-619822
Total flow kg/hr.	31.8809
kgmol/hr	242.295
Density kg/m ³	7.60000
Viscosity centipoise	1736.78
Thermal cond. cal/cm*s*K190476
Specific heat kJ/kg*K522893E-03
Surface tension dyne/cm.	1.61428
S. G. (60/60)	12.9412
m ³ /hr (15.6 deg C & 1 atm)985094
Vol. flowrate m ³ /hr246347
Liquid		.139510
mole		Liquid
fraction		flowrate
Nitrogen	.296180E-01	kgmol/hr
Oxygen	.970382	.225097
		7.37490

*** Stream no. 38 ***

Temperature deg C.	All Vapor
Pressure bars.	14.3137
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	215.867
Ave. mol. wt.270798
Total flow kg/hr.	28.0407
kgmol/hr	299.812
Density kg/m ³	10.6920
Viscosity centipoise	1.64442
Thermal cond. cal/cm*s*K172909E-01
Specific heat kJ/kg*K624492E-04
Z factor	1.04284
m ³ /hr (15.6 deg C & 1 atm)999233
Vol. flowrate m ³ /hr	253.296
Vapor		182.323
mole		Vapor
fraction		flowrate
Nitrogen	.993056	kgmol/hr
Oxygen	.694353E-02	10.6178
		.742405E-01

CODE REPORT

*** Stream no. 39 ***

Temperature deg C.	All Vapor
Pressure bars.	-182.235
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	464.271
Ave. mol. wt.	-.297990
Total flow kg/hr.	28.0407
kgmol/hr	908.519
Density kg/m ³	32.4000
Viscosity centipoise	5.40076
Thermal cond. cal/cm*s*K612474E-02
Specific heat kJ/kg*K350410E-04
		1.11342

Z factor961993
m3/hr (15.6 deg C & 1 atm) .	767.562
Vol. flowrate m3/hr	168.223
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .993052	32.1749
Oxygen .694790E-02	.225112

*** Stream no. 40 ***

Temperature deg C.	All Vapor -175.151
Pressure bars.	1.40055
Enthalpy MJ/hr	471.407
Entropy MJ/hr*K	-.222393
Ave. mol. wt.	28.0407
Total flow kg/hr.	908.519
kgmol/hr	32.4000
Density kg/m3	4.97527
Viscosity centipoise660189E-02
Thermal cond. cal/cm*s*K . .	.358104E-04
Specific heat kJ/kg*K . . .	1.10482
Z factor968776
m3/hr (15.6 deg C & 1 atm) .	767.562
Vol. flowrate m3/hr	182.609
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .993052	32.1749
Oxygen .694790E-02	.225112

CODE REPORT

*** Stream no. 41 ***

Temperature deg C.	All Liquid -183.027
Pressure bars.	190.000
Enthalpy MJ/hr	59.0735
Entropy MJ/hr*K	-.632921
Ave. mol. wt.	31.8810
Total flow kg/hr.	242.296
kgmol/hr	7.60001
Density kg/m3	1851.30
Viscosity centipoise190477
Thermal cond. cal/cm*s*K . .	.522893E-03
Specific heat kJ/kg*K . . .	1.53818
Surface tension dyne/cm. . .	12.9162
S. G. (60/60)985095
m3/hr (15.6 deg C & 1 atm) .	246348
Vol. flowrate m3/hr	130880
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen .296015E-01	.224972
Oxygen .970398	7.37504

*** Stream no. 42 ***

Temperature deg C.	All Liquid -180.000
Pressure bars.	190.000
Enthalpy MJ/hr	60.2039

Entropy MJ/hr*K	-620586
Ave. mol. wt.	31.8810
Total flow kg/hr.	242.296
kgmol/hr	7.60001
Density kg/m3	1832.14
Viscosity centipoise177378
Thermal cond. cal/cm*s*K517994E-03
Specific heat kJ/kg*K	1.54479
Surface tension dyne/cm.	12.1671
S. G. (60/60)985095
m3/hr (15.6 deg C & 1 atm)246348
Vol. flowrate m3/hr132249
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen	.296015E-01	.224972
Oxygen	.970398	7.37504

150

CODE REPORT

*** Stream no. 43 ***

Temperature deg C.	All Vapor
Pressure bars.	-175.000
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	471.559
Ave. mol. wt.	-.220847
Total flow kg/hr.	28.0407
kgmol/hr	908.517
Density kg/m3	32.4000
Viscosity centipoise	4.96696
Thermal cond. cal/cm*s*K661203E-02
Specific heat kJ/kg*K358270E-04
Z factor	1.10413
m3/hr (15.6 deg C & 1 atm)968902
Vol. flowrate m3/hr	767.562
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.993056	32.1750
Oxygen	.694356E-02	.224971

*** Stream no. 44 ***

Temperature deg C.	All Vapor
Pressure bars.	-162.695
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	483.838
Ave. mol. wt.	-.102970
Total flow kg/hr.	28.0407
kgmol/hr	908.517
Density kg/m3	32.4000
Viscosity centipoise	4.37578
Thermal cond. cal/cm*s*K743164E-02
Specific heat kJ/kg*K371982E-04
Z factor	1.09313
m3/hr (15.6 deg C & 1 atm)977283
Vol. flowrate m3/hr	767.562
Vapor mole fraction	Vapor flowrate kgmol/hr

Nitrogen .993056 J2.1750
 Oxygen .694356E-02 .224971

CODE REPORT

160

***** Stream no. 45 *****

	All Liquid
Temperature deg C.	-166.695
Pressure bars.	190.000
Enthalpy MJ/hr	65.2489
Entropy MJ/hr*K	-.569975
Ave. mol. wt.	31.8810
Total flow kg/hr.	242.296
kgmol/hr	7.60001
Density kg/m3	1747.94
Viscosity centipoise	.136068
Thermal cond. cal/cm*s*K	.496463E-03
Specific heat kJ/kg*K	1.58880
Surface tension dyne/cm.	8.96505
S. G. (60/60)	.985095
m3/hr (15.6 deg C & 1 atm)	.246348
Vol. flowrate m3/hr	.138620
Liquid	Liquid
mole	flowrate
fraction	kgmol/hr
Nitrogen .296015E-01	.224972
Oxygen .970398	7.37504

***** Stream no. 46 *****

	All Vapor
Temperature deg C.	-18.0000
Pressure bars.	190.000
Enthalpy MJ/hr	129.703
Entropy MJ/hr*K	-.190834
Ave. mol. wt.	31.8810
Total flow kg/hr.	242.296
kgmol/hr	7.60001
Density kg/m3	337.460
Viscosity centipoise	.189521E-01
Thermal cond. cal/cm*s*K	.540524E-04
Specific heat kJ/kg*K	1.41163
Z factor	.846140
m3/hr (15.6 deg C & 1 atm)	180.046
Vol. flowrate m3/hr	.718010
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen .296015E-01	.224972
Oxygen .970398	7.37504

CODE REPORT

***** Stream no. 47 *****

	All Vapor
Temperature deg C.	-15.0000
Pressure bars.	1.40055
Enthalpy MJ/hr	626.332
Entropy MJ/hr*K	.718553
Ave. mol. wt.	28.0407
Total flow kg/hr.	908.517
kgmol/hr	32.4000
Density kg/m3	1.83226

Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr
 Vapor
 mole
 fraction
 Nitrogen .993056
 Oxygen .694356E-02

Vapor
 flowrate
 kgmol/hr
 32.1750
 .224971

*** Stream no. 48 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr
 Vapor
 mole
 fraction
 Nitrogen .993056
 Oxygen .694353E-02

All Vapor
 -15.0000
 1.40055
 206.689
 .237124
 28.0407
 299.812
 10.6920
 1.83227
 .158808E-01
 .574715E-04
 1.04545
 .998621
 253.296
 163.631
 Vapor
 flowrate
 kgmol/hr
 10.6178
 .742405E-01

CODE REPORT

*** Stream no. 49 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr
 Vapor
 mole
 fraction
 Nitrogen .993056
 Oxygen .694353E-02

All Vapor
 12.0000
 1.40055
 215.143
 .268271
 28.0407
 299.812
 10.6920
 1.65783
 .171821E-01
 .620435E-04
 1.04336
 .999193
 253.296
 180.849
 Vapor
 flowrate
 kgmol/hr
 10.6178
 .742405E-01

*** Stream no. 50 ***

Temperature deg C.
 All Vapor
 -15.0000

Pressure bars,	1140055
Enthalpy MJ/hr	419.642
Entropy MJ/hr*K	.481432
Ave. mol. wt.	28.0407
Total flow kg/hr.	608.708
kgmol/hr	21.7080
Density kg/m ³	1.83227
Viscosity centipoise	.158808E-01
Thermal cond. cal/cm*s*K	.574715E-04
Specific heat kJ/kg*K	1.04576
Z factor	.998621
m ³ /hr (15.6 deg C & 1 atm)	514.267
Vol. flowrate m ³ /hr	332.220
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.993056
Oxygen	.694356E-02

162

CODE REPORT

*** Stream no. 51 ***

Temperature deg C.	All Vapor
Pressure bars.	9.97684
Enthalpy MJ/hr	190.000
Entropy MJ/hr*K	138.784
Ave. mol. wt.	-.157037
Total flow kg/hr.	31.8810
kgmol/hr	242.296
Density kg/m ³	7.60001
Viscosity centipoise	284.551
Thermal cond. cal/cm*s*K	.203259E-01
Specific heat kJ/kg*K	.585559E-04
Z factor	1.27814
m ³ /hr (15.6 deg C & 1 atm)	.904312
Vol. flowrate m ³ /hr	180.046
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.296015E-01
Oxygen	.970398

*** Stream no. 52 ***

Temperature deg C.	All Vapor
Pressure bars.	-5.32648
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	635.509
Ave. mol. wt.	.753482
Total flow kg/hr.	28.0407
kgmol/hr	908.517
Density kg/m ³	32.4000
Viscosity centipoise	1.76568
Thermal cond. cal/cm*s*K	.163539E-01
Specific heat kJ/kg*K	.590755E-04
Z factor	1.04497
m ³ /hr (15.6 deg C & 1 atm)	.998852
Vol. flowrate m ³ /hr	767.562
Vapor mole	Vapor flowrate

	fraction	kgmol/hr
Nitrogen	.993056	32.1750
Oxygen	.00694356E-02	.224971

163

CODE REPORT

TOWR # 1 Data file: PLANT.OUT Profile file: link.prf

Column Summary

Stg	Temp C	Pres bars	Net Flow Rates			Duties MJ/hr
			Liquid	Vapor	Feed	
1	-175.9	6.713	18.302			16.000 -163.51
2	-174.8	6.713	18.437	34.301		
3	-174.6	6.714	18.315	34.437		
4	-174.4	6.715	18.186	34.315		
5	-174.1	6.716	18.053	34.186		
6	-173.9	6.717	17.920	34.053		
7	-173.7	6.718	17.788	33.920		
8	-173.5	6.719	17.664	33.788		
9	-173.2	6.720	17.548	33.664		
10	-173.0	6.721	17.441	33.548		
11	-172.9	6.722	17.345	33.441		
12	-172.7	6.723	17.260	33.345		
13	-172.5	6.724	17.188	33.260		
14	-172.4	6.725	17.125	33.188		
15	-172.3	6.726	20.019	33.125	16.000	
16	-172.3	6.727	20.017	20.019		
17	-172.3	6.728	20.013	20.017		
18	-172.3	6.729	20.007	20.013		
19	-172.2	6.731	19.995	20.006		
20	-172.2	6.732	19.977	19.995		
21	-172.2	6.733	19.977	24.000	24.000	

Stream # 25 fed to Stg 15 is 81.760 % Vapor
 Stream # 26 fed to Stg 21 is 83.239 % Vapor

Stream # 27 is Liquid Distillate from Stg 1
 Stream # 28 is Liquid Bottoms from Stg 21

Condenser duty is -163.509 MJ/hr
 Reboiler duty is .000000 MJ/hr
 CODE REPORT

TOWR # 1 Data file: PLANT.OUT Profile file: link.prf
 Tray Liquid Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard	Actual	Actual
			Vol. flow m3/hr	Vol. flow m3/hr	Density kg/m3
1	514.1	28.09	.58	.39	1329.90 .0947
2	519.3	28.17	.59	.39	1324.17 .0934
3	517.4	28.25	.58	.39	1330.77 .0941
4	515.4	28.34	.58	.39	1337.73 .0949
5	513.4	28.44	.58	.38	1344.89 .0957
6	511.3	28.53	.57	.38	1352.12 .0966

7	509.2	28.63	.57	.37	1359.24	.0974
8	507.3	28.72	.56	.37	1366.10	.0983
9	505.5	28.81	.56	.37	1372.55	.0991
10	503.8	28.89	.56	.37	1378.48	.0998
11	502.3	28.96	.55	.36	1383.83	.1005
12	501.0	29.03	.55	.36	1388.54	.1011
13	499.9	29.08	.55	.36	1392.62	.1016
14	498.9	29.13	.55	.36	1396.12	.1021
15	584.0	29.17	.64	.42	1399.05	.1024
16	584.0	29.18	.64	.42	1399.15	.1024
17	584.0	29.18	.64	.42	1399.33	.1025
18	583.9	29.18	.64	.42	1399.65	.1025
19	583.7	29.19	.64	.42	1400.19	.1026
20	583.4	29.20	.64	.42	1401.05	.1027
21	701.4	29.23	.77	.50	1402.45	.1029

Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard	Actual	Actual density kg/m3
			Vol. flow m3/hr	Vol. flow m3/hr	
*** Total Condenser : No Vapor Outlet ***					
1	963.5	28.09	812.6	34.93	.8356
2	968.7	28.13	815.8	35.16	.8364
3	966.8	28.18	812.9	35.14	.8372
4	964.8	28.22	809.9	35.11	.8380
5	962.8	28.27	806.7	35.09	.8388
6	960.7	28.32	803.6	35.06	.8397
7	958.6	28.37	800.4	35.03	.8405
8	956.7	28.42	797.5	35.01	.8413
9	954.9	28.46	794.8	34.98	.8420
10	953.2	28.50	792.2	34.96	.8427
11	951.7	28.54	789.9	34.93	.8432
12	950.4	28.58	787.9	34.91	.8438
13	949.3	28.60	786.2	34.89	.8442
14	948.3	28.63	784.7	34.88	.8446
15	573.1	28.63	474.3	21.08	.8446
16	573.1	28.63	474.2	21.07	.8446
17	573.1	28.63	474.1	21.07	.8446
18	573.0	28.64	474.0	21.06	.8446
19	572.8	28.65	473.7	21.06	.8448
20	572.5	28.66	473.3	21.05	.8449
21					27.1935

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

TOWR # 1 Data file: PLANT.TSZ Profile file: link.prf

Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm)	7.0000
Downcomer area (ft ² or m ²)	.0151
Weir length (in. or cm)	24.0000
Weir height (in. or cm)	3.0000

Liquid	Vapor	Liquid	Vapor	Flooding Se
--------	-------	--------	-------	-------------

Stg	Diameter meter	Vol. flow m3/min	Vol. flow m3/sec	density kg/m3	density kg/m3	velocity m/sec	height meter
1	***Condenser: No tray sizing***						
2	.3051	.01	.01	1324.1680	27.5840	.1673	.0361
3	.3053	.01	.01	1330.7740	27.5504	.1681	.0361
4	.3047	.01	.01	1337.7290	27.5148	.1689	.0361
5	.3041	.01	.01	1344.8930	27.4775	.1696	.0361
6	.3034	.01	.01	1352.1240	27.4397	.1704	.0362
7	.3028	.01	.01	1359.2350	27.4016	.1711	.0362
8	.3022	.01	.01	1366.1010	27.3654	.1718	.0362
9	.3017	.01	.01	1372.5490	27.3307	.1725	.0362
10	.3012	.01	.01	1378.4840	27.2989	.1731	.0362
11	.3007	.01	.01	1383.8270	27.2703	.1736	.0363
12	.3003	.01	.01	1388.5370	27.2452	.1741	.0363
13	.3000	.01	.01	1392.6240	27.2239	.1745	.0363
14	.2997	.01	.01	1396.1170	27.2065	.1749	.0363
15	.3029	.01	.01	1399.0490	27.1921	.1701	.0359
16	.2615	.01	.01	1399.1460	27.1950	.1517	.0359
17	.2615	.01	.01	1399.3340	27.1978	.1517	.0359
18	.2614	.01	.01	1399.6510	27.1997	.1517	.0359
19	.2614	.01	.01	1400.1900	27.2007	.1517	.0359
20	.2613	.01	.01	1401.0490	27.1989	.1518	.0359
21	.2661	.01	.01	1402.4480	27.1939	.1444	.0354
CODE REPORT							

TOWR # 2

Data file: PLANT.OUT

Profile file: pool.prf

Column Summary

Stg	Temp C	Pres bars	Net Flow Rates			Duties MJ/hr
			Liquid	Vapor	Feed	
1	-192.6	1.401	14.924		16.000	32.400
2	-192.6	1.404	14.920	31.324		
3	-192.6	1.408	14.915	31.320		
4	-192.5	1.411	14.907	31.315		
5	-192.5	1.415	14.896	31.307		
6	-192.5	1.418	14.881	31.296		
7	-192.4	1.422	14.861	31.281		
8	-192.3	1.425	14.832	31.261		
9	-192.2	1.429	14.793	31.232		
10	-192.1	1.432	14.742	31.194		
11	-192.0	1.436	14.674	31.142		
12	-191.8	1.440	14.587	31.074		
13	-191.6	1.443	14.478	30.987		
14	-191.3	1.447	14.348	30.878		
15	-191.0	1.450	14.198	30.748		
16	-190.6	1.454	14.037	30.598		
17	-190.2	1.457	34.459	30.437	24.000	
18	-190.2	1.461	34.464	26.859		
19	-190.2	1.464	34.469	26.864		
20	-190.2	1.468	34.474	26.869		
21	-190.2	1.471	34.479	26.874		
22	-190.1	1.475	34.484	26.879		
23	-190.1	1.479	34.489	26.884		
24	-190.1	1.482	34.494	26.889		
25	-190.1	1.486	34.499	26.894		

26	-190.0	1.489	34.504	26.899
27	-190.0	1.493	34.509	26.904
28	-190.0	1.496	34.514	26.909
29	-190.0	1.500	34.519	26.914
30	-189.9	1.503	34.524	26.919
31	-189.9	1.507	34.529	26.924
32	-189.9	1.510	34.534	26.929
33	-189.9	1.514	34.539	26.934
34	-189.8	1.518	34.544	26.939
35	-189.8	1.521	34.549	26.944
36	-189.8	1.525	34.554	26.949
37	-189.8	1.528	34.559	26.954
38	-189.8	1.532	34.564	26.959
39	-189.7	1.535	34.569	26.964
40	-189.7	1.539	34.574	26.969
41	-189.7	1.542	34.578	26.973
42	-189.7	1.546	34.583	26.978
43	-189.6	1.549	34.588	26.983
44	-189.6	1.553	34.592	26.988
45	-189.6	1.557	34.596	26.992
46	-189.6	1.560	34.599	26.996
47	-189.5	1.564	34.602	26.999

166

CODE REPORT

	Liquid	Vapor	Feed	Product
48 -189.5	1.567	34.600	27.002	
49 -189.5	1.571	34.589	27.000	
50 -189.4	1.574	34.553	26.989	
51 -189.3	1.578	34.451	26.953	
52 -189.0	1.581	34.183	26.851	
53 -188.3	1.585	33.566	26.583	
54 -186.7	1.588	32.488	25.966	
55 -184.1	1.592	31.382	24.888	
56 -181.2	1.596	30.829	23.782	
57 -179.5	1.599	23.229		7.6000
				153.21

Stream # 32 fed to Stg 1 is 6.690 % Vapor
 Stream # 33 fed to Stg 17 is 14.889 % Vapor

Stream # 34 is Vapor Distillate from Stg 1
 Stream # 35 is Liquid Bottoms from Stg 57

Condenser duty is .000000 MJ/hr
 Reboiler duty is 153.215 MJ/hr
 CODE REPORT

TOWER # 2 Data file: PLANT.OUT Profile file: pool.prf
 Tray Liquid Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard	Actual Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Viscosity centipoise	Surface tension dyne/cm
			Vol. flow m3/hr					
1	419.3	28.09	.48	.28	1504.02	.1477	8.32	
2	419.2	28.10	.47	.28	1504.13	.1477	8.32	
3	419.2	28.11	.47	.28	1504.38	.1476	8.32	
4	419.1	28.11	.47	.28	1504.79	.1476	8.33	
5	419.0	28.12	.47	.28	1505.46	.1477	8.34	
6	418.8	28.14	.47	.28	1506.44	.1477	8.36	

7	418.5	28.16	.47	.28	1507.86	.1479	8.38
8	418.1	28.19	.47	.28	1509.88	.1482	8.42
9	417.6	28.23	.47	.28	1512.68	.1485	8.47
10	417.0	28.28	.47	.27	1516.48	.1491	8.53
11	416.1	28.35	.47	.27	1521.53	.1499	8.63
12	414.9	28.45	.47	.27	1528.10	.1509	8.75
13	413.5	28.56	.46	.27	1536.34	.1522	8.90
14	411.8	28.70	.46	.27	1546.31	.1539	9.08
15	409.8	28.86	.45	.26	1557.79	.1558	9.30
16	407.7	29.04	.45	.26	1570.25	.1580	9.54
17	1007.2	29.23	1.10	.64	1582.90	.1602	9.78
18	1007.3	29.23	1.10	.64	1582.70	.1601	9.78
19	1007.5	29.23	1.10	.64	1582.51	.1600	9.77
20	1007.6	29.23	1.10	.64	1582.31	.1598	9.77
21	1007.8	29.23	1.10	.64	1582.10	.1597	9.76
22	1007.9	29.23	1.10	.64	1581.91	.1596	9.76
23	1008.1	29.23	1.10	.64	1581.71	.1595	9.75
24	1008.2	29.23	1.10	.64	1581.51	.1594	9.75
25	1008.4	29.23	1.10	.64	1581.31	.1593	9.74
26	1008.5	29.23	1.10	.64	1581.12	.1592	9.74
27	1008.7	29.23	1.10	.64	1580.92	.1591	9.73
28	1008.8	29.23	1.10	.64	1580.72	.1590	9.72
29	1009.0	29.23	1.10	.64	1580.53	.1589	9.72
30	1009.1	29.23	1.10	.64	1580.33	.1588	9.71
31	1009.3	29.23	1.11	.64	1580.14	.1587	9.71
32	1009.4	29.23	1.11	.64	1579.94	.1586	9.70
33	1009.6	29.23	1.11	.64	1579.75	.1584	9.70
34	1009.7	29.23	1.11	.64	1579.55	.1583	9.69
35	1009.9	29.23	1.11	.64	1579.36	.1582	9.69
36	1010.0	29.23	1.11	.64	1579.16	.1581	9.68
37	1010.2	29.23	1.11	.64	1578.97	.1580	9.68
38	1010.3	29.23	1.11	.64	1578.77	.1579	9.67
39	1010.5	29.23	1.11	.64	1578.58	.1578	9.67
40	1010.6	29.23	1.11	.64	1578.38	.1577	9.66
41	1010.8	29.23	1.11	.64	1578.19	.1576	9.66
42	1010.9	29.23	1.11	.64	1578.00	.1575	9.65
43	1011.0	29.23	1.11	.64	1577.81	.1574	9.65
44	1011.2	29.23	1.11	.64	1577.62	.1573	9.64
45	1011.3	29.23	1.11	.64	1577.44	.1572	9.63
46	1011.4	29.23	1.11	.64	1577.28	.1571	9.63
47	1011.5	29.23	1.11	.64	1577.15	.1570	9.63
48	1011.6	29.24	1.11	.64	1577.13	.1569	9.62
49	1011.5	29.24	1.11	.64	1577.43	.1569	9.63

CODE REPORT

TOWR # 2 Data file: PLANT.OUT
Tray Liquid Properties

Profile file: pool.prf

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Viscosity centipoise	Surface tension dyne/cm
50	1011.1	29.26	1.11	.64	1578.51	.1571	9.65
51	1009.8	29.31	1.10	.64	1581.73	.1576	9.71
52	1006.4	29.44	1.10	.63	1590.22	.1591	9.87
53	998.6	29.75	1.08	.62	1610.17	.1627	10.26
54	986.0	30.35	1.05	.60	1646.33	.1692	10.98
55	976.1	31.10	1.01	.58	1684.68	.1753	11.72
56	975.4	31.64	1.00	.57	1703.76	.1764	12.04

57 242.3 31.88 .25 .14 1708.78 .1755 12.08
 CODE REPORT

TOWER # 2 Data file: PLANT.OUT Profile file: pool.prf
 Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual density kg/m3
				Z	
1	908.5	28.04	767.6	146.77	6.1900
2	878.4	28.04	742.1	141.58	6.2042
3	878.3	28.04	742.0	141.25	6.2183
4	878.3	28.05	741.8	140.93	6.2322
5	878.2	28.05	741.7	140.60	6.2461
6	878.1	28.06	741.4	140.28	6.2595
7	877.9	28.06	741.1	139.95	6.2726
8	877.6	28.07	740.6	139.63	6.2850
9	877.2	28.09	739.9	139.31	6.2969
10	876.7	28.11	739.0	139.00	6.3077
11	876.1	28.13	737.8	138.68	6.3174
12	875.2	28.16	736.2	138.36	6.3256
13	874.0	28.21	734.1	138.03	6.3322
14	872.6	28.26	731.5	137.70	6.3369
15	870.9	28.32	728.4	137.36	6.3399
16	868.9	28.40	724.9	137.02	6.3416
17	866.8	28.48	721.1	136.66	6.3427
18	764.9	28.48	636.3	120.33	6.3569
19	765.0	28.48	636.4	120.08	6.3712
20	765.2	28.48	636.5	119.83	6.3855
21	765.3	28.48	636.7	119.59	6.3997
22	765.5	28.48	636.8	119.35	6.4141
23	765.6	28.48	636.9	119.10	6.4283
24	765.8	28.48	637.0	118.86	6.4426
25	765.9	28.48	637.1	118.62	6.4568
26	766.1	28.48	637.2	118.39	6.4712
27	766.2	28.48	637.4	118.15	6.4854
28	766.4	28.48	637.5	117.91	6.4997
29	766.5	28.48	637.6	117.68	6.5139
30	766.7	28.48	637.7	117.44	6.5282
31	766.8	28.48	637.8	117.21	6.5425
32	767.0	28.48	638.0	116.98	6.5568
33	767.1	28.48	638.1	116.74	6.5710
34	767.3	28.48	638.2	116.51	6.5853
35	767.4	28.48	638.3	116.28	6.5995
36	767.6	28.48	638.4	116.06	6.6138
37	767.7	28.48	638.5	115.83	6.6280
38	767.9	28.48	638.7	115.60	6.6423
39	768.0	28.48	638.8	115.38	6.6565
40	768.2	28.48	638.9	115.16	6.6707
41	768.3	28.48	639.0	114.93	6.6850
42	768.5	28.48	639.1	114.71	6.6992
43	768.6	28.48	639.2	114.49	6.7135
44	768.8	28.49	639.3	114.27	6.7277
45	768.9	28.49	639.4	114.04	6.7419
46	769.0	28.49	639.5	113.82	6.7561
47	769.1	28.49	639.6	113.60	6.7703
48	769.2	28.49	639.7	113.39	6.7842
49	769.3	28.49	639.6	113.16	6.7980

TOWER # 2 Data file: PLANT.OUT Profile file: pool.prf
 Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Z	Actual density kg/m3
50	769.2	28.50	639.4	112.94	.9463	6.8107
51	768.8	28.52	638.5	112.71	.9464	6.8211
52	767.5	28.58	636.1	112.45	.9467	6.8252
53	764.1	28.74	629.8	112.12	.9475	6.8151
54	756.3	29.13	615.1	111.52	.9494	6.7821
55	743.7	29.88	589.6	110.28	.9524	6.7435
56	733.8	30.86	563.4	108.76	.9549	6.7468
57	733.1	31.56	550.3	108.14	.9562	6.7793

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986
 All Rights Reserved.

TOWER # 2 Data file: PLANT.TSZ Profile file: pool.prf

Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm)	7.0000
Downcomer area (ft ² or m ²)	.0236
Weir length (in. or cm)	30.0000
Weir height (in. or cm)	3.0000

Stg	Diameter meter	Liquid Vol. flow m3/min	Vapor Vol. flow m3/sec	Liquid density kg/m3	Vapor density kg/m3	Flooding velocity m/sec	Settling height meter
1	.3980	.00	.04	1504.0210	6.1901	.4043	.0373
2	.3922	.00	.04	1504.1320	6.2043	.4046	.0373
3	.3920	.00	.04	1504.3760	6.2184	.4042	.0373
4	.3918	.00	.04	1504.7910	6.2323	.4038	.0373
5	.3915	.00	.04	1505.4570	6.2462	.4035	.0373
6	.3913	.00	.04	1506.4410	6.2596	.4032	.0373
7	.3910	.00	.04	1507.8640	6.2727	.4029	.0373
8	.3907	.00	.04	1509.8790	6.2851	.4028	.0373
9	.3904	.00	.04	1512.6780	6.2970	.4028	.0373
10	.3899	.00	.04	1516.4810	6.3078	.4029	.0373
11	.3894	.00	.04	1521.5340	6.3175	.4033	.0373
12	.3889	.00	.04	1528.0950	6.3257	.4038	.0373
13	.3882	.00	.04	1536.3390	6.3323	.4046	.0374
14	.3874	.00	.04	1546.3100	6.3370	.4057	.0374
15	.3865	.00	.04	1557.7890	6.3400	.4070	.0374
16	.3856	.00	.04	1570.2530	6.3417	.4085	.0374
17	.3935	.01	.04	1582.9040	6.3428	.3874	.0353
18	.3775	.01	.03	1582.7020	6.3570	.3785	.0353
19	.3774	.01	.03	1582.5090	6.3713	.3780	.0353
20	.3773	.01	.03	1582.3070	6.3856	.3775	.0353
21	.3772	.01	.03	1582.1040	6.3998	.3769	.0353

22	.3771	.01	.03	1581.9110	6.4142	.3764	.0353
23	.3770	.01	.03	1581.7080	6.4284	.3759	.0353
24	.3769	.01	.03	1581.5140	6.4427	.3754	.0353
25	.3768	.01	.03	1581.3120	6.4569	.3748	.0353
26	.3767	.01	.03	1581.1170	6.4713	.3743	.0353
27	.3766	.01	.03	1580.9240	6.4855	.3738	.0353
28	.3765	.01	.03	1580.7210	6.4998	.3733	.0353

CODE REPORT

Smith-Dresser-Ohlswager Shortcut Technique

Stg	Diameter meter	Liquid Vol. flow m3/min	Vapor Vol. flow m3/sec	Liquid density kg/m3	Vapor density kg/m3	Flooding velocity m/sec	Settling height meter
29	.3764	.01	.03	1580.5260	6.5140	.3728	.0353
30	.3763	.01	.03	1580.3320	6.5283	.3723	.0353
31	.3762	.01	.03	1580.1370	6.5426	.3718	.0353
32	.3761	.01	.03	1579.9430	6.5569	.3713	.0353
33	.3761	.01	.03	1579.7490	6.5711	.3708	.0353
34	.3760	.01	.03	1579.5550	6.5854	.3703	.0353
35	.3759	.01	.03	1579.3600	6.5996	.3697	.0353
36	.3758	.01	.03	1579.1650	6.6139	.3692	.0353
37	.3757	.01	.03	1578.9690	6.6281	.3687	.0353
38	.3756	.01	.03	1578.7750	6.6423	.3683	.0353
39	.3755	.01	.03	1578.5800	6.6566	.3678	.0353
40	.3754	.01	.03	1578.3850	6.6708	.3673	.0353
41	.3753	.01	.03	1578.1900	6.6851	.3668	.0353
42	.3752	.01	.03	1578.0040	6.6993	.3663	.0353
43	.3752	.01	.03	1577.8130	6.7136	.3658	.0353
44	.3751	.01	.03	1577.6250	6.7278	.3653	.0353
45	.3750	.01	.03	1577.4420	6.7420	.3648	.0353
46	.3749	.01	.03	1577.2790	6.7562	.3643	.0353
47	.3748	.01	.03	1577.1480	6.7704	.3639	.0353
48	.3747	.01	.03	1577.1340	6.7843	.3634	.0353
49	.3746	.01	.03	1577.4270	6.7981	.3630	.0353
50	.3744	.01	.03	1578.5150	6.8108	.3628	.0353
51	.3740	.01	.03	1581.7290	6.8212	.3630	.0353
52	.3732	.01	.03	1590.2230	6.8253	.3642	.0353
53	.3714	.01	.03	1610.1710	6.8152	.3675	.0354
54	.3682	.01	.03	1646.3300	6.7822	.3738	.0355
55	.3643	.01	.03	1684.6800	6.7436	.3799	.0356
56	.3618	.01	.03	1703.7650	6.7469	.3814	.0356

57 ***Reboiler: No tray sizing***

Appendix C

Computer results of the optimized exchanger network of the oxygen plant

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986
 All Rights Reserved.

TOPOLOGY

Equipment	Stream Numbers
1 HXER	5 16 -7 -17
2 HXER	4 15 -6 -18
3 HXER	9 14 -12 -15
4 HXER	19 -3 0 0
5 DVDR	3 -5 -4 0
6 DVDR	8 -9 -10 0
7 MIXR	7 6 -8 0
8 ENTM	10 -11 0 0
9 VALV	12 -13 0 0
10 HXER	1 17 -2 -20
11 HXER	2 18 -19 -21

Stream Connections

Stream	Equipment	From	To
1		0	10
2		10	11
3		4	5
4		5	2
5		5	1
6		2	7
7		1	7
8		7	6
9		6	3
10		6	8
11		8	0
12		3	9
13		9	0
14		0	3
15		3	2
16		0	1
17		1	10
18		2	11
19		11	4
20		10	0
21		11	0

COMPONENTS 2

ID numbers 46, 47,

THERMODYNAMICS

Kvalue option: Peng-Robinson
 Enthalpy option: Peng-Robinson
 Density option: API method
 CODE REPORT

MISCELLANEOUS

173

Recycle calculations are converged.

Recycle equipment list (KE2): 4. 5. 1. 10. 6. 3. 2. 3. 1.

Preferred cut stream list (KE3): 19, 8,

Convergence tolerances,	Error
Flowrates:	.00100000
Vapor fraction:	.00100000
Temperature:	.00100000
Pressure:	.00100000
Enthalpy:	.00100000
Flash calcs:	.00005000

Max. loops in recycle calc.: 30
in flash calcs: 75

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved

All Rights Reserved
© 2012 BUNNERS

*** DIVIDERS ***

Equipment no.		5	6
External name			
Fraction/flow	1	.33200	.40000
	2	.66800	.60000
	3	.00000	.00000
	4	.00000	.00000
	5	.00000	.00000
	6	.00000	.00000

*** MIXERS-W/FLASH ***
Equipment no. 7
Customer

*** VALVES ***
Equipment no. 9
External name
Outlet pressure bars 6.6132

EXCHANGER/CONDENSERS

Equipment no.	1	2	3	4	
External name					174
Heat transfer coeff.	300.00	300.00	300.00	.00000	
Area	6.6210	20.686	1.8796	.00000	
Number of shells	.0	.0	.0	.0	
Shell passes	.0	.0	.0	.0	
Tube passes	.0	.0	.0	.0	
Mode	3.0	3.0	3.0	5.0	
Min. delta T or T-out	24.00	19.00	28.00	12.00	
Delta P, stream 1	.00000	.00000	.00000	.00000	
Delta P, stream 2	.00000	.00000	.00000	.00000	
Q, stream 1 MJ/hr	-71.526	-144.00	-18.371	2.9531	
Water usage, DM3/hr	.00000	.00000	.00000	.00000	
Corrected delta T	36.01	23.20	32.58	.00	
Equipment no.	10	11			
External name					
Heat transfer coeff.	300.00	300.00			
Area	1.2357	1.8388			
Number of shells	.0	.0			
Shell passes	.0	.0			
Tube passes	.0	.0			
Mode	3.0	3.0			
Min. delta T or T-out	12.00	14.40			
Delta P, stream 1	.00000	.00000			
Delta P, stream 2	.00000	.00000			
Q, stream 1 MJ/hr	-7.0601	-8.5576			
Water usage, DM3/hr	.00000	.00000			
Corrected delta T	19.04	15.51			

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986

All Rights Reserved.

*** ENTROPY MACHINES ***

Equipment no.	B
External name	
Mode	.0
Outlet pressure bars	6.6065
Adiabatic efficiency	.92000
Theoretical hp	-10.158
Actual horsepower	-9.3453
Entropy in KJ/deg C	-760.17
Entropy out, ideal	-760.68
Entropy out, actual	-739.87

CODE REPORT

*** Stream no. 1 ***

	All Vapor
Temperature deg C	22.0000
Pressure bars	44.0000
Enthalpy MJ/hr	802.024
Entropy MJ/hr*K	-.142020
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	52.5200

Viscosity centipoise 162249E-01
 Thermal cond. cal/cm*s*K 631572E-04
 Specific heat kJ/kg*K 1.09868
 Z factor 0.982210
 m3/hr (15.6 deg C & 1 atm) 947.608
 Vol. flowrate m3/hr 21.9122

	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.810000	32.4000
Oxygen	.190000	7.60000

*** Stream no. 2 ***

Temperature deg C. 16.4189
 Pressure bars. 44.0000
 Enthalpy MJ/hr 794.964
 Entropy MJ/hr*K -.166208
 Ave. mol. wt. 28.7703
 Total flow kg/hr. 1150.81
 kgmol/hr 40.0000
 Density kg/m3 53.6688 .
 Viscosity centipoise 179587E-01
 Thermal cond. cal/cm*s*K 621728E-04
 Specific heat kJ/kg*K 1.10302
 Z factor 0.979712
 m3/hr (15.6 deg C & 1 atm) 947.608
 Vol. flowrate m3/hr 21.4432

	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.810000	32.4000
Oxygen	.190000	7.60000

CODE REPORT

*** Stream no. 3 ***

Temperature deg C. 12.0000
 Pressure bars. 44.0000
 Enthalpy MJ/hr 789.335
 Entropy MJ/hr*K -.165758
 Ave. mol. wt. 28.7703 .
 Total flow kg/hr. 1150.81
 kgmol/hr 40.0000
 Density kg/m3 54.6189
 Viscosity centipoise 177463E-01
 Thermal cond. cal/cm*s*K 614027E-04
 Specific heat kJ/kg*K 1.10628
 Z factor 0.977588
 m3/hr (15.6 deg C & 1 atm) 947.608
 Vol. flowrate m3/hr 21.0702

	Vapor	Vapor
	mole	flowrate
	fraction	kgmol/hr
Nitrogen	.810000	32.4000
Oxygen	.190000	7.60000

*** Stream no. 4 ***

	All Vapor
Temperature deg C.	12.0000

Pressure bars.	44.0000
Enthalpy MJ/hr	527.276
Entropy MJ/hr*K	-.124086
Ave. mol. wt.	28.7703
Total flow kg/hr.	768.743
kgmol/hr	26.7200
Density kg/m3	54.6189
Viscosity centipoise177463E-01
Thermal cond. cal/cm*s*K614027E-04
Specific heat kJ/kg*K	1.10651
Z factor977588
m3/hr (15.6 deg C & 1 atm)	633.002
Vol. flowrate m3/hr	14.0749
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .810000	21.6432
Oxygen .190000	5.07680

176

CODE REPORT

*** Stream no. 5 ***

Temperature deg C.	All Vapor
Pressure bars.	12.0000
Enthalpy MJ/hr	44.0000
Entropy MJ/hr*K	262.059
Ave. mol. wt.	-.616716E-01
Total flow kg/hr.	28.7703
kgmol/hr	382.070
Density kg/m3	13.2800
Viscosity centipoise	54.6189
Thermal cond. cal/cm*s*K177463E-01
Specific heat kJ/kg*K614027E-04
Z factor	1.10704
m3/hr (15.6 deg C & 1 atm)977588
Vol. flowrate m3/hr	314.606
Vapor mole fraction	6.99530
Nitrogen .810000	Vapor flowrate kgmol/hr
Oxygen .190000	10.7568

*** Stream no. 6 ***

Temperature deg C.	All Vapor
Pressure bars.	-128.560
Enthalpy MJ/hr	44.0000
Entropy MJ/hr*K	383.278
Ave. mol. wt.	-.846519
Total flow kg/hr.	28.7703
kgmol/hr	768.743
Density kg/m3	26.7200
Viscosity centipoise	177.787
Thermal cond. cal/cm*s*K105647E-01
Specific heat kJ/kg*K407851E-04
Z factor	2.63376
m3/hr (15.6 deg C & 1 atm)592288
Vol. flowrate m3/hr	633.002
Vapor mole	Vapor flowrate

	fraction	kgmol/hr
Nitrogen	.810000	21,6432
Oxygen	.190000	5.07680

CODE REPORT

177

*** Stream no. 7 ***

	All Vapor	
Temperature deg C.	-128.519	
Pressure bars.	44.0000	
Enthalpy MJ/hr	190.533	
Entropy MJ/hr*K	-.420439	
Ave. mol. wt.	28.7703	
Total flow kg/hr.	382.070	
kgmol/hr	13.2800	
Density kg/m3	177.565	
Viscosity centipoise	.105659E-01	
\bar{v} Thermal cond. cal/cm*s*K	.407901E-04	
Specific heat kJ/kg*K	2.62877	
Z factor	.592862	
m3/hr (15.6 deg C & 1 atm)	314.606	
Vol. flowrate m3/hr	2.15175	
Vapor	Vapor	
mole	flowrate	
fraction	kgmol/hr	
Nitrogen	.810000	10.7568
Oxygen	.190000	2.52320

*** Stream no. 8 ***

	All Vapor	
Temperature deg C.	-128.546	
Pressure bars.	44.0000	
Enthalpy MJ/hr	573.811	
Entropy MJ/hr*K	-1.26695	
Ave. mol. wt.	28.7703	
Total flow kg/hr.	1150.81	
kgmol/hr	40.0000	
Density kg/m3	177.711	
Viscosity centipoise	.105651E-01	
Thermal cond. cal/cm*s*K	.407868E-04	
Specific heat kJ/kg*K	2.63183	
Z factor	.592484	
m3/hr (15.6 deg C & 1 atm)	947.608	
Vol. flowrate m3/hr	6.47584	
Vapor	Vapor	
mole	flowrate	
fraction	kgmol/hr	
Nitrogen	.810000	32.4000
Oxygen	.190000	7.60000

CODE REPORT

*** Stream no. 9 ***

	All Vapor
Temperature deg C.	-128.546
Pressure bars.	44.0000
Enthalpy MJ/hr	229.525
Entropy MJ/hr*K	-.506780
Ave. mol. wt.	28.7703
Total flow kg/hr.	460.325
kgmol/hr	16.0000
Density kg/m3	177.711

Viscosity centipoise105651E-01
Thermal cond. cal/cm*s*K . .	.407868E-04
Specific heat kJ/kg*K . .	2.63231
Z factor592484
m3/hr (15.6 deg C & 1 atm) .	379.043
Vol. flowrate m3/hr	2.59034
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen	12.9600
Oxygen	3.04000

178

*** Stream no. 10 ***

Temperature deg C.	All Vapor
Pressure bars.	-128.546
Enthalpy MJ/hr	44.0000
Entropy MJ/hr*K	344.287
Ave. mol. wt.	-.760170
Total flow kg/hr.	28.7703
kgmol/hr	690.488
Density kg/m3	24.0000
Viscosity centipoise	177.711
Thermal cond. cal/cm*s*K . .	.105651E-01
Specific heat kJ/kg*K407868E-04
Z factor	2.63220
m3/hr (15.6 deg C & 1 atm) .	.592484
Vol. flowrate m3/hr	568.565
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen	19.4400
Oxygen	4.56000

CODE REPORT

*** Stream no. 11 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.433		
Pressure bars.	6.60650		
Vapor fraction794075		
Enthalpy MJ/hr	319.118	273.333	45.7936
Entropy MJ/hr*K	-.739788	-.392903	-.346885
Ave. mol. wt.	28.7703	28.6535	29.2210
Total flow kg/hr	690.488	546.073	144.416
kgmol/hr	24.0000	19.0578	4.94220
Density kg/m3		26.6876	1405.25
Viscosity centipoise689482E-02	.103319
Thermal cond. cal/cm*s*K . .		.357702E-04	.505124E-03
Specific heat kJ/kg*K		1.25212	2.17687
Z factor846983	
m3/hr (15.6 deg C & 1 atm) .		451.521	
Vol. flowrate m3/hr		20.4619	
Surface tension dyne/cm.			5.65960
S. G. (60/60)914499
m3/hr (15.6 deg C & 1 atm) .			.158166
Vol. flowrate m3/hr102770
Vapor	Liquid	Vapor	Liquid
mole	mole	flowrate	flowrate
fraction	fraction	kgmol/hr	kgmol/hr
Nitrogen83931	.69694	15.9954
			3.44440

Oxygen	.16069	.30306	3.06243	1.49780	179
--------	--------	--------	---------	---------	-----

*** Stream no. 12 ***

	All Vapor	
Temperature deg C.	-137.366	
Pressure bars.	44.0000	
Enthalpy MJ/hr	211.154	
Entropy MJ/hr*K	-.638783	
Ave. mol. wt.	28.7703	
Total flow kg/hr.	460.325	
kgmol/hr	16.0000	
Density kg/m3	297.981	
Viscosity centipoise	.108017E-01	
Thermal cond. cal/cm*s*K	.397190E-04	
Specific heat kJ/kg*K	9.94371	
Z factor	.376301	
m3/hr (15.6 deg C & 1 atm)	379.043	
Vol. flowrate m3/hr	1.54484	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen	.810000	12.9600
Oxygen	.190000	3.04000

CODE REPORT

*** Stream no. 13 ***

	Overall	Vapor	Liquid	
Temperature deg C.	-172.462			
Pressure bars.	6.61320			
Vapor fraction	.774389			
Enthalpy MJ/hr	211.154	177.671	33.4807	
Entropy MJ/hr*K	-.509091	-.255681	-.253410	
Ave. mol. wt.	28.7703	28.6438	29.2045	
Total flow kg/hr	460.325	354.904	105.422	
kgmol/hr	16.0000	12.3902	3.60978	
Density kg/m3	26.7247		1403.92	
Viscosity centipoise	.689110E-02		.103236	
Thermal cond. cal/cm*s*K	.357706E-04		.505164E-03	
Specific heat kJ/kg*K	1.25342		2.18061	
Z factor	.846719			
m3/hr (15.6 deg C & 1 atm)	293.518			
Vol. flowrate m3/hr	13.2802			
Surface tension dyne/cm.			5.64067	
S. G. (60/60)			.914055	
m3/hr (15.6 deg C & 1 atm)			.115515	
Vol. flowrate m3/hr			.750916E-01	
Vapor mole fraction	Vapor mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr	
Nitrogen	.84174	.70109	10.4293	2.53077
Oxygen	.15826	.29891	1.96094	1.07901

*** Stream no. 14 ***

	All Vapor
Temperature deg C.	-175.000
Pressure bars.	1.40055
Enthalpy MJ/hr	471.559
Entropy MJ/hr*K	-.220848
Ave. mol. wt.	28.0407
Total flow kg/hr.	908.518

kgmol/hr	32.4000	
Density kg/m ³	4.96697	180
Viscosity centipoise661202E-02	
Thermal cond. cal/cm*s*K358270E-04	
Specific heat kJ/kg*K	1.10482	
Z factor968902	
m ³ /hr (15.6 deg C & 1 atm)	767.562	
Vol. flowrate m ³ /hr	182.915	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen993061 32.1752	
Oxygen693858E-02 .224810	
CODE REPORT		
*** Stream no. 15 ***		
Temperature deg C.	All Vapor -156.546	
Pressure bars.	1.40055	
Enthalpy MJ/hr	489.930	
Entropy MJ/hr*K	-.492974E-01	
Ave. mol. wt.	28.0407	
Total flow kg/hr.	908.518	
kgmol/hr	32.4000	
Density kg/m ³	4.13194	
Viscosity centipoise783564E-02	
Thermal cond. cal/cm*s*K378999E-04	
Specific heat kJ/kg*K	1.08797	
Z factor980378	
m ³ /hr (15.6 deg C & 1 atm)	767.562	
Vol. flowrate m ³ /hr	219.880	
Vapor mole fraction	Vapor flowrate kgmol/hr	
Nitrogen993061 32.1752	
Oxygen693858E-02 .224810	
*** Stream no. 16 ***		
Temperature deg C.	All Liquid -180.000	
Pressure bars.	190.000	
Enthalpy MJ/hr	60.2035	
Entropy MJ/hr*K	-.620586	
Ave. mol. wt.	31.8811	
Total flow kg/hr.	242.296	
kgmol/hr	7.60000	
Density kg/m ³	1832.15	
Viscosity centipoise177380	
Thermal cond. cal/cm*s*K517994E-03	
Specific heat kJ/kg*K	1.54479	
Surface tension dyne/cm.	12.1673	
S. G. (60/60)985098	
m ³ /hr (15.6 deg C & 1 atm)246347	
Vol. flowrate m ³ /hr132248	
Liquid mole fraction	Liquid flowrate kgmol/hr	
Nitrogen295803E-01 .224810	
Oxygen970420 7.37519	
CODE REPORT		

*** Stream no. 17 ***

	All Vapor
Temperature deg C.	-12.0000
Pressure bars.	190.000
Enthalpy MJ/hr	131.730
Entropy MJ/hr*K182981
Ave. mol. wt.	31.6811
Total flow kg/hr.	242.296
kgmol/hr	7.60000
Density kg/m3	324.183
Viscosity centipoise192427E-01
Thermal cond. cal/cm*s*K549925E-04
Specific heat kJ/kg*K	1.37680
Z factor860560
m3/hr (15.6 deg C & 1 atm)	180.045
Vol. flowrate m3/hr747417
	Vapor
	mole
	fraction
Nitrogen295803E-01
Oxygen970420
	flowrate
	kgmol/hr

181

*** Stream no. 18 ***

	All Vapor
Temperature deg C.	-7.00000
Pressure bars.	1.40055
Enthalpy MJ/hr	633.928
Entropy MJ/hr*K747534
Ave. mol. wt.	28.0407
Total flow kg/hr.	908.518
kgmol/hr	32.4000
Density kg/m3	1.77685
Viscosity centipoise162726E-01
Thermal cond. cal/cm*s*K587952E-04
Specific heat kJ/kg*K	1.04497
Z factor978815
m3/hr (15.6 deg C & 1 atm)	767.562
Vol. flowrate m3/hr511.316
	Vapor
	mole
	fraction
Nitrogen993061
Oxygen693858E-02
	flowrate
	kgmol/hr

CODE REPORT

*** Stream no. 19 ***

	All Vapor
Temperature deg C.	9.68271
Pressure bars.	44.0000
Enthalpy MJ/hr	786.406
Entropy MJ/hr*K196156
Ave. mol. wt.	28.7703
Total flow kg/hr.	1150.81
kgmol/hr	40.0000
Density kg/m3	55.1324
Viscosity centipoise176345E-01
Thermal cond. cal/cm*s*K610021E-04
Specific heat kJ/kg*K	1.10737
Z factor976418

m3/hr (15.6 deg C & 1 atm) .	947.608	
Vol. flowrate MJ/hr	20.0731	
	Vapor	
	mole	Vapor
	fraction	flowrate
Nitrogen810000	kgmol/hr
Oxygen190000	32.4000
	7.60000	

182

*** Stream no. 20 ***

Temperature deg C.	All Vapor	
Pressure bars.	10.0000	
Enthalpy MJ/hr	190.000	
Entropy MJ/hr*K	138.790	
Ave. mol. wt.	-1.157012	
Total flow kg/hr.	31.8811	
kgmol/hr	242.296	
Density kg/m3	7.60000	
Viscosity centipoise	284.517	
Thermal cond. cal/cm*s*k203271E-01	
Specific heat kJ/kg*K585597E-04	
Z factor	1.27878	
m3/hr (15.6 deg C & 1 atm) .	.904349	
Vol. flowrate m3/hr	180.045	
	Vapor	
	mole	Vapor
	fraction	flowrate
Nitrogen295803E-01	kgmol/hr
Oxygen970420	.224810
	7.37519	

CODE REPORT

*** Stream no. 21 ***

Temperature deg C.	All Vapor	
Pressure bars.	2.01687	
Enthalpy MJ/hr	1.40055	
Entropy MJ/hr*K	642.486	
Ave. mol. wt.779154	
Total flow kg/hr.	28.0407	
kgmol/hr	908.518	
Density kg/m3	32.4000	
Viscosity centipoise	1.71828	
Thermal cond. cal/cm*s*k167079E-01	
Specific heat kJ/kg*K693186E-04	
Z factor	1.04360	
m3/hr (15.6 deg C & 1 atm) .	.999007	
Vol. flowrate m3/hr	767.562	
	Vapor	
	mole	Vapor
	fraction	flowrate
Nitrogen993061	kgmol/hr
Oxygen693858E-02	.32.1752
	.224810	

Appendix D

Computer results of the column-tray sizing under the nominal flow rate of air

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

TOPOLOGY

Equipment	Stream Numbers
1 TOWR	1 2 -3 -4
2 HXER	3 10 -5 -13
3 HXER	4 13 -6 -15
4 HXER	8 11 -9 -12
5 VALV	5 -7 0 0
6 VALV	6 -8 0 0
7 TOWR	7 9 -10 -11
8 PUMP	12 -14 0 0
9 HXER	15 -16 0 0
10 HXER	14 -17 0 0

Stream Connections

Stream Equipment

From	To
1	0
2	0
3	1
4	1
5	2
6	3
7	5
8	6
9	4
10	7
11	7
12	4
13	2
14	8
15	3
16	9
17	10

COMPONENTS 2
ID numbers 46, 47,

THERMODYNAMICS

Kvalue option: Peng-Robinson
Enthalpy option: Peng-Robinson
Density option: API method
CODE REPORT

MISCELLANEOUS

Recycle calculations are converged.

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986
 All Rights Reserved.

TOPOLOGY

Equipment		Stream Numbers			
1	TOWR	1	2	-3	-4
2	HXER	3	10	-5	-13
3	HXER	4	13	-6	-15
4	HXER	8	11	-9	-12
5	VALV	5	-7	0	0
6	VALV	6	-8	0	0
7	TOWR	7	9	-10	-11
8	PUMP	12	-14	0	0
9	HXER	15	-16	0	0
10	HXER	14	-17	0	0

Stream Connections

Stream	Equipment	From	To
1		0	1
2		0	1
3		1	2
4		1	3
5		2	5
6		3	6
7		5	7
8		6	4
9		4	7
10		7	2
11		7	4
12		4	8
13		2	3
14		8	10
15		3	9
16		9	0
17		10	0

COMPONENTS 2
 ID numbers 46, 47,

THERMODYNAMICS

Kvalue option: Peng-Robinson
 Enthalpy option: Peng-Robinson
 Density option: API method
 CODE REPORT

MISCELLANEOUS

Recycle calculations are converged.

Recycle equipment list (KE2): 2, 3, 5, 6, 4, 7.

Streams used in conv. routine (KE4): (0)
(0)=Delay factor

Preferred cut stream list (KE3): 10, 11.

Convergence tolerances,	Error
Flowrates:	.00100000
Vapor fraction:	.00100000
Temperature:	.00100000
Pressure:	.00100000
Enthalpy:	.00100000
Flash calcs:	.00005000

Max. loops in recycle calc.: 30
in flash calcs: 75

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

*** VALVES ***		
Equipment no.	5	6
External name		
Outlet pressure bars	1.4006	1.4572

EXCHANGER/CONDENSERS

Equipment no.	2	3	4	7
External name				
Heat transfer coeff.	300.00	300.00	300.00	.00000
Area	5.7562	5.8855	.53238	.00000
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	3.0	3.0	3.0	5.0
Min. delta T or T-out	6.00	3.00	7.00	-175.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	-10.614	-7.1327	1.3775	.15194
Water usage, DM3/hr	.00000	.00000	.00000	.00000
Corrected delta T	6.15	4.04	8.62	8.62
Equipment no.	10			
External name				
Heat transfer coeff.	.00000			
Area	.00000			
Number of shells	.0			
Shell passes	.0			
Tube passes	.0			
Mode	5.0			
Min. delta T or T-out	-180.00			
Delta P, stream 1	.00000			
Delta P, stream 2	.00000			
Q, stream 1 MJ/hr	1.1303			
Water usage, DM3/hr	.00000			
Corrected delta T	B.62			

*** PUMPS/COMPRESSORS ***

186

Equipment no.	8
External name	
Number of stages	.0
Work capacity MJ/hr	.00000
Outlet pressure bars	190.00
Power type:	.0
(+) steam	
(0) electricity	
(-) fuel gas	
H, steam out KJ / kg	.00000
Fuel usage, M3 /hr	.00000
Water usage, DM3/hr	.00000
Steam usage Tonnes/hr	.00000
Kilowatt usage	1.1632

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1985
All Rights Reserved.

*** RIGOROUS TOWERS ***

Equipment no.	1	7
External name		
Number of stages	21.0	57.0
Feed 1, stage #	15.0	1.0
Feed 2, stage #	21.0	17.0
Feed 3, stage #	.0	.0
Feed 4, stage #	.0	.0
Feed 5, stage #	.0	.0
Sidestream # 1 stage	.0	.0
Sidestream # 2 stage	.0	.0
Sidestream # 3 stage	.0	.0
Sidestream # 4 stage	.0	.0
Cond. pressure bars	6.7131	1.4006
Cond. delta P bars	.00000	.00000
Colm. delta P bars	.19400E-01	.19850
Condenser type	.0	1.0
Condenser mode	4.0	.0
Value of cond. spec.	16.000	.00000
Cond comp 1 position	1.0	.0
Cond comp 2 position	.0	.0
Cond. deg. subcooled	1.00	.00
Reboiler mode	.0	4.0
Val. of reboiler spec	.00000	7.6000
Rebr comp 1 position	.0	2.0
Rebr comp 2 position	.0	.0
Damping ratio	.00000	1.0000
Sidestream 1 mode	.0	.0
Sidestream 2 mode	.0	.0
Sidestream 3 mode	.0	.0
Sidestream 4 mode	.0	.0
Sidestream # 1 spec	.00000	.00000
Sidestream # 2 spec	.00000	.00000
Sidestream # 3 spec	.00000	.00000
Sidestream # 4 spec	.00000	.00000
Sidestrm 1 comp posn	.0	.0
Sidestrm 2 comp posn	.0	.0
Sidestrm 3 comp posn	.0	.0

Sidestrm 4 comp posn	.0	.0
Cond. duty MJ/hr	-163.53	.00000
Rebr duty MJ/hr	.00000	153.21
Est. dist rate Kgmol/hr	16.000	.00000
Est. reflux Kgmol/hr	.00000	.00000
Est. side draw rate 1	.00000	.00000
Est. side draw rate 2	.00000	.00000
Est. side draw rate 3	.00000	.00000
Est. side draw rate 4	.00000	.00000
Est. temp stg 1 C	-178.00	-193.00
Est. temp stg N C	-174.00	-183.00

.....

CODE REPORT

*** Stream no. 1 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.366		
Pressure bars.	6.61325		
Vapor fraction	.817724		
Enthalpy MJ/hr	214.671	187.651	27.0201
Entropy MJ/hr*K	-.474310	-.269548	-.204763
Ave. mol. wt.	28.7704	28.6655	29.2409
Total flow kg/hr	460.324	375.046	85.2782
kgmol/hr	15.9999	13.0835	2.91641
Density kg/m3		26.7085	1406.48
Viscosity centipoise		.690154E-02	.103552
Thermal cond. cal/cm*s*K		.357696E-04	.505017E-03
Specific heat kJ/kg*K		1.25193	2.17510
Z factor		.847052	
m3/hr (15.6 deg C & 1 atm)		309.950	
Vol. flowrate m3/hr		14.0424	
Surface tension dyne/cm.			5.67519
G. G. (60/60)			.915033
m3/hr (15.6 deg C & 1 atm)			.933430E-01
Vol. flowrate m3/hr			.606330E-01
Nitrogen	.83630	Liquid mole fraction	Liquid flowrate
Oxygen	.16370	.69196	kgmol/hr
		10.9418	kgmol/hr
		2.14175	.898379

*** Stream no. 2 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.347		
Pressure bars.	6.60647		
Vapor fraction	.832464		
Enthalpy MJ/hr	323.802	286.566	37.2358
Entropy MJ/hr*K	-.693623	-.411253	-.282370
Ave. mol. wt.	28.7702	28.6730	29.2536
Total flow kg/hr	690.486	572.861	117.625
kgmol/hr	24.0000	19.9791	4.02087
Density kg/m3		26.6758	1407.54
Viscosity centipoise		.690418E-02	.103704
Thermal cond. cal/cm*s*K		.357676E-04	.504990E-03
Specific heat kJ/kg*K		1.25085	2.17289
Z factor		.847282	
m3/hr (15.6 deg C & 1 atm)		473.307	
Vol. flowrate m3/hr		21.4752	
Surface tension dyne/cm.			5.69055

S. G. (60/60)915375	188
m3/hr (15.6 deg C & 1 atm)128700	
Vol. flowrate m3/hr835686E-01	
	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr
Nitrogen .83443	.68876	16.6712	2.76943	
Oxygen .16557	.31124	3.30796	1.25144	

CODE REPORT

*** Stream no. 3 ***	All Liquid
Temperature deg C.	-175.936
Pressure bars.	6.71314
Enthalpy MJ/hr	152.106
Entropy MJ/hr*K	-1.12246
Ave. mol. wt.	28.0883
Total flow kg/hr.	449.415
kgmol/hr	16.0001
Density kg/m3	1329.85
Viscosity centipoise947183E-01
Thermal cond. cal/cm*s*K510523E-03
Specific heat kJ/kg*K	2.34680
Surface tension dyne/cm.	4.74204
S. G. (60/60)883883
m3/hr (15.6 deg C & 1 atm)509252
Vol. flowrate m3/hr337948
	Liquid
	mole
	fraction
Nitrogen .981119	15.6980
Oxygen .188808E-01	.302095

*** Stream no. 4 ***	All Liquid
Temperature deg C.	-172.151
Pressure bars.	6.73254
Enthalpy MJ/hr	222.844
Entropy MJ/hr*K	-1.68106
Ave. mol. wt.	29.2250
Total flow kg/hr.	701.395
kgmol/hr	23.9999
Density kg/m3	1402.47
Viscosity centipoise102867
Thermal cond. cal/cm*s*K504666E-03
Specific heat kJ/kg*K	2.18426
Surface tension dyne/cm.	5.60303
S. G. (60/60)914606
m3/hr (15.6 deg C & 1 atm)768084
Vol. flowrate m3/hr500121
	Liquid
	mole
	fraction
Nitrogen .695938	16.7024
Oxygen .304062	7.29745

CODE REPORT

*** Stream no. 5 ***	All Liquid
Temperature deg C.	-186.647

Pressure bars.	6.71314		
Enthalpy MJ/hr	141.493	180	
Entropy MJ/hr*K	-1.23802		
Ave. mol. wt.	28.0883		
Total flow kg/hr.	449.415		
kgmol/hr	16.0001		
Density kg/m3	1445.97		
Viscosity centipoise123407		
Thermal cond. cal/cm*s*K527857E-03		
Specific heat kJ/kg*K	2.09369		
Surface tension dyne/cm.	6.98386		
S. G. (60/60)883883		
m3/hr (15.6 deg C & 1 atm)509252		
Vol. flowrate m3/hr310809		
Liquid	Liquid		
mole	flowrate		
fraction	kgmol/hr		
Nitrogen981119	15.6980	
Oxygen188808E-01	.302095	
*** Stream no. 6 ***			
Temperature deg C.	All Liquid		
Pressure bars.	-176.935		
Enthalpy MJ/hr	6.73254		
Entropy MJ/hr*K	215.711		
Ave. mol. wt.	-1.75337		
Total flow kg/hr.	29.2250		
kgmol/hr	701.395		
Density kg/m3	23.9999		
Viscosity centipoise	1454.64		
Thermal cond. cal/cm*s*K113776		
Specific heat kJ/kg*K512408E-03		
Surface tension dyne/cm.	2.06976		
S. G. (60/60)	6.67650		
m3/hr (15.6 deg C & 1 atm)914606		
Vol. flowrate m3/hr768084		
Liquid	Liquid		
mole	flowrate		
fraction	kgmol/hr		
Nitrogen695938	16.7024	
Oxygen304062	7.29745	
CODU REPORT			
*** Stream no. 7 ***			
Temperature deg C.	Overall	Vapor	Liquid
Pressure bars.	-192.652		
Vapor fraction	1.40055		
Enthalpy MJ/hr668986E-01		
Entropy MJ/hr*K	141.493	15.1463	126.409
Ave. mol. wt.	-1.23147	-1.39799E-01	-1.21749
Total flow kg/hr	28.0883	28.0396	28.0918
kgmol/hr	449.415	30.0132	419.402
Density kg/m3	16.0001	1.07038	14.9297
Viscosity centipoise		6.12529	1503.80
Thermal cond. cal/cm*s*K541943E-02	.147473
Specific heat kJ/kg*K340943E-04	.537581E-03
Z factor		1.13199	1.97733
m3/hr (15.6 deg C & 1 atm)947868	
		25.6268	

Vol. flowrate m3/hr	4.89995			190
Surface tension dyne/cm.		8.31471		
S. G. (60/60)883978		
m3/hr (15.6 deg C & 1 atm)475192		
Vol. flowrate m3/hr278897		
Vapor mole fraction	Vapor mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr	
Nitrogen99332	.98023	1.06323	14.6346
Oxygen66838E-02	.19765E-01	.715428E-02	.295089
*** Stream no. 8 ***				
Overall Temperature deg C.	-190.027	Vapor	Liquid	
Pressure bars.	1.45725			
Vapor fraction138577			
Enthalpy MJ/hr	215.711	46.6854	168.988	
Entropy MJ/hr*K	-1.73072	-.409738E-01	-1.68975	
Ave. mol. wt.	29.2250	28.5267	29.3371	
Total flow kg/hr	701.395	94.8750	606.515	
kgmol/hr	23.9999	3.32583	20.6740	
Density kg/m3		6.34762	1590.31	
Viscosity centipoise566974E-02	.161625	
Thermal cond. cal/cm*s*K339222E-04	.533619E-03	
Specific heat kJ/kg*K		1.10301	1.84662	
Z factor949507		
m3/hr (15.6 deg C & 1 atm)		78.6319		
Vol. flowrate m3/hr		14.9468		
Surface tension dyne/cm.			9.92894	
S. G. (60/60)917617	
m3/hr (15.6 deg C & 1 atm)662004	
Vol. flowrate m3/hr381387	
Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr	
Nitrogen87111	.66782	2.89717	13.8066
Oxygen12889	.33218	.428659	.686744
CODE REPORT				
*** Stream no. 9 ***				
Overall Temperature deg C.	-190.008	Vapor	Liquid	
Pressure bars.	1.45725			
Vapor fraction148943			
Enthalpy MJ/hr	217.088	50.2454	166.870	
Entropy MJ/hr*K	-1.71318	-.440288E-01	-1.66915	
Ave. mol. wt.	29.2250	28.5310	29.3463	
Total flow kg/hr	701.395	101.987	599.406	
kgmol/hr	23.9999	3.57461	20.4252	
Density kg/m3		6.33859	1590.93	
Viscosity centipoise567168E-02	.161681	
Thermal cond. cal/cm*s*K339417E-04	.533592E-03	
Specific heat kJ/kg*K		1.10308	1.84427	
Z factor949532		
m3/hr (15.6 deg C & 1 atm)		84.6253		
Vol. flowrate m3/hr		16.0901		
Surface tension dyne/cm.			9.94126	
S. G. (60/60)917666	
m3/hr (15.6 deg C & 1 atm)654067	
Vol. flowrate m3/hr376768	

	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr
Nitrogen	.87004	.66549	3.11004	13.5929
Oxygen	.12996	.33451	.464567	6.83238

*** Stream no. 10 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr.
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Z factor
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr

All Vapor
 -192.647
 1.40055
 453.660
 -.421952
 28.0407
 908.515
 32.3999
 6.19011
 .541992E-02
 .339356E-04
 1.13097
 .947875
 767.560
 146.771

	Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen	.993061	32.1751
Oxygen	.693924E-02	.224831

CODE REPORT

*** Stream no. 11 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr.
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.
 kgmol/hr
 Density kg/m3
 Viscosity centipoise
 Thermal cond. cal/cm*s*K
 Specific heat kJ/kg*K
 Surface tension dyne/cm.
 S. G. (60/60)
 m3/hr (15.6 deg C & 1 atm)
 Vol. flowrate m3/hr

All Liquid
 -179.524
 1.59905
 58.1288
 -.604787
 31.8808
 242.295
 7.60001
 1708.80
 .175472
 .517224E-03
 1.63202
 12.0760
 .985091
 .246347
 .141794

	Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen	.296430E-01	.225287
Oxygen	.970357	7.37472

*** Stream no. 12 ***

Temperature deg C.
 Pressure bars.
 Enthalpy MJ/hr.
 Entropy MJ/hr*K
 Ave. mol. wt.
 Total flow kg/hr.

All Liquid
 -183.027
 1.59905
 56.7513
 -.619821
 31.8809
 242.294

kgmol/hr	7.59999
Density kg/m ³	1736.77
Viscosity centipoise190473
Thermal cond. cal/cm*s*K . .	.522893E-03
Specific heat kJ/kg*K . .	1.61429
Surface tension dyne/cm. . .	12.9410
S. G. (60/60)985091
m ³ /hr (15.6 deg C & 1 atm) .	.246347
Vol. flowrate m ³ /hr139510
Liquid	Liquid
mole	flowrate
fraction	kgmol/hr
Nitrogen296411E-01
Oxygen970359

102

CODE REPORT

*** Stream no. 13 ***

Temperature deg C.	All Vapor
Pressure bars.	-182.231
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	464.274
Ave. mol. wt.	-.297945
Total flow kg/hr.	28.0407
kgmol/hr	908.515
Density kg/m ³	32.3999
Viscosity centipoise	5.40047
Thermal cond. cal/cm*s*K . .	.612500E-02
Specific heat kJ/kg*K . .	.350415E-04
Z factor	1.11274
m ³ /hr (15.6 deg C & 1 atm) .	.961997
Vol. flowrate m ³ /hr	767.562
Vapor	Vapor
mole	flowrate
fraction	kgmol/hr
Nitrogen	32.1751
Oxygen693924E-02

*** Stream no. 14 ***

Temperature deg C.	All Liquid
Pressure bars.	-183.027
Enthalpy MJ/hr	190.000
Entropy MJ/hr*K	59.0741
Ave. mol. wt.	-.632918
Total flow kg/hr.	31.8809
kgmol/hr	242.294
Density kg/m ³	7.59999
Viscosity centipoise	1851.29
Thermal cond. cal/cm*s*K . .	.190473
Specific heat kJ/kg*K . .	.522893E-03
Surface tension dyne/cm. . .	1.53771
S. G. (60/60)	12.9159
m ³ /hr (15.6 deg C & 1 atm) .	.985091
Vol. flowrate m ³ /hr246347
Liquid	Liquid
mole	flowrate
fraction	kgmol/hr
Nitrogen296411E-01
Oxygen970359

*** Stream no. 15 ***

Temperature deg C.	All Vapor
Pressure bars.	-175.151
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	471.407
Ave. mol. wt.	-.222392
Total flow kg/hr.	28.0407
kgmol/hr	908.515
Density kg/m3	32.3999
Viscosity centipoise	4.97526
Thermal cond. cal/cm*s*K660189E-02
Specific heat kJ/kg*K358105E-04
Z factor	1.10448
m3/hr (15.6 deg C & 1 atm)968776
Vol. flowrate m3/hr	767.562
Vapor		182.609
mole		Vapor
fraction		flowrate
Nitrogen	.993061	kgmol/hr
Oxygen	.693924E-02	32.1751
		.224831

*** Stream no. 16 ***

Temperature deg C.	All Vapor
Pressure bars.	-175.000
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	471.558
Ave. mol. wt.	-.220847
Total flow kg/hr.	28.0407
kgmol/hr	908.515
Density kg/m3	32.3999
Viscosity centipoise	4.96695
Thermal cond. cal/cm*s*K661202E-02
Specific heat kJ/kg*K358270E-04
Z factor	1.10414
m3/hr (15.6 deg C & 1 atm)968902
Vol. flowrate m3/hr	767.562
Vapor		182.915
mole		Vapor
fraction		flowrate
Nitrogen	.993061	kgmol/hr
Oxygen	.693924E-02	32.1751

CODE REPORT

*** Stream no. 17 ***

Temperature deg C.	All Liquid
Pressure bars.	-180.000
Enthalpy MJ/hr	190.000
Entropy MJ/hr*K	60.2043
Ave. mol. wt.	-.620582
Total flow kg/hr.	31.8809
kgmol/hr	242.294
Density kg/m3	7.59999
Viscosity centipoise	1832.13
Thermal cond. cal/cm*s*K177374
Specific heat kJ/kg*K517994E-03

1	514.0	28.09	.50	.39	1329.83	.9947	4.74
2	519.4	28.17	.59	.39	1324.15	.9934	4.62
3	517.4	28.25	.58	.39	1330.76	.9941	4.69
4	515.4	28.34	.58	.39	1337.70	.9949	4.78
5	513.3	28.44	.58	.38	1344.85	.9957	4.87
6	511.2	28.53	.57	.38	1352.06	.9966	4.95
7	509.1	28.63	.57	.37	1359.18	.9974	5.04
8	506.7	28.72	.56	.37	1366.03	.9983	5.13
9	505.6	28.80	.56	.37	1372.47	.9991	5.21
10	503.8	28.89	.56	.37	1378.42	.9998	5.29
11	502.3	28.96	.55	.36	1383.77	.1005	5.36
12	501.0	29.03	.55	.36	1388.50	.1011	5.42
13	500.1	29.08	.55	.36	1392.60	.1016	5.47
14	499.1	29.13	.55	.36	1396.11	.1020	5.52
15	584.2	29.17	.64	.42	1399.05	.1024	5.56
16	584.0	29.18	.64	.42	1399.15	.1024	5.56
17	584.1	29.18	.64	.42	1399.33	.1025	5.57
18	584.0	29.18	.64	.42	1399.65	.1025	5.57
19	583.7	29.19	.64	.42	1400.20	.1026	5.58
20	583.5	29.20	.64	.42	1401.05	.1027	5.58
21	701.4	29.22	.77	.50	1402.45	.1029	5.60

Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow	Actual Vol. flow	Actual density
			m3/hr	m3/hr	Z

1	*** Total Condenser : No Vapor Outlet ***				
2	963.4	28.09	812.5	34.93	.8356
3	968.8	28.13	815.9	35.17	.8364
4	966.8	28.18	812.9	35.14	.8372
5	964.8	28.22	809.8	35.11	.8380
6	962.7	28.27	806.7	35.08	.8388
7	960.6	28.32	803.5	35.06	.8397
8	958.5	28.37	800.4	35.03	.8405
9	956.1	28.42	797.0	34.98	.8413
10	955.0	28.46	794.8	34.98	.8420
11	953.3	28.50	792.3	34.96	.8426
12	951.7	28.54	790.0	34.93	.8432
13	950.4	28.57	788.0	34.91	.8437
14	949.5	28.60	786.4	34.90	.8442
15	948.5	28.63	784.9	34.88	.8446
16	573.2	28.63	474.3	21.08	.8446
17	573.1	28.63	474.2	21.07	.8446
18	573.2	28.63	474.2	21.07	.8446
19	573.1	28.64	474.1	21.07	.8446
20	572.8	28.65	473.7	21.06	.8448
21	572.6	28.66	473.3	21.06	.8449

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

TOWER # 1 Data file: column.TSZ Profile file: link.prf

Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm) 7.0000
 Downcomer area (ft² or m²) .0151
 Weir length (in. or cm) 24.0000
 Weir height (in. or cm) 3.0000

Stg	Diameter meter	Liquid Vol. flow m ³ /min	Vapor Vol. flow m ³ /sec	Liquid density kg/m ³	Vapor density kg/m ³	Flooding velocity m/sec	Settling height meter
1 ***Condenser: No tray sizing***							
2	.3051	.01	.01	1324.1490	27.5839	.1672	.0361
3	.3053	.01	.01	1330.7550	27.5507	.1681	.0361
4	.3047	.01	.01	1337.6980	27.5150	.1689	.0361
5	.3040	.01	.01	1344.8480	27.4776	.1696	.0361
6	.3034	.01	.01	1352.0650	27.4397	.1704	.0362
7	.3028	.01	.01	1359.1760	27.4020	.1711	.0362
8	.3022	.01	.01	1366.0340	27.3657	.1718	.0362
9	.3016	.01	.01	1372.4660	27.3312	.1724	.0362
10	.3012	.01	.01	1378.4210	27.2995	.1731	.0362
11	.3007	.01	.01	1383.7730	27.2706	.1738	.0363
12	.3003	.01	.01	1388.5020	27.2455	.1741	.0363
13	.3000	.01	.01	1392.6050	27.2244	.1745	.0363
14	.2997	.01	.01	1396.1050	27.2063	.1749	.0363
15	.3029	.01	.01	1399.0530	27.1921	.1701	.0359
16	.2615	.01	.01	1399.1530	27.1950	.1517	.0359
17	.2615	.01	.01	1399.3330	27.1977	.1517	.0359
18	.2615	.01	.01	1399.6530	27.1997	.1517	.0359
19	.2614	.01	.01	1400.1980	27.2007	.1517	.0359
20	.2613	.01	.01	1401.0550	27.1990	.1518	.0359
21	.2661	.01	.01	1402.4480	27.1939	.1444	.0359

CODE REPORT

TOWER # 2

Data file: column.OUT

Profile file: pool.prf

Column Summary

Stg	Temp C	Pres bars	Net Flow Rates			Duties MJ/hr
			Liquid	Vapor	Feed	
1	-192.6	1.401	14.924		16.000	32.400
2	-192.6	1.404	14.920	31.324		
3	-192.6	1.408	14.915	31.320		
4	-192.5	1.411	14.907	31.315		
5	-192.5	1.415	14.896	31.307		
6	-192.5	1.418	14.881	31.296		
7	-192.4	1.422	14.861	31.281		
8	-192.3	1.425	14.832	31.261		
9	-192.2	1.429	14.794	31.232		
10	-192.1	1.432	14.742	31.194		
11	-192.0	1.436	14.674	31.142		
12	-191.8	1.440	14.587	31.074		
13	-191.6	1.443	14.478	30.987		
14	-191.3	1.447	14.348	30.878		
15	-191.0	1.450	14.199	30.748		
16	-190.6	1.454	14.037	30.598		
17	-190.2	1.457	34.458	30.437	24.000	
18	-190.2	1.461	34.463	26.858		
19	-190.2	1.464	34.468	26.863		

20 -190.2	1.468	34.473	26.868
21 -190.2	1.471	34.478	26.873
22 -190.1	1.475	34.483	26.878
23 -190.1	1.479	34.488	26.883
24 -190.1	1.482	34.493	26.888
25 -190.1	1.486	34.498	26.893
26 -190.0	1.489	34.503	26.898
27 -190.0	1.493	34.508	26.903
28 -190.0	1.496	34.513	26.908
29 -190.0	1.500	34.518	26.913
30 -189.9	1.503	34.523	26.918
31 -189.9	1.507	34.528	26.923
32 -189.9	1.510	34.533	26.928
33 -189.9	1.514	34.538	26.933
34 -189.8	1.518	34.542	26.938
35 -189.8	1.521	34.547	26.943
36 -189.8	1.525	34.552	26.947
37 -189.8	1.528	34.557	26.952
38 -189.8	1.532	34.562	26.957
39 -189.7	1.535	34.567	26.962
40 -189.7	1.539	34.572	26.967
41 -189.7	1.542	34.577	26.972
42 -189.7	1.546	34.582	26.977
43 -189.6	1.549	34.587	26.982
44 -189.6	1.553	34.591	26.987
45 -189.6	1.557	34.595	26.991
46 -189.6	1.560	34.600	26.995
47 -189.5	1.564	34.602	27.000

CODE REPORT

	Liquid	Vapor	Feed	Product
--	--------	-------	------	---------

48 -189.5	1.567	34.601	27.002	
49 -189.5	1.571	34.590	27.001	
50 -189.4	1.574	34.554	26.990	
51 -189.3	1.578	34.451	26.954	
52 -189.0	1.581	34.183	26.851	
53 -188.3	1.585	33.566	26.583	
54 -186.7	1.588	32.488	25.966	
55 -184.1	1.592	31.381	24.888	
56 -181.3	1.596	30.829	23.781	
57 -179.5	1.599		23.229	
			7.6000	153.21

Stream # 7 fed to Stg 1 is 6.690 % Vapor
 Stream # 9 fed to Stg 17 is 14.894 % Vapor

Stream # 10 is Vapor Distillate from Stg 1
 Stream # 11 is Liquid Bottoms from Stg 57

Condenser duty is .000000 MJ/hr
 Reboiler duty is 153.208 MJ/hr
 CODE REPORT

TOWR # 2 Data file: column.OUT Profile file: pool.prf
 Tray Liquid Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Actual Viscosity centipoise	Surface tension dyne/cm
-----	-----------------	------------	--------------------------	------------------------	----------------------	-----------------------------	-------------------------

1	419.3	28.09	.48	.20	1504.01	.1477	0.32
2	419.2	28.10	.47	.28	1504.12	.1477	0.32
3	419.2	28.11	.47	.28	1504.37	.1476	0.33
4	419.1	28.11	.47	.28	1504.79	.1477	0.34
5	419.0	28.12	.47	.28	1505.45	.1477	0.36
6	418.8	28.14	.47	.28	1506.43	.1477	0.38
7	418.5	28.16	.47	.28	1507.85	.1479	0.42
8	418.1	28.19	.47	.28	1509.87	.1482	0.47
9	417.6	28.23	.47	.27	1512.67	.1485	0.53
10	417.0	28.28	.47	.27	1516.47	.1491	0.63
11	416.1	28.35	.47	.27	1521.52	.1499	0.75
12	414.9	28.45	.47	.27	1528.09	.1507	0.90
13	413.5	28.56	.46	.27	1536.33	.1522	0.98
14	411.8	28.70	.46	.27	1546.30	.1537	1.08
15	409.8	28.86	.45	.26	1557.78	.1558	1.30
16	407.7	29.04	.45	.26	1570.25	.1580	1.54
17	1007.2	29.23	1.10	.64	1582.90	.1602	1.78
18	1007.3	29.23	1.10	.64	1582.70	.1601	1.78
19	1007.5	29.23	1.10	.64	1582.51	.1600	1.77
20	1007.6	29.23	1.10	.64	1582.10	.1597	1.76
21	1007.8	29.23	1.10	.64	1581.91	.1596	1.76
22	1007.9	29.23	1.10	.64	1581.71	.1595	1.75
23	1008.1	29.23	1.10	.64	1581.51	.1594	1.75
24	1008.2	29.23	1.10	.64	1581.31	.1593	1.74
25	1008.4	29.23	1.10	.64	1581.12	.1592	1.74
26	1008.5	29.23	1.10	.64	1580.92	.1591	1.73
27	1008.6	29.23	1.10	.64	1580.72	.1590	1.72
28	1008.8	29.23	1.10	.64	1580.53	.1589	1.72
29	1008.9	29.23	1.10	.64	1580.33	.1588	1.71
30	1009.1	29.23	1.10	.64	1580.14	.1587	1.71
31	1009.2	29.23	1.11	.64	1579.94	.1586	1.70
32	1009.4	29.23	1.11	.64	1579.75	.1584	1.70
33	1009.5	29.23	1.11	.64	1579.55	.1583	1.69
34	1009.7	29.23	1.11	.64	1579.36	.1582	1.69
35	1009.8	29.23	1.11	.64	1579.16	.1581	1.68
36	1010.0	29.23	1.11	.64	1578.97	.1580	1.68
37	1010.1	29.23	1.11	.64	1578.78	.1579	1.67
38	1010.3	29.23	1.11	.64	1578.58	.1578	1.67
39	1010.4	29.23	1.11	.64	1578.39	.1577	1.66
40	1010.6	29.23	1.11	.64	1578.19	.1576	1.66
41	1010.7	29.23	1.11	.64	1578.01	.1575	1.65
42	1010.9	29.23	1.11	.64	1577.81	.1574	1.65
43	1011.0	29.23	1.11	.64	1577.62	.1573	1.64
44	1011.1	29.23	1.11	.64	1577.44	.1572	1.63
45	1011.3	29.23	1.11	.64	1577.27	.1571	1.63
46	1011.4	29.23	1.11	.64	1577.14	.1570	1.62
47	1011.5	29.23	1.11	.64	1577.14	.1569	1.62
48	1011.6	29.24	1.11	.64	1577.42	.1569	1.63
49	1011.5	29.24	1.11				

CODE REPORT

TOWER # 2 Data file: column.OUT
Tray Liquid Properties

Profile file: pool.prf

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Actual Viscosity centipoise	Surface tension dyne/cm
50	1011.1	29.26	1.11	.64	1578.52	.1571	9.65

51	1007.8	29.31	1.10	.61	1581.72	.1576	7.71
52	1006.4	29.44	1.10	.63	1590.22	.1591	9.87
53	998.6	29.75	1.08	.62	1610.15	.1627	10.26
54	986.0	30.35	1.05	.60	1646.29	.1692	10.78
55	976.1	31.10	1.01	.58	1684.65	.1753	11.72
56	975.4	31.64	1.00	.57	1703.76	.1764	12.04
57	242.3	31.88	.25	.14	1708.77	.1755	12.08

CODE REPORT

TOWR # 2 Data file: column.OUT Profile file: pool.prf
Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Z	Actual density kg/m3
1	908.5	28.04	767.6	146.77	.9479	6.1900
2	878.4	28.04	742.1	141.58	.9478	6.2042
3	878.3	28.04	742.0	141.25	.9477	6.2183
4	878.3	28.05	741.8	140.92	.9476	6.2323
5	878.2	28.05	741.7	140.60	.9476	6.2461
6	878.1	28.06	741.4	140.28	.9475	6.2595
7	877.9	28.06	741.1	139.95	.9474	6.2726
8	877.6	28.07	740.6	139.63	.9474	6.2850
9	877.2	28.09	739.9	139.31	.9474	6.2969
10	876.7	28.11	739.0	139.00	.9475	6.3077
11	876.1	28.13	737.8	138.68	.9475	6.3174
12	875.2	28.16	736.2	138.36	.9477	6.3256
13	874.0	28.21	734.1	138.03	.9479	6.3322
14	872.6	28.26	731.5	137.70	.9481	6.3369
15	870.9	28.32	728.4	137.36	.9484	6.3399
16	868.9	28.40	724.9	137.02	.9488	6.3416
17	866.8	28.48	721.1	136.66	.9492	6.3427
18	764.9	28.48	636.3	120.32	.9491	6.3569
19	765.0	28.48	636.4	120.07	.9490	6.3712
20	765.2	28.48	636.5	119.83	.9489	6.3855
21	765.3	28.48	636.6	119.58	.9488	6.3997
22	765.5	28.48	636.7	119.34	.9488	6.4141
23	765.6	28.48	636.9	119.10	.9487	6.4283
24	765.8	28.48	637.0	118.86	.9486	6.4426
25	765.9	28.48	637.1	118.62	.9485	6.4568
26	766.1	28.48	637.2	118.38	.9484	6.4712
27	766.2	28.48	637.3	118.14	.9483	6.4854
28	766.4	28.48	637.5	117.91	.9482	6.4997
29	766.5	28.48	637.6	117.67	.9481	6.5139
30	766.7	28.48	637.7	117.44	.9480	6.5282
31	766.8	28.48	637.8	117.20	.9479	6.5425
32	766.9	28.48	637.9	116.97	.9478	6.5568
33	767.1	28.48	638.0	116.74	.9478	6.5710
34	767.2	28.48	638.2	116.51	.9477	6.5853
35	767.4	28.48	638.3	116.28	.9476	6.5995
36	767.5	28.48	638.4	116.05	.9475	6.6138
37	767.7	28.48	638.5	115.82	.9474	6.6280
38	767.8	28.48	638.6	115.60	.9473	6.6423
39	768.0	28.48	638.7	115.37	.9472	6.6565
40	768.1	28.48	638.9	115.15	.9471	6.6707
41	768.3	28.48	639.0	114.93	.9470	6.6850
42	768.4	28.48	639.1	114.70	.9469	6.6992
43	768.6	28.48	639.2	114.48	.9469	6.7135

200

44	768.7	28.49	639.3	114.26	.9468	6.7277
45	768.9	28.49	639.4	114.04	.9467	6.7419
46	769.0	28.49	639.5	113.82	.9466	6.7561
47	769.1	28.49	639.6	113.60	.9465	6.7702
48	769.2	28.49	639.7	113.38	.9464	6.7843
49	769.3	28.49	639.7	113.16	.9463	6.7980

CODE REPORT

TOWER # 2 Data file: column.OUT Profile file: pool.prf
Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard	Actual	Actual
			Vol. flow m³/hr	Vol. flow m³/hr	density kg/m³
50	769.2	28.50	639.4	112.94	.9463
51	768.8	28.52	638.5	112.71	.9464
52	767.5	28.58	636.1	112.46	.9467
53	764.1	28.74	629.8	112.12	.9475
54	756.3	29.13	615.1	111.51	.9494
55	743.7	29.88	589.6	110.28	.9524
56	733.8	30.85	563.4	108.76	.9549
57	733.1	31.56	550.3	108.13	.9562

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

TOWER # 2 Data file: column.TSZ Profile file: pool.prf
Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm)	7.0000
Downcomer area (ft² or m²)	.0236
Weir length (in. or cm)	30.0000
Weir height (in. or cm)	3.0000

Stg	Diameter meter	Liquid	Vapor	Liquid	Vapor	Flooding	Settling
		Vol. flow m³/min	Vol. flow m³/sec	density kg/m³	density kg/m³	velocity m/sec	height meter
1	.3980	.00	.04	1504.0130	6.1901	.4043	.0373
2	.3922	.00	.04	1504.1240	6.2043	.4046	.0373
3	.3920	.00	.04	1504.3670	6.2184	.4042	.0373
4	.3918	.00	.04	1504.7920	6.2324	.4038	.0373
5	.3915	.00	.04	1505.4480	6.2462	.4035	.0373
6	.3913	.00	.04	1506.4320	6.2596	.4032	.0373
7	.3910	.00	.04	1507.8550	6.2727	.4029	.0373
8	.3907	.00	.04	1509.8700	6.2851	.4028	.0373
9	.3904	.00	.04	1512.6680	6.2970	.4028	.0373
10	.3899	.00	.04	1516.4700	6.3078	.4029	.0373
11	.3894	.00	.04	1521.5240	6.3175	.4033	.0373
12	.3889	.00	.04	1528.0850	6.3257	.4038	.0373
13	.3882	.00	.04	1536.3290	6.3323	.4046	.0374
14	.3874	.00	.04	1546.3010	6.3370	.4057	.0374
15	.3865	.00	.04	1557.7830	6.3400	.4070	.0374

16	.3856	.00	.04	1570.2490	6.3417	.4085	.0374
17	.3935	.01	.04	1582.9050	6.3428	.3874	.0353
18	.3775	.01	.03	1582.7020	6.3570	.3785	.0353
19	.3774	.01	.03	1582.5090	6.3713	.3780	.0353
20	.3773	.01	.03	1582.3070	6.3856	.3775	.0353
21	.3772	.01	.03	1582.1040	6.3998	.3769	.0353
22	.3771	.01	.03	1581.9110	6.4142	.3764	.0353
23	.3770	.01	.03	1581.7080	6.4284	.3759	.0353
24	.3769	.01	.03	1581.5140	6.4427	.3754	.0353
25	.3768	.01	.03	1581.3110	6.4569	.3748	.0353
26	.3767	.01	.03	1581.1170	6.4713	.3743	.0353
27	.3766	.01	.03	1580.9240	6.4855	.3738	.0353
28	.3765	.01	.03	1580.7200	6.4998	.3733	.0353

CODE REPORT

Smith-Dresser-Ohlswager Shortcut Technique

Stg	Diameter meter	Liquid Vol. flow m3/min	Vapor Vol. flow m3/sec	Liquid density kg/m3	Vapor density kg/m3	Flooding velocity m/sec	Settling height meter
29	.3764	.01	.03	1580.5260	6.5140	.3728	.0353
30	.3763	.01	.03	1580.3320	6.5283	.3723	.0353
31	.3762	.01	.03	1580.1380	6.5426	.3718	.0353
32	.3761	.01	.03	1579.9430	6.5569	.3713	.0353
33	.3760	.01	.03	1579.7490	6.5711	.3708	.0353
34	.3760	.01	.03	1579.5550	6.5854	.3703	.0353
35	.3759	.01	.03	1579.3600	6.5996	.3697	.0353
36	.3758	.01	.03	1579.1650	6.6139	.3692	.0353
37	.3757	.01	.03	1578.9700	6.6281	.3687	.0353
38	.3756	.01	.03	1578.7750	6.6423	.3683	.0353
39	.3755	.01	.03	1578.5810	6.6566	.3678	.0353
40	.3754	.01	.03	1578.3860	6.6708	.3673	.0353
41	.3753	.01	.03	1578.1910	6.6851	.3668	.0353
42	.3752	.01	.03	1578.0060	6.6993	.3663	.0353
43	.3751	.01	.03	1577.8130	6.7136	.3658	.0353
44	.3751	.01	.03	1577.6220	6.7278	.3653	.0353
45	.3750	.01	.03	1577.4390	6.7420	.3648	.0353
46	.3749	.01	.03	1577.2740	6.7562	.3643	.0353
47	.3748	.01	.03	1577.1440	6.7703	.3639	.0353
48	.3747	.01	.03	1577.1360	6.7844	.3634	.0353
49	.3746	.01	.03	1577.4200	6.7981	.3630	.0353
50	.3744	.01	.03	1578.5160	6.8109	.3628	.0353
51	.3740	.01	.03	1581.7190	6.8212	.3630	.0353
52	.3732	.01	.03	1590.2160	6.8254	.3642	.0353
53	.3714	.01	.03	1610.1480	6.8153	.3675	.0354
54	.3682	.01	.03	1646.2870	6.7822	.3738	.0355
55	.3643	.01	.03	1684.6500	6.7435	.3798	.0356
56	.3618	.01	.03	1703.7580	6.7469	.3814	.0356
57	***Reboiler: No tray sizing***						

Appendix E

Computer results of the column-tray sizing under 1.7 times the nominal flow rate of air

CODE: Chemical Engineering Simulation System
 (C) Copyright, CONDE / McGraw-Hill, 1986
 All Rights Reserved.

TOPOLOGY

Equipment	Stream Numbers
-----------	----------------

1 TOWR	1 2 -3 -4
2 HXER	3 10 -5 -13
3 HXER	4 13 -6 -15
4 HXER	8 11 -9 -12
5 VALV	5 -7 0 0
6 VALV	6 -8 0 0
7 TOWR	7 9 -10 -11
8 PUMP	12 -14 0 0
9 HXER	15 -16 0 0
10 HXER	14 -17 0 0

Stream Connections

Stream	Equipment
From	To

1	9	1
2	0	1
3	1	2
4	1	3
5	2	5
6	3	6
7	5	7
8	6	4
9	4	7
10	7	2
11	7	4
12	4	8
13	2	3
14	8	10
15	0	9
16	9	0
17	10	0

COMPONENTS 2

IP numbers 46, 47,

THERMODYNAMICS

Kvalue option: Peng-Robinson
 Enthalpy option: Peng-Robinson
 Density option: API method
 CODE REPORT

MISCELLANEOUS

Recycle calculations are converged.

Recycle equipment list (KE2): 2, 3, 5, 6, 4, 7,

Streams used in conv. routine (KE4): (0)
(0)=Delay factor

Preferred cut stream list (KE3): 10, 11,

Convergence tolerances,	Error
Flowrates:	.00100000
Vapor fraction:	.00100000
Temperature:	.00100000
Pressure:	.00100000
Enthalpy:	.00100000
Flash calcs:	.00005000

Max. loops in recycle calc.: 30
in flash calcs: 75

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986

All Rights Reserved.

*** VALVES ***	5	6
Equipment no.		
External name		
Outlet pressure bars	1.4006	1.4572

EXCHANGER/CONDENSERS				
	2	3	4	9
Equipment no.				
External name				
Heat transfer coeff.	300.00	300.00	300.00	.00000
Area	9.7848	10.011	.91125	.00000
Number of shells	.0	.0	.0	.0
Shell passes	.0	.0	.0	.0
Tube passes	.0	.0	.0	.0
Mode	3.0	3.0	3.0	5.0
Min. delta T or T-out	6.00	3.00	7.00	-175.00
Delta P, stream 1	.00000	.00000	.00000	.00000
Delta P, stream 2	.00000	.00000	.00000	.00000
Q, stream 1 MJ/hr	-18.045	-12.132	2.3613	.25606
Water usage, DM3/hr	.00000	.00000	.00000	.00000
Corrected delta T	6.15	4.04	8.64	8.64
Equipment no.	10			
External name				
Heat transfer coeff.	.00000			
Area	.00000			
Number of shells	.0			
Shell passes	.0			
Tube passes	.0			
Mode	5.0			
Min. delta T or T-out	-180.00			
Delta P, stream 1	.00000			
Delta P, stream 2	.00000			
Q, stream 1 MJ/hr	1.9207			
Water usage, DM3/hr	.00000			
Corrected delta T	8.64			

*** PUMPS/COMPRESSORS ***

205

Equipment no.	8
External name	
Number of stages	.0
Work capacity MJ/hr	.00000
Outlet pressure bars	190.00
Power type:	.0
(+) steam	
(0) electricity	
(-) fuel gas	
H _i steam out KJ / kg	.00000
Fuel usage, M3 /hr	.00000
Water usage, DM3/hr	.00000
Steam usage Tonnes/hr	.00000
Kilowatt usage	1.9769

CODE: Chemical Engineering Simulation System

(C) Copyright, COADE / McGraw-Hill, 1986

All Rights Reserved.

*** RIGOROUS TOWERS ***

Equipment no.	1	7
External name		
Number of stages	21.0	57.0
Feed 1, stage #	15.0	1.0
Feed 2, stage #	21.0	17.0
Feed 3, stage #	.0	.0
Feed 4, stage #	.0	.0
Feed 5, stage #	.0	.0
Sidestream # 1 stage	.0	.0
Sidestream # 2 stage	.0	.0
Sidestream # 3 stage	.0	.0
Sidestream # 4 stage	.0	.0
Cond. pressure bars	6.7131	1.4006
Cond. delta P bars	.00000	.00000
Colm. delta P bars	.19400E-01	.19850
Condenser type	.0	1.0
Condenser mode	4.0	.0
Value of cond. spec.	27.200	.00000
Cond comp 1 position	1.0	.0
Cond comp 2 position	.0	.0
Cond. deg. subcooled	1.00	.00
Reboiler mode	.0	4.0
Val. of reboiler spec	.00000	12.920
Rebr comp 1 position	.0	2.0
Rebr comp 2 position	.0	.0
Damping ratio	.00000	1.0000
Sidestream 1 mode	.0	.0
Sidestream 2 mode	.0	.0
Sidestream 3 mode	.0	.0
Sidestream 4 mode	.0	.0
Sidestream # 1 spec	.00000	.00000
Sidestream # 2 spec	.00000	.00000
Sidestream # 3 spec	.00000	.00000
Sidestream # 4 spec	.00000	.00000
Sidestrm 1 comp posn	.0	.0
Sidestrm 2 comp posn	.0	.0
Sidestrm 3 comp posn	.0	.0

Sidestream 4 comp posn	.0	.0
Cond. duty MJ/hr	-277.55	.00000
Refr. duty MJ/hr	.00000	260.55
Est. dist. rate Kgmol/hr	16.000	.00000
Est. reflux Kgmol/hr	.00000	.00000
Est. side draw rate 1	.00000	.00000
Est. side draw rate 2	.00000	.00000
Est. side draw rate 3	.00000	.00000
Est. side draw rate 4	.00000	.00000
Est. temp stg 1 C	-178.00	-193.00
Est. temp stg N C	-174.00	-183.00

CODE REPORT

*** Stream no. 1 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.366		
Pressure bars.	6.61325		
Vapor fraction .	.814077		
Enthalpy MJ/hr .	364.679	317.794	46.8845
Entropy MJ/hr*K .	-1.811794	-.456491	-.358303
Ave. mol. wt. .	28.7725	28.6655	29.2409
Total flow kg/hr .	783.129	635.157	147.972
kgmol/hr .	27.2180	22.1575	5.06045
Density kg/m3 .		26.7085	1406.48
Viscosity centipoise .		.690154E-02	.103556
Thermal cond. cal/cm*s*k .		.357696E-04	.505017E-03
Specific heat kJ/kg*K .		1.25117	2.17498
Z factor .		.847052	
m3/hr (15.6 deg C & 1 atm) .		524.913	
Vol. flowrate m3/hr .		23.7814	
Surface tension dyne/cm. .			5.67517
S. G. (60/60) .			.915033
m3/hr (15.6 deg C & 1 atm) .			.161966
Vol. flowrate m3/hr .			.105208
Nitrogen	.83630	Vapor mole fraction	Liquid mole fraction
Oxygen	.16370	.69196	18.5304
			3.50163
			1.55683

*** Stream no. 2 ***

	Overall	Vapor	Liquid
Temperature deg C.	-172.347		
Pressure bars. .	6.60647		
Vapor fraction .	.832295		
Enthalpy MJ/hr .	550.428	487.063	63.3645
Entropy MJ/hr*K .	-1.17950	-.698989	-.480512
Ave. mol. wt. .	28.7703	28.6730	29.2536
Total flow kg/hr .	1173.83	973.666	200.164
kgmol/hr .	40.8000	33.9576	6.84237
Density kg/m3 .		26.6758	1407.54
Viscosity centipoise .		.690418E-02	.103704
Thermal cond. cal/cm*s*k .		.357676E-04	.504990E-03
Specific heat kJ/kg*K .		1.25043	2.17322
Z factor .		.847282	
m3/hr (15.6 deg C & 1 atm) .		804.459	
Vol. flowrate m3/hr .		36.5005	
Surface tension dyne/cm. .			5.69054

207

S. G. (60/60)915375
m3/hr (15.6 deg C & 1 atm)219011
Vol. flowrate m3/hr142210
	Liquid
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .83443 .68876	28.3352
Oxygen .16557 .31124	5.62239

CODE REPORT

*** Stream no. 3 *** All Liquid

Temperature deg C.	-175.934
Pressure bars.	6.71314
Enthalpy MJ/hr	258.574
Entropy MJ/hr*K	-1.90819
Ave. mol. wt.	28.0894
Total flow kg/hr.	764.031
	kgmol/hr
Density kg/m3	27.1999
Viscosity centipoise	1329.95
Thermal cond. cal/cm*s*K947303E-01
Specific heat kJ/kg*K510520E-03
Surface tension dyne/cm.	2.34611
S. G. (60/60)	4.74328
m3/hr (15.6 deg C & 1 atm)883915
Vol. flowrate m3/hr865727
	Liquid
	mole
	fraction
Nitrogen .980825	flowrate kgmol/hr
Oxygen .191745E-01	26.6784

*** Stream no. 4 *** All Liquid

Temperature deg C.	-172.150
Pressure bars.	6.73254
Enthalpy MJ/hr	378.995
Entropy MJ/hr*K	-2.85909
Ave. mol. wt.	29.2255
Total flow kg/hr.	1192.93
	kgmol/hr
Density kg/m3	40.8180
Viscosity centipoise	1402.51
Thermal cond. cal/cm*s*K102874
Specific heat kJ/kg*K504664E-03
Surface tension dyne/cm.	2.18318
S. G. (60/60)	5.60360
m3/hr (15.6 deg C & 1 atm)914620
Vol. flowrate m3/hr	1.30633
	Liquid
	mole
	fraction
Nitrogen .695808	flowrate kgmol/hr
Oxygen .304192	28.4015

CODE REPORT

*** Stream no. 5 *** All Liquid

Temperature deg C.	-186.645
----------------------------	----------

Pressure bars.	6.71314	
Enthalpy MJ/hr	240.529	208
Entropy MJ/hr*K	-2.10464	
Ave. mol. wt.	28.0894	
Total flow kg/hr.	764.031	
kgmol/hr	27.1999	
Density kg/m ³	1446.06	
Viscosity centipoise	.123420	
Thermal cond. cal/cm*s*K	.527854E-03	
Specific heat kJ/kg*K	2.09313	
Surface tension dyne/cm.	6.98535	
S. G. (60/60)	.883915	
m ³ /hr (15.6 deg C & 1 atm)	.865727	
Vol. flowrate m ³ /hr	.528360	
Liquid	Liquid	
mole	flowrate	
fraction	kgmol/hr	
Nitrogen	.980825	26.6784
Oxygen	.191745E-01	.521546

*** Stream no. 6 ***

	All Liquid	
Temperature deg C.	-176.936	
Pressure bars.	6.73254	
Enthalpy MJ/hr	366.863	
Entropy MJ/hr*K	-2.98213	
Ave. mol. wt.	29.2255	
Total flow kg/hr.	1192.93	
kgmol/hr	40.0180	
Density kg/m ³	1454.70	
Viscosity centipoise	.113786	
Thermal cond. cal/cm*s*K	.512410E-03	
Specific heat kJ/kg*K	2.06949	
Surface tension dyne/cm.	6.67757	
S. G. (60/60)	.914620	
m ³ /hr (15.6 deg C & 1 atm)	1.30633	
Vol. flowrate m ³ /hr	.820056	
Liquid	Liquid	
mole	flowrate	
fraction	kgmol/hr	
Nitrogen	.695808	28.4015
Oxygen	.304192	12.4165

CODE REPORT

*** Stream no. 7 ***

	Overall	Vapor	Liquid
Temperature deg C.	-192.650		
Pressure bars.	1.40055		
Vapor fraction	.668756E-01		
Enthalpy MJ/hr	240.529	25.7341	214.895
Entropy MJ/hr*K	-2.09356	-.237600E-01	-2.06980
Ave. mol. wt.	28.0894	28.0401	28.0930
Total flow kg/hr.	764.031	51.0205	713.011
kgmol/hr	27.1999	1.81956	25.3804
Density kg/m ³		6.12840	1503.90
Viscosity centipoise		.541963E-02	.147499
Thermal cond. cal/cm*s*K		.340865E-04	.537577E-03
Specific heat kJ/kg*K		1.13159	1.97731
Z factor		.947871	
m ³ /hr (15.6 deg C & 1 atm)		43.5406	

Vol. flowrate m ³ /hr	8.32537			
Surface tension dyne/cm.		8.31641		
S. G. (60/60)801012		
m ³ /hr (15.6 deg C & 1 atm)807828		
Vol. flowrate m ³ /hr474114		
Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr	
Nitrogen99321	.97793	1.80720	24.8710
Oxygen67883E-02	.20072E-01	.123517E-01	.509438

*** Stream no. 8 ***

	Overall	Vapor	Liquid	
Temperature deg C.	-190.026			
Pressure bars.	1.45725			
Vapor fraction138561			
Enthalpy MJ/hr	366.863	79.3547	287.431	
Entropy MJ/hr*K	-2.94376	-.696667E-01	-2.87410	
Ave. mol. wt.	29.2255	28.5270	29.3375	
Total flow kg/hr	1192.93	161.342	1031.57	
kgmol/hr	40.8180	5.65578	35.1622	
Density kg/m ³		6.35054	1590.34	
Viscosity centipoise566984E-02	.161652	
Thermal cond. cal/cm*s*K339152E-04	.533617E-03	
Specific heat kJ/kg*K		1.10402	1.84609	
Z factor949508		
m ³ /hr (15.6 deg C & 1 atm)		133.656		
Vol. flowrate m ³ /hr		25.4064		
Surface tension dyne/cm.			9.92959	
S. G. (60/60)917630	
m ³ /hr (15.6 deg C & 1 atm)			1.12593	
Vol. flowrate m ³ /hr648657	
Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr	
Nitrogen87106	.66770	4.92650	23.4778
Oxygen12894	.33230	.729281	11.6844

CODE REPORT

*** Stream no. 9 ***

	Overall	Vapor	Liquid
Temperature deg C.	-190.007		
Pressure bars.	1.45725		
Vapor fraction149001		
Enthalpy MJ/hr	369.224	85.4078	283.830
Entropy MJ/hr*K	-2.91394	-.748885E-01	-2.83905
Ave. mol. wt.	29.2255	28.5313	29.3468
Total flow kg/hr	1192.93	173.525	1019.39
kgmol/hr	40.8180	6.08192	34.7361
Density kg/m ³		6.34457	1590.97
Viscosity centipoise567178E-02	.161735
Thermal cond. cal/cm*s*K339275E-04	.533589E-03
Specific heat kJ/kg*K		1.10395	1.84485
Z factor949533	
m ³ /hr (15.6 deg C & 1 atm)		143.847	
Vol. flowrate m ³ /hr		27.3505	
Surface tension dyne/cm.			9.94191
S. G. (60/60)917879
m ³ /hr (15.6 deg C & 1 atm)			1.11234
Vol. flowrate m ³ /hr640746

	Vapor mole fraction	Liquid mole fraction	Vapor flowrate kgmol/hr	Liquid flowrate kgmol/hr
Nitrogen	.86998	.66537	5.29115	23.1124
Oxygen	.13002	.33463	.790769	11.6237

*** Stream no. 10 ***

All Vapor

Temperature deg C.	-192.645
Pressure bars.	1.40055
Enthalpy MJ/hr	771.479
Entropy MJ/hr*K	-.717509
Ave. mol. wt.	28.0411
Total flow kg/hr.	1545.01
kgmol/hr	55.0980
Density kg/m3	6.19003
Viscosity centipoise542011E-02
Thermal cond. cal/cm*s*K339356E-04
Specific heat kJ/kg*K	.. .	1.13147
Z factor947878
m3/hr (15.6 deg C & 1 atm)	.. .	1305.28
Vol. flowrate m3/hr	249.600
Vapor mole fraction		Vapor flowrate kgmol/hr
Nitrogen	.992954	54.7098
Oxygen	.704628E-02	.388236

CODE REPORT

*** Stream no. 11 ***

All Liquid

Temperature deg C.	-179.494
Pressure bars.	1.59905
Enthalpy MJ/hr	98.8147
Entropy MJ/hr*K	-1.02800
Ave. mol. wt.	31.8848
Total flow kg/hr.	411.950
kgmol/hr	12.9199
Density kg/m3	1708.86
Viscosity centipoise175450
Thermal cond. cal/cm*s*K517176E-03
Specific heat kJ/kg*K	.. .	1.63153
Surface tension dyne/cm.	.. .	12.0763
S. G. (60/60)985195
m3/hr (15.6 deg C & 1 atm)418796
Vol. flowrate m3/hr241069
Liquid mole fraction		Liquid flowrate kgmol/hr
Nitrogen	.286495E-01	.370150
Oxygen	.971351	12.5498

*** Stream no. 12 ***

All Liquid

Temperature deg C.	-183.026
Pressure bars.	1.59905
Enthalpy MJ/hr	96.4537
Entropy MJ/hr*K	-1.05378
Ave. mol. wt.	31.8848
Total flow kg/hr.	411.951

kgmol/hr	12.9200
Density kg/m ³	1737.06
Viscosity centipoise190574
Thermal cond. cal/cm*s*K522892E-03
Specific heat kJ/kg*K	1.61370
Surface tension dyne/cm.	12.9483
S. G. (60/60)985194
m ³ /hr (15.6 deg C & 1 atm)418798
Vol. flowrate m ³ /hr237156
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen .286528E-01	.370194
Oxygen .971347	12.5498

CODE REPORT

*** Stream no. 13 ***

Temperature deg C.	All Vapor
Pressure bars.	-182.231
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	789.524
Ave. mol. wt.	-.506673
Total flow kg/hr.	28.0411
kgmol/hr	1545.01
Density kg/m ³	55.0980
Viscosity centipoise	5.40057
Thermal cond. cal/cm*s*K612507E-02
Specific heat kJ/kg*K350412E-04
Z factor	1.11367
m ³ /hr (15.6 deg C & 1 atm)961997
Vol. flowrate m ³ /hr	1305.28
Vapor mole fraction	Vapor flowrate kgmol/hr
Nitrogen .992954	54.7098
Oxygen .704622E-02	.388233

*** Stream no. 14 ***

Temperature deg C.	All Liquid
Pressure bars.	-183.026
Enthalpy MJ/hr	190.000
Entropy MJ/hr*K	100.402
Ave. mol. wt.	-.1.07603
Total flow kg/hr.	31.8848
kgmol/hr	411.951
Density kg/m ³	12.9200
Viscosity centipoise	1851.52
Thermal cond. cal/cm*s*K190574
Specific heat kJ/kg*K522892E-03
Surface tension dyne/cm.	1.53708
S. G. (60/60)	12.9240
m ³ /hr (15.6 deg C & 1 atm)985194
Vol. flowrate m ³ /hr418798
Liquid mole fraction	Liquid flowrate kgmol/hr
Nitrogen .286528E-01	.222496
Oxygen .971347	.370194
	12.5498

*** Stream no. 15 ***

Temperature deg C	All Vapor
Pressure bars.	-175.150
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	801.656
Ave. mol. wt.	-.378173
Total flow kg/hr.	28.0411
kgmol/hr	1545.01
Density kg/m3	55.0980
Viscosity centipoise	4.97529
Thermal cond. cal/cm*s*K660203E-02
Specific heat kJ/kg*K358103E-04
Z factor	1.10436
m3/hr (15.6 deg C & 1 atm)968776
Vol. flowrate m3/hr	1305.28
Vapor mole fraction	310.541
Nitrogen	Vapor
Oxygen	flowrate
	kgmol/hr
Nitrogen	54.7098
Oxygen388233

*** Stream no. 16 ***

Temperature deg C	All Vapor
Pressure bars.	-175.000
Enthalpy MJ/hr	1.40055
Entropy MJ/hr*K	801.912
Ave. mol. wt.	-.375563
Total flow kg/hr.	28.0411
kgmol/hr	1545.01
Density kg/m3	55.0980
Viscosity centipoise	4.96705
Thermal cond. cal/cm*s*K661210E-02
Specific heat kJ/kg*K358268E-04
Z factor	1.10436
m3/hr (15.6 deg C & 1 atm)968902
Vol. flowrate m3/hr	1305.28
Vapor mole fraction	311.056
Nitrogen	Vapor
Oxygen	flowrate
	kgmol/hr
Nitrogen	54.7098
Oxygen388233

CODE REPORT

*** Stream no. 17 ***

Temperature deg C.	All Liquid
Pressure bars.	-180.000
Enthalpy MJ/hr	190.000
Entropy MJ/hr*K	102.323
Ave. mol. wt.	-.1.05506
Total flow kg/hr.	31.8848
kgmol/hr	411.951
Density kg/m3	12.9200
Viscosity centipoise	1832.37
Thermal cond. cal/cm*s*K177473
Specific heat kJ/kg*K517995E-03
	1.54391

Surface tension dyne/cm. . .	12.1752
S. G. (60/60)985194
m3/hr (15.6 deg C & 1 atm) .	.418798
Vol. flowrate m3/hr224821
	Liquid
	mole
	fraction
Nitrogen286528E-01
Oxygen971347
	Liquid
	flowrate
	kgmol/hr
CODE REPORT	12.5498

213

TOWR # 1 Data file: coll7.out Profile file: link.prf

Column Summary

Stg	Temp C	Pres bars	Net Flow Rates				Duties MJ/hr
			Liquid	Vapor	Feed	Product	
1	-175.9	6.713	18.299			27.200	-277.55
2	-174.8	6.713	18.440	34.299			
3	-174.6	6.714	18.315	34.440			
4	-174.4	6.715	18.184	34.315			
5	-174.1	6.716	18.051	34.184			
6	-173.9	6.717	17.916	34.051			
7	-173.7	6.718	17.784	33.916			
8	-173.5	6.719	17.644	33.784			
9	-173.2	6.720	17.551	33.644			
10	-173.0	6.721	17.443	33.551			
11	-172.9	6.722	17.345	33.443			
12	-172.7	6.723	17.261	33.346			
13	-172.5	6.724	17.197	33.261			
14	-172.4	6.725	17.132	33.197			
15	-172.3	6.726	20.023	33.132	27.218		
16	-172.3	6.727	20.017	20.023			
17	-172.3	6.728	20.018	20.017			
18	-172.3	6.729	20.011	20.018			
19	-172.2	6.731	19.996	20.011			
20	-172.2	6.732	19.979	19.996			
21	-172.2	6.733		19.979	40.800	24.000	

Stream # 1 fed to Stg 15 is 81.408 % Vapor
 Stream # 2 fed to Stg 21 is 83.229 % Vapor

Stream # 3 is Liquid Distillate from Stg 1
Stream # 4 is Liquid Bottoms from Stg 2

Condenser duty is -277.550 MJ/hr
Reboiler duty is .000000 MJ/hr
CODE REPORT

TOWR # 1 Data file: col17.OUT Profile file: link.prf
Tray Liquid Properties

Stg	Mass flow	Avg	Standard	Actual	Actual	Surface
	kg/hr	mol wt	Vol. flow m ³ /hr	Vol. flow m ³ /hr	Density kg/m ³	Viscosity centipoise

1	514.0	28.09	.58	.39	1329.83	.0947	4.74
2	519.4	28.17	.59	.39	1324.15	.0934	4.62
3	517.4	28.25	.58	.39	1330.76	.0941	4.69
4	515.4	28.34	.58	.39	1337.70	.0949	4.78
5	513.3	28.44	.58	.38	1344.85	.0957	4.87
6	511.2	28.53	.57	.38	1352.06	.0966	4.95
7	509.1	28.63	.57	.37	1359.18	.0974	5.04
8	506.7	28.72	.56	.37	1366.03	.0983	5.13
9	505.6	28.80	.56	.37	1372.47	.0991	5.21
10	503.8	28.89	.56	.37	1378.42	.0998	5.29
11	502.3	28.96	.55	.36	1383.77	.1005	5.36
12	501.0	29.03	.55	.36	1388.50	.1011	5.42
13	500.1	29.08	.55	.36	1392.60	.1016	5.47
14	499.1	29.13	.55	.36	1396.11	.1020	5.52
15	584.2	29.17	.64	.42	1399.05	.1024	5.56
16	584.0	29.18	.64	.42	1399.15	.1024	5.56
17	584.1	29.18	.64	.42	1399.33	.1025	5.56
18	584.0	29.18	.64	.42	1399.65	.1025	5.57
19	583.7	29.19	.64	.42	1400.20	.1026	5.57
20	583.5	29.20	.64	.42	1401.05	.1027	5.58
21	701.4	29.22	.77	.50	1402.45	.1029	5.60

Tray Vapor Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Z	Actual density kg/m3
1 *** Total Condenser : No Vapor Outlet ***						
2	963.4	28.09	812.5	34.93	.8356	27.5835
3	968.8	28.13	815.9	35.17	.8364	27.5503
4	966.8	28.18	812.9	35.14	.8372	27.5146
5	964.8	28.22	809.8	35.11	.8380	27.4772
6	962.7	28.27	806.7	35.08	.8388	27.4394
7	960.6	28.32	803.5	35.06	.8397	27.4016
8	958.5	28.37	800.4	35.03	.8405	27.3653
9	956.1	28.42	797.0	34.98	.8413	27.3308
10	955.0	28.46	794.8	34.98	.8420	27.2991
11	953.3	28.50	792.3	34.96	.8426	27.2702
12	951.7	28.54	790.0	34.93	.8432	27.2451
13	950.4	28.57	788.0	34.91	.8437	27.2240
14	949.5	28.60	786.4	34.90	.8442	27.2059
15	948.5	28.63	784.9	34.88	.8446	27.1917
16	573.2	28.63	474.3	21.08	.8446	27.1946
17	573.1	28.63	474.2	21.07	.8446	27.1973
18	573.2	28.63	474.2	21.07	.8446	27.1993
19	573.1	28.64	474.1	21.07	.8446	27.2003
20	572.8	28.65	473.7	21.06	.8448	27.1986
21	572.6	28.66	473.3	21.06	.8449	27.1935

CODE: Chemical Engineering Simulation System
 (C) Copyright, COADE / McGraw-Hill, 1986
 All Rights Reserved.

TOWER # 1 Data file: col17.TZ2 Profile file: link.prf

Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm) 7.0000
 Downcomer area (ft² or m²) .0151
 Weir length (in. or cm) 24.0000
 Weir height (in. or cm) 3.0000

215

Stg	Diameter meter	Liquid Vol. flow m ³ /min	Vapor Vol. flow m ³ /sec	Liquid density kg/m ³	Vapor density kg/m ³	Flooding velocity m/sec	Settling height meter
1	***Condenser: No tray sizing***						
2	.3051	.01	.01	1324.1490	27.5839	.1672	.0361
3	.3053	.01	.01	1330.7550	27.5507	.1681	.0361
4	.3047	.01	.01	1337.6980	27.5150	.1689	.0361
5	.3040	.01	.01	1344.8480	27.4776	.1696	.0361
6	.3034	.01	.01	1352.0650	27.4397	.1704	.0362
7	.3028	.01	.01	1359.1760	27.4020	.1711	.0362
8	.3022	.01	.01	1366.0340	27.3657	.1718	.0362
9	.3016	.01	.01	1372.4660	27.3312	.1724	.0362
10	.3012	.01	.01	1378.4210	27.2995	.1731	.0362
11	.3007	.01	.01	1383.7730	27.2706	.1736	.0363
12	.3003	.01	.01	1388.5020	27.2455	.1741	.0363
13	.3000	.01	.01	1392.6050	27.2244	.1745	.0363
14	.2997	.01	.01	1396.1050	27.2063	.1749	.0363
15	.3029	.01	.01	1399.0530	27.1921	.1701	.0359
16	.2615	.01	.01	1399.1530	27.1950	.1517	.0359
17	.2615	.01	.01	1399.3330	27.1977	.1517	.0359
18	.2615	.01	.01	1399.6530	27.1997	.1517	.0359
19	.2614	.01	.01	1400.1980	27.2007	.1517	.0359
20	.2613	.01	.01	1401.0550	27.1990	.1518	.0359
21	.2661	.01	.01	1402.4480	27.1939	.1444	.0354

CODE REPORT

TOWR # 2

Data file: col17.OUT

Profile file: pool.prf

Column Summary

Stg	Temp C	Pres bars	Net Flow Rates kgmols/hr				Duties MJ/hr
			Liquid	Vapor	Feed	Product	
1	-192.6	1.401	14.924		27.200	55.098	
2	-192.6	1.404	14.920	31.324			
3	-192.6	1.408	14.915	31.320			
4	-192.5	1.411	14.907	31.315			
5	-192.5	1.415	14.896	31.307			
6	-192.5	1.418	14.881	31.296			
7	-192.4	1.422	14.861	31.281			
8	-192.3	1.425	14.832	31.261			
9	-192.2	1.429	14.794	31.232			
10	-192.1	1.432	14.742	31.194			
11	-192.0	1.436	14.674	31.142			
12	-191.8	1.440	14.587	31.074			
13	-191.6	1.443	14.478	30.987			
14	-191.3	1.447	14.348	30.878			
15	-191.0	1.450	14.199	30.748			
16	-190.6	1.454	14.037	30.598			
17	-190.2	1.457	34.458	30.437	40.818		
18	-190.2	1.461	34.463	26.858			
19	-190.2	1.464	34.468	26.863			

20 -190.2	1.468	34.473	26.868
21 -190.2	1.471	34.478	26.873
22 -190.1	1.475	34.483	26.878
23 -190.1	1.479	34.488	26.883
24 -190.1	1.482	34.493	26.888
25 -190.1	1.486	34.498	26.893
26 -190.0	1.489	34.503	26.898
27 -190.0	1.493	34.508	26.903
28 -190.0	1.496	34.513	26.908
29 -190.0	1.500	34.518	26.913
30 -189.9	1.503	34.523	26.918
31 -189.9	1.507	34.528	26.923
32 -189.9	1.510	34.533	26.928
33 -189.9	1.514	34.538	26.933
34 -189.8	1.518	34.542	26.938
35 -189.8	1.521	34.547	26.943
36 -189.8	1.525	34.552	26.947
37 -189.8	1.528	34.557	26.952
38 -189.8	1.532	34.562	26.957
39 -189.7	1.535	34.567	26.962
40 -189.7	1.539	34.572	26.967
41 -189.7	1.542	34.577	26.972
42 -189.7	1.546	34.582	26.977
43 -189.6	1.549	34.587	26.982
44 -189.6	1.553	34.591	26.987
45 -189.6	1.557	34.595	26.991
46 -189.6	1.560	34.600	26.995
47 -189.5	1.564	34.602	27.000

CODE REPORT

	Liquid	Vapor	Feed	Product
48 -189.5	1.567	34.601	27.002	
49 -189.5	1.571	34.590	27.001	
50 -189.4	1.574	34.554	26.990	
51 -189.3	1.578	34.451	26.954	
52 -189.0	1.581	34.183	26.851	
53 -188.3	1.585	33.566	26.583	
54 -186.7	1.588	32.488	25.966	
55 -184.1	1.592	31.381	24.088	
56 -181.3	1.596	30.829	23.781	
57 -179.5	1.599	23.229		7.6000 260.55

Stream # 7 fed to Stg 1 is 6.690 % Vapor
 Stream # 9 fed to Stg 17 is 14.900 % Vapor

Stream # 10 is Vapor Distillate from Stg 1
 Stream # 11 is Liquid Bottoms from Stg 57

Condenser duty is .000000 MJ/hr
 Reboiler duty is 260.549 MJ/hr
 CODE REPORT

TOWER # 2 Data file: col17.OUT Profile file: pool.prf
 Tray Liquid Properties

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Viscosity centipoise	Surface tension dyne/cm
-----	-----------------	------------	--------------------------	------------------------	----------------------	----------------------	-------------------------

1	419.3	28.09	.48	.28	1504.01	.1477	9.32
2	419.2	28.10	.47	.28	1504.12	.1477	9.32
3	419.2	28.11	.47	.28	1504.37	.1476	9.32
4	419.1	28.11	.47	.28	1504.79	.1476	9.33
5	419.0	28.12	.47	.28	1505.45	.1477	9.34
6	418.8	28.14	.47	.28	1506.43	.1477	9.36
7	418.5	28.16	.47	.28	1507.85	.1479	9.38
8	418.1	28.19	.47	.28	1509.87	.1482	9.42
9	417.6	28.23	.47	.28	1512.67	.1485	9.47
10	417.0	28.28	.47	.27	1516.47	.1491	9.53
11	416.1	28.35	.47	.27	1521.52	.1499	9.63
12	414.9	28.45	.47	.27	1528.09	.1509	9.75
13	413.5	28.56	.46	.27	1536.33	.1522	9.90
14	411.8	28.70	.46	.27	1546.30	.1539	9.08
15	409.8	28.86	.45	.26	1557.78	.1558	9.30
16	407.7	29.04	.45	.26	1570.25	.1580	9.54
17	1007.2	29.23	1.10	.64	1582.90	.1602	9.78
18	1007.3	29.23	1.10	.64	1582.70	.1601	9.78
19	1007.5	29.23	1.10	.64	1582.51	.1600	9.77
20	1007.6	29.23	1.10	.64	1582.31	.1598	9.77
21	1007.8	29.23	1.10	.64	1582.10	.1597	9.76
22	1007.9	29.23	1.10	.64	1581.91	.1596	9.76
23	1008.1	29.23	1.10	.64	1581.71.	.1595	9.75
24	1008.2	29.23	1.10	.64	1581.51	.1594	9.75
25	1008.4	29.23	1.10	.64	1581.31	.1593	9.74
26	1008.5	29.23	1.10	.64	1581.12	.1592	9.74
27	1008.6	29.23	1.10	.64	1580.92	.1591	9.73
28	1008.8	29.23	1.10	.64	1580.72	.1590	9.72
29	1008.9	29.23	1.10	.64	1580.53	.1589	9.72
30	1009.1	29.23	1.10	.64	1580.33	.1588	9.71
31	1009.2	29.23	1.11	.64	1580.14	.1587	9.71
32	1009.4	29.23	1.11	.64	1579.94	.1586	9.70
33	1009.5	29.23	1.11	.64	1579.75	.1584	9.70
34	1009.7	29.23	1.11	.64	1579.55	.1583	9.69
35	1009.8	29.23	1.11	.64	1579.36	.1582	9.69
36	1010.0	29.23	1.11	.64	1579.16	.1581	9.68
37	1010.1	29.23	1.11	.64	1578.97	.1580	9.68
38	1010.3	29.23	1.11	.64	1578.78	.1579	9.67
39	1010.4	29.23	1.11	.64	1578.58	.1578	9.67
40	1010.6	29.23	1.11	.64	1578.39	.1577	9.66
41	1010.7	29.23	1.11	.64	1578.19	.1576	9.66
42	1010.9	29.23	1.11	.64	1578.01	.1575	9.65
43	1011.0	29.23	1.11	.64	1577.81	.1574	9.65
44	1011.1	29.23	1.11	.64	1577.62	.1573	9.64
45	1011.3	29.23	1.11	.64	1577.44	.1572	9.63
46	1011.4	29.23	1.11	.64	1577.27	.1571	9.63
47	1011.5	29.23	1.11	.64	1577.14	.1570	9.63
48	1011.6	29.24	1.11	.64	1577.14	.1569	9.62
49	1011.5	29.24	1.11	.64	1577.42	.1569	9.63

CODE REPORT

TOWER # 2 Data file: col17.out
Tray Liquid Properties

Profile file: pool.prf

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Actual Density kg/m3	Actual Viscosity centipoise	Surface tension dyne/cm
50	1011.1	29.26	1.11	.64	1578.52	.1571	9.65

51	1009.8	29.31	1.10	.64	1581.72	.1576	9.71
52	1006.4	29.44	1.10	.63	1590.22	.1591	9.87
53	998.6	29.75	1.08	.62	1610.15	.1627	10.26
54	986.0	30.35	1.05	.60	1646.29	.1692	10.98
55	976.1	31.10	1.01	.58	1684.65	.1753	11.72
56	975.4	31.64	1.00	.57	1703.76	.1764	12.04
57	242.3	31.88	.25	.14	1708.77	.1755	12.08

CODE REPORT

TOWER # 2 Data file: col17.DAT
Tray Vapor Properties

Profile file: pool.prf

Stg	Mass flow kg/hr	Avg mol wt	Standard Vol. flow m3/hr	Actual Vol. flow m3/hr	Z	Actual density kg/m3
1	908.5	28.04	767.6	146.77	.9479	6.1900
2	878.4	28.04	742.1	141.58	.9478	6.2042
3	878.3	28.04	742.0	141.25	.9477	6.2183
4	878.3	28.05	741.8	140.92	.9476	6.2323
5	878.2	28.05	741.7	140.60	.9476	6.2461
6	878.1	28.06	741.4	140.28	.9475	6.2595
7	877.9	28.06	741.1	139.95	.9474	6.2726
8	877.6	28.07	740.6	139.63	.9474	6.2850
9	877.2	28.09	739.9	139.31	.9474	6.2969
10	876.7	28.11	739.0	139.00	.9475	6.3077
11	876.1	28.13	737.8	138.68	.9475	6.3174
12	875.2	28.16	736.2	138.36	.9477	6.3256
13	874.0	28.21	734.1	138.03	.9479	6.3322
14	872.6	28.26	731.5	137.70	.9481	6.3369
15	870.9	28.32	728.4	137.36	.9484	6.3399
16	868.9	28.40	724.9	137.02	.9488	6.3416
17	866.8	28.48	721.1	136.66	.9492	6.3427
18	764.9	28.48	636.3	120.32	.9491	6.3569
19	765.0	28.48	636.4	120.07	.9490	6.3712
20	765.2	28.48	636.5	119.83	.9489	6.3855
21	765.3	28.48	636.6	119.58	.9488	6.3997
22	765.5	28.48	636.7	119.34	.9488	6.4141
23	765.6	28.48	636.9	119.10	.9487	6.4283
24	765.8	28.48	637.0	118.86	.9486	6.4426
25	765.9	28.48	637.1	118.62	.9485	6.4568
26	766.1	28.48	637.2	118.38	.9484	6.4712
27	766.2	28.48	637.3	118.14	.9483	6.4854
28	766.4	28.48	637.5	117.91	.9482	6.4997
29	766.5	28.48	637.6	117.67	.9481	6.5139
30	766.7	28.48	637.7	117.44	.9480	6.5282
31	766.8	28.48	637.8	117.20	.9479	6.5425
32	766.9	28.48	637.9	116.97	.9478	6.5568
33	767.1	28.48	638.0	116.74	.9478	6.5710
34	767.2	28.48	638.2	116.51	.9477	6.5853
35	767.4	28.48	638.3	116.28	.9476	6.5995
36	767.5	28.48	638.4	116.05	.9475	6.6138
37	767.7	28.48	638.5	115.82	.9474	6.6280
38	767.8	28.48	638.6	115.60	.9473	6.6423
39	768.0	28.48	638.7	115.37	.9472	6.6565
40	768.1	28.48	638.9	115.15	.9471	6.6707
41	768.3	28.48	639.0	114.93	.9470	6.6850
42	768.4	28.48	639.1	114.70	.9469	6.6992
43	768.6	28.48	639.2	114.48	.9469	6.7135

44	768.7	28.49	639.3	114.26	.9468	6.7277	219
45	768.9	28.49	639.4	114.04	.9467	6.7419	
46	769.0	28.49	639.5	113.82	.9466	6.7561	
47	769.1	28.49	639.6	113.60	.9465	6.7702	
48	769.2	28.49	639.7	113.38	.9464	6.7843	
49	769.3	28.49	639.7	113.16	.9463	6.7980	

CODE REPORT

TOWER # 2 Data file: col17.OUT Profile file: pool.prf
Tray Vapor Properties

Stg	Mass flow	Avg	Standard	Actual	Actual	kg/m3
	kg/hr	mol wt	Vol. flow m3/hr	Vol. flow m3/hr	Z	
50	769.2	28.50	639.4	112.94	.9463	6.8108
51	768.8	28.52	638.5	112.71	.9464	6.8211
52	767.5	28.58	636.1	112.46	.9467	6.8253
53	764.1	28.74	629.8	112.12	.9475	6.8152
54	756.3	29.13	615.1	111.51	.9494	6.7821
55	743.7	29.88	589.6	110.28	.9524	6.7434
56	733.8	30.85	563.4	108.76	.9549	6.7469
57	733.1	31.56	550.3	108.13	.9562	6.7792

CODE: Chemical Engineering Simulation System
(C) Copyright, COADE / McGraw-Hill, 1986
All Rights Reserved.

TOWER # 2 Data file: col17.TSZ Profile file: pool.prf

Smith-Dresser-Ohlswager Shortcut Technique

User Provided Input Data

Tray spacing (in. or cm)	7.0000
Downcomer area (ft ² or m ²)	.0236
Weir length (in. or cm)	30.0000
Weir height (in. or cm)	3.0000

Stg	Diameter	Liquid	Vapor	Liquid	Vapor	Flooding	Settling
	meter	Vol. flow m3/min	Vol. flow m3/sec	density kg/m3	density kg/m3	velocity m/sec	height meter
1	.3980	.00	.04	1504.0130	6.1901	.4043	.0373
2	.3922	.00	.04	1504.1240	6.2043	.4046	.0373
3	.3920	.00	.04	1504.3670	6.2184	.4042	.0373
4	.3918	.00	.04	1504.7920	6.2324	.4038	.0373
5	.3915	.00	.04	1505.4480	6.2462	.4035	.0373
6	.3913	.00	.04	1506.4320	6.2596	.4032	.0373
7	.3910	.00	.04	1507.8550	6.2727	.4029	.0373
8	.3907	.00	.04	1509.8700	6.2851	.4028	.0373
9	.3904	.00	.04	1512.6680	6.2970	.4028	.0373
10	.3899	.00	.04	1516.4700	6.3078	.4029	.0373
11	.3894	.00	.04	1521.5240	6.3175	.4033	.0373
12	.3889	.00	.04	1528.0850	6.3257	.4038	.0373
13	.3882	.00	.04	1536.3290	6.3323	.4046	.0374
14	.3874	.00	.04	1546.3010	6.3370	.4057	.0374
15	.3865	.00	.04	1557.7830	6.3400	.4070	.0374

392091

16	.3856	.00	.04	1570.2490	6.3417	.4085	.0374
17	.3935	.01	.04	1582.9050	6.3428	.3874	.0353
18	.3775	.01	.03	1582.7020	6.3570	.3785	.0353
19	.3774	.01	.03	1582.5090	6.3713	.3780	.0353
20	.3773	.01	.03	1582.3070	6.3856	.3775	.0353
21	.3772	.01	.03	1582.1040	6.3998	.3769	.0353
22	.3771	.01	.03	1581.9110	6.4142	.3764	.0353
23	.3770	.01	.03	1581.7080	6.4284	.3759	.0353
24	.3769	.01	.03	1581.5140	6.4427	.3754	.0353
25	.3768	.01	.03	1581.3110	6.4569	.3748	.0353
26	.3767	.01	.03	1581.1170	6.4713	.3743	.0353
27	.3766	.01	.03	1580.9240	6.4855	.3738	.0353
28	.3765	.01	.03	1580.7200	6.4998	.3733	.0353

CODE REPORT

Smith-Dresser-Ohlswager Shortcut Technique

Stg	Diameter meter	Liquid Vol. flow m3/min	Vapor Vol. flow m3/sec	Liquid density kg/m3	Vapor density kg/m3	Flooding velocity m/sec	Settling height meter
29	.3764	.01	.03	1580.5260	6.5140	.3728	.0353
30	.3763	.01	.03	1580.3320	6.5283	.3723	.0353
31	.3762	.01	.03	1580.1380	6.5426	.3718	.0353
32	.3761	.01	.03	1579.9430	6.5569	.3713	.0353
33	.3760	.01	.03	1579.7490	6.5711	.3708	.0353
34	.3760	.01	.03	1579.5550	6.5854	.3703	.0353
35	.3759	.01	.03	1579.3600	6.5996	.3697	.0353
36	.3758	.01	.03	1579.1650	6.6139	.3692	.0353
37	.3757	.01	.03	1578.9700	6.6281	.3687	.0353
38	.3756	.01	.03	1578.7750	6.6423	.3683	.0353
39	.3755	.01	.03	1578.5810	6.6566	.3678	.0353
40	.3754	.01	.03	1578.3860	6.6708	.3673	.0353
41	.3753	.01	.03	1578.1910	6.6851	.3668	.0353
42	.3752	.01	.03	1578.0060	6.6993	.3663	.0353
43	.3751	.01	.03	1577.8130	6.7136	.3658	.0353
44	.3751	.01	.03	1577.6220	6.7278	.3653	.0353
45	.3750	.01	.03	1577.4390	6.7420	.3648	.0353
46	.3749	.01	.03	1577.2740	6.7562	.3643	.0353
47	.3748	.01	.03	1577.1440	6.7703	.3639	.0353
48	.3747	.01	.03	1577.1360	6.7844	.3634	.0353
49	.3746	.01	.03	1577.4200	6.7981	.3630	.0353
50	.3744	.01	.03	1578.5160	6.8109	.3628	.0353
51	.3740	.01	.03	1581.7190	6.8212	.3630	.0353
52	.3732	.01	.03	1590.2160	6.8254	.3642	.0353
53	.3714	.01	.03	1610.1480	6.8153	.3675	.0354
54	.3682	.01	.03	1646.2870	6.7822	.3738	.0355
55	.3643	.01	.03	1684.6500	6.7435	.3798	.0356
56	.3618	.01	.03	1703.7580	6.7469	.3814	.0356

57 ***Reboiler: No tray sizing***